

PRELIMINARY DESIGN OF A FREE VORTEX
AXIAL FLOW TURBINE

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PRELIMINARY DESIGN OF A FREE VORTEX AXIAL FLOW TURBINE

by

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ABSTRACT

The thesis covers the mechanical, thermodynamics, and aerodynamics of designing a free vortex axial flow turbine with the aid of a digital computer. It uses the Improved Ainley-Mathieson Performance Estimation Method to determine the aerodynamic pressure losses so off-design and design point calculations can be conducted to produce performance curves. It also gives physical and thermodynamic properties of the turbine.

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The purpose of this thesis is to develop a computer program which can be used in the Preliminary Design of a Free Vortex Axial Flow Turbine. The design plan consists of three main parts. The first will be cycle calculations to determine stage characteristics at each station such as angles, density, temperature, flow areas, etc. The properties from this phase will be used in parts 2 and 3. The second part is to determine mechanical characteristics such as size and weights of turbine parts and stress and vibrational characteristics. The final phase is to determine turbine characteristics at design and off design parameters such as speed, pressure ratio, etc. This last phase is optional since compressible flow tables and gas properties are required and many turbine designers may be more interested in the size and weights than off design characteristics in the preliminary design phase. These three parts are covered in chapters one through three, chapter four will cover the operation of the computer program developed in this text. Appendix one is a list of symbols used in the text, appendix two is a user's guide to the computer program and has an example turbine design which is compared with a design proposed by reference 20. Appendix three is the computer program.

The computer program is set up so it has many options of input variables so that as the designer becomes more knowledgeable

in his design study he can specify more characteristics and have more control over the design output. There are checks in the program to limit poor design properties such as excessive swirl in output, high tip to hub ratios, negative reactions, etc. The computer program can be used to design a constant hub, mean, or tip turbine. The design does not consider the effects of cooling in turbines, since time was not available to incorporate this into the computer program. This limitation mostly affects aircraft gas turbine and aircraft derivative marine gas turbines since these engines operate at high turbine inlet temperatures and cooling is required to allow for a high mean time between failures.

There has been no effort made to prove the relations used in the design calculations. These can be found in basic physics, thermodynamics, and fluid mechanics books.

The notation as to signs of angles in the velocity diagrams will be the same as used in chapter 4 of Gas Turbine Handbook which is reference (3). An illustration is enclosed in chapter 1 of the sign convention and notation used in this text. A deviation from this notation will be used in section (3.2).

1.1 The basic design of an axial flow turbine starts with the cycle calculations to determine the thermodynamic properties throughout the turbine. The basis of these calculations are from known inlet conditions and assumed design parameters. The computer program is set up so that a minimum number of these parameters must be specified and leaves the option to the designer to more fully specify the design parameters, if he so desires. This chapter intends to show how these basic calculations are carried out and the methods employed to determine the physical properties such as flow area from these design parameters. Figures (1-1), (1-2), and (1-3) show the sign convention and notation that will be used in this chapter. The following are some of the design parameters needed for the cycle calculations; inlet temperature ($^{\circ}\text{R}$) T_{01} , turbine inlet pressure (psi) P_{01} , omega (rpm) Ω , specific heat (btu/lbm $^{\circ}\text{R}$) C_p , mass flow (lbm./sec) \dot{m} , ratio of specific heat γ , tip radius (ft) r_t , tip to hub ratio (r_t/r_h), desired efficiency η_t , loading coefficient ψ , etc. Appendix two (2) will specify which parameters must be known and which ones are optional.

1.2 Determination of overall turbine parameters from inlet conditions and assumed design parameters.

$$(a) \quad (1.2-1)$$

$$\Delta h_{0t} = \frac{\text{Power (hp)} \cdot 550}{\dot{m} \eta_t} \quad (1.2-2)$$

$$T_{0e} = T_{0i} - \frac{\Delta h_{0t}}{C_p} (^{\circ}\text{R})$$

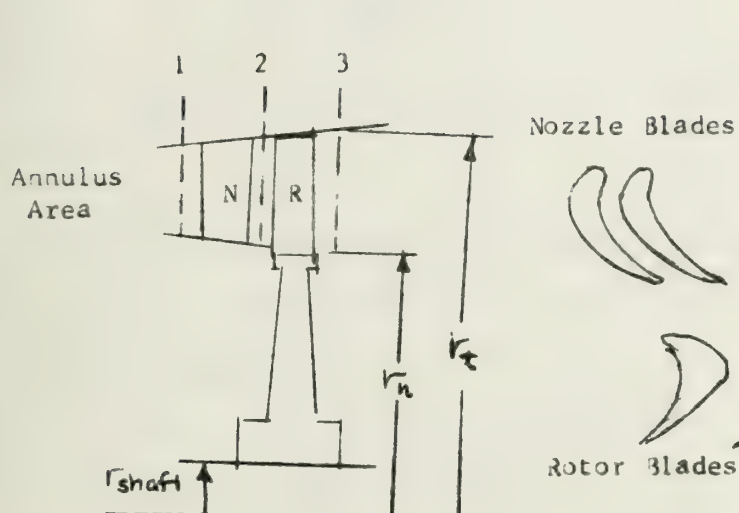


Fig. 1-1 Axial Flow Turbine Stage

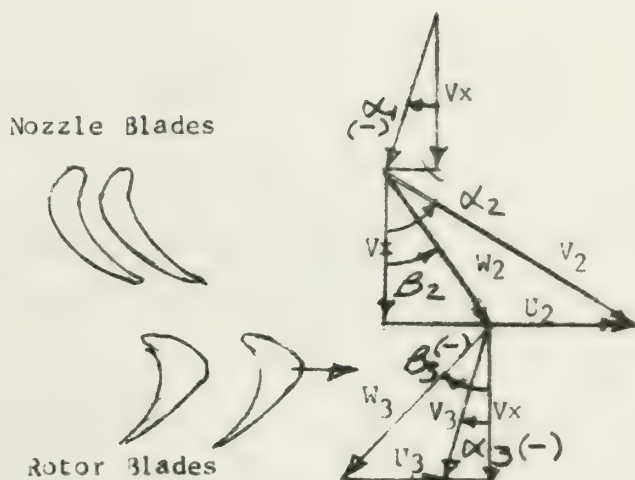


Fig. 1-2 Velocity Diagram

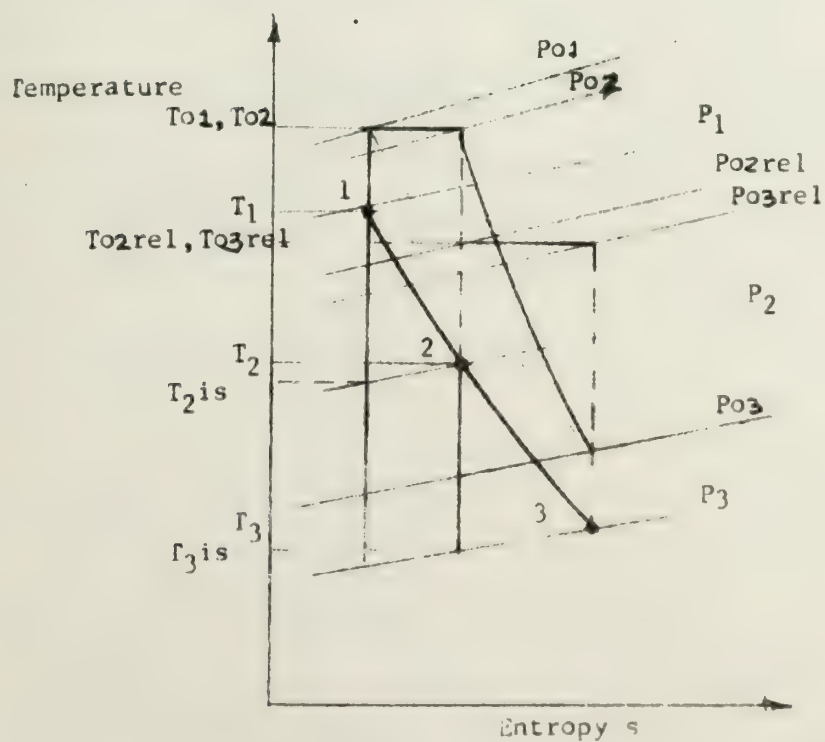


Fig. 1-3 T-S diagram for a Reaction stage

$$\eta_p = \frac{\ln(T_{oe}/T_{oi})}{\ln \left[\frac{(T_{oe}/T_{oi} - 1)}{\eta_t} + 1 \right]} \quad (1.2-3)$$

$$P_{oe} = P_{oi} (T_{oe}/T_{oi})^{\frac{\gamma}{\eta_p(\gamma-1)}} \quad (1.2-4)$$

(b) If the pressure ratio were specified instead of output power the following calculations will be performed.

Polytropic Efficiency

$$\eta_p = \frac{\ln(1 - \eta_t(1 - (P_{oe}/P_{oi})^{\frac{\gamma-1}{\gamma}}))}{\ln[(P_{oe}/P_{oi})^{\frac{\gamma}{\gamma-1}}]} \quad (1.2-5)$$

$$T_{oe} = T_{oi} (P_{oe}/P_{oi})^{\frac{\gamma}{\eta_p(\gamma-1)}} \quad (1.2-6)$$

$$\Delta h_o = C_p(T_{oi} - T_{oe}) \quad (1.2-7)$$

$$\text{Power out} = \frac{\dot{m} \Delta h_o \eta_t J}{550} \quad (1.2-8)$$

(c)

$$\omega = \frac{52(2\pi k)}{60} \quad (\text{rad/sec}) \quad (1.2-9)$$

$$r_h = r_t / (r_t / r_h) \quad (1.2-10)$$

$$r_m = (r_h + r_t) / 2 \quad (1.2-11)$$

$$U_h = r_h \omega (\text{ft/sec}) \quad (1.2-12)$$

$$\Delta h_{o\text{stage}} = \frac{U_h V_x (\tan \alpha_2 - \tan \alpha_3)}{g_0 J} \quad (1.2-13)$$

$$\psi = \frac{g_0 J \Delta h_{o\text{stage}}}{U_h^2} \quad (1.2-14)$$

The value of ψ is taken at the hub of the last stage because this is the most critical in the design of a turbine in reference to loading. Normally the value of ψ is from (1.5 - 2.9). The lower value is for lightly loaded stages while the higher is for highly loaded stages. To have no exit swirl ψ must be less than or equal to 2.0.

$$\psi \leq 2.0 \Rightarrow (\text{no exit swirl})$$

$$\text{for } \alpha_h = 0 \quad R = 1 - \psi/2 \quad (1.2-15)$$

where

$$R = \frac{h_2 - h_3}{h_{01} - h_{03}} \quad \begin{array}{l} \text{ratio of change in static enthalpy} \\ \text{across the rotor to change in} \\ \text{stagnation enthalpy across the stage.} \end{array} \quad (1.2-15a)$$

With the value of ψ specified, the number of stages required to develop the power out desired can be determined.

$$\Delta h_{o\text{stage}} = \frac{\psi U_h^2}{g_0 J} \quad (1.2-16)$$

Determining number of stages

$$N = \text{integer } \frac{\Delta h_{ot}}{\Delta h_{o\text{stage}}}$$

If N were specified instead of ψ , the change in enthalpy per stage would be determined.

$$\Delta h_{o\text{stage}} = \frac{\Delta h_{ot}}{N}$$

This program uses a constant change in enthalpy for each stage.

If $\psi \leq 2.0$ and $\alpha_h = 0$ the radius of the hub can be determined from equation (1.2-17).

$$r_{he} = \left[\frac{g_o J \Delta h_{o_{stage}}}{\omega^2} \right]^{\frac{1}{2}} \quad (1.2-17)$$

Another parameter which will be used is $\phi \equiv V_x / U_h$ (flow coefficient). Where V_x is the axial velocity, which is constant in a free vortex turbine. This parameter ranges from 0.4 - 1.3, the higher values are used for aircraft turbines which use high axial velocity for propulsion. In marine and industrial turbines a low value is desired because the energy of the exhaust gases are not recoverable in this form, unless it is used to drive another turbine.

Another parameter that may be specified is the critical Mach number at the exit.

$$M_{crit} = \frac{V_x}{\left[\frac{2}{\gamma+1} g_o R T_{oe} \right]^{\frac{1}{2}}} \quad (1.2-18)$$

where R is the gas constant

$$R \equiv C_p(\gamma-1/\gamma)J \quad (1.2-19)$$

The V_x is computed exactly

$$V_x = M_{crit} \left[(2/(\gamma+1)) g_o R T_{oe} \right]^{\frac{1}{2}} \quad (1.2-18a)$$

If the exit Mach number is specified the turbine exit area is varied until the required area is obtained to give the specified axial velocity V_x . If this number is not specified but the exit area is specified the value of V_x is varied until the required area is achieved.

From the continuity equation for gases

$$V_x = \frac{\dot{m}}{2\pi \int_{r_h}^{r_t} \rho(r) r dr} \quad (1.2-20)$$

where

$$\rho(r) = \frac{P(r)}{RT(r)} \quad (1.2-21)$$

$$T(r) = T_o - \frac{V(r)^2}{2C_{pgoJ}} \quad (1.2-22)$$

$$P(r) = P_o / (T_o / T(r))^{\frac{\gamma}{\gamma-1}} \quad (1.2-23)$$

$$V(r) = \frac{V_x}{\cos(\alpha(r))} \quad (1.2-24)$$

where r varies between the hub and tip. If the exit angle is not equal to zero

$$\alpha_{hub_e} = \tan^{-1} \left[(1 - R_{he} - \psi/2) / \phi \right] \quad (1.2-25)$$

and for free vortex

$$rV_\theta = rV_x(\tan \alpha) = \text{constant} \quad (1.2-26)$$

then

$$\alpha(r_e) = \tan^{-1} \left[\frac{r_{he} \tan(\alpha_{he})}{r_e} \right] \quad (1.2-27)$$

where r_e is taken at hub mean, and tip. This calculation is carried out in function program ANGLE. Thus the exit angle has been determined at the hub, mean, and tip radii.

The pressure, density, and temperature calculations are performed in subroutine TPl. The integral calculation is performed in subroutine DIENR which uses Simpson's Rule to compute the axial velocity V_x . With the absolute angles $\alpha(r_e)$ determined the relative angles $\beta(r_e)$ can then be determined by equation (1.2-28). This is carried out in function program ANGLEB.

$$\beta(r_e) = \tan^{-1} \left[\tan(\alpha(r_e)) - \frac{U(r_e)}{V_x} \right] \quad (1.2-28)$$

From equation (1.2-15a)

$$R(r_e) = 1 - \frac{\Delta h_{\text{stage } 30J}}{2(\omega r_e)^2} - \frac{V_x \tan(\alpha_e)}{\omega r_e} \quad (1.2-29)$$

which is computed in function program REACTI. With the exit conditions specified, the properties at station 1 and 2 can then be determined for each stage.

The following properties are now known at the exit of the turbine; V_x , T_{0e} , P_{0e} , $f(r_e)$, $P(r_e)$, $T(r_e)$, $\alpha(r_e)$, $\beta(r_e)$, $R(r_e)$, and r_e , where $r_e = r_{\text{hub}}, r_{\text{mean}}, r_{\text{tip}}$.

1.3 Determination of properties at station 1 for stage n where $n = 1 \dots N$.

$$T_{01} = T_{03n} + \frac{\Delta h_{\text{stage}}}{C_p} \quad (1.3-1)$$

$$P_{01n} = P_{03n} / (T_{03n} / T_{01n})^{\frac{\gamma}{\gamma-1}} \quad (1.3-2)$$

If this is the 1st stage $\alpha_{1n} = 0.0$ where $n = 1$

If not $\alpha_{1n}(r_c) = \alpha_{3n}(r_c)$ where r_c is the radius which remains constant through out the turbine, ie(constant hub, tip, or mean).

$$\beta_{ln}(r_c) = \tan^{-1} \left[\tan(\alpha(r_c)) - \frac{U(r_c)}{V_x} \right] \quad (1.3-3)^{16}$$

$$T_{ln}(r_c) = T_{o,ln} - \frac{V_{ln}^2(r_c)}{2g_o J C_p} \quad (1.3-4)$$

$$V_{ln}(r_c) = \frac{V_x}{\cos(\alpha_{ln}(r_c))} \quad (1.3-5)$$

$$P_{ln}(r_c) = P_{o,ln} / (T_{o,ln} / T_{ln})^{\frac{\gamma}{\gamma-1}} \quad (1.3-6)$$

$$\rho_{ln}(r_c) = \frac{P_{ln}(r_c)}{RT_{ln}(r_c)} \quad (1.3-7)$$

which is identical to equations in section for exit of the turbine.

A trial blade height (h) at station ln will be established.

$$h = \frac{\dot{m}}{2\pi V_x r_c \rho(r_c)} \quad (1.3-8)$$

Let $\Delta h = h/10$. Now depending on whether the turbine is constant hub, mean, or tip, radius are established at the other radii.

Constant hub;

$$r_{ln}(\text{hub}) = r_c \quad (1.3-9)$$

$$r_{ln}(\text{mean}) = r_c + h/2 \quad (1.3-10)$$

$$r_{ln}(\text{tip}) = r_c + h \quad (1.3-11)$$

Similarly for constant mean or constant tip.

α can then be determined at all radii for station l and temperature; pressure, and density can be calculated at all radii in the same manner as was carried out at the exit of the turbine. Then

V_x prime can be calculated in the same manner as V_x was calculated.

$$V_x \text{ prime} = \frac{\dot{m}}{2\pi \int_{r_h}^{r_t} \rho(r) r dr} \quad (1.3-12)$$

When $V_x \text{ prime} = V_x$ the proper radii has been calculated at station 1. The method employed continuously decreases delta h until convergence is accomplished. This is the same method which was used at the exit of the turbine and will be used in station 2 calculations. The properties of the gas is now determined as per exit of the turbine for station 1. If this isn't the first stage the properties at station 1 are the same as station 3 of the preceeding stage.

1.4 Determination of properties at station 2 for stage n where n = 1 N

$$T_{02n} = T_{01n} \quad (1.4-1)$$

$$\alpha_{2n}(r_c) = \tan^{-1} \left[\frac{g_0 \Delta h_{os}}{V_x \omega r_c} + \tan(\alpha_{3n}(r_c)) \right] \quad (1.4-2)$$

$$V(r_c) = V_x / \cos(\alpha_{2n}(r_c)) \quad (1.4-3)$$

$$T_{2n} = T_{02n} - V(r_c)^2 / 2g_0 J C_p \quad (1.4-4)$$

$$P_{2n} \approx P_{1n} (T_{2n}/T_{1n})^{\frac{\gamma}{\gamma-1}} \quad (1.4-5)$$

which is an approximation from figure (1-3), where the actual pressure is determined from pressure loss data as will be covered in Chapter 3.

$$P_{02n} = P_{2n} (T_{02n}/T_{2n})^{\frac{\gamma}{\gamma-1}} \quad (1.4-6)$$

$$\rho_{2n} = \frac{P_{2n}(r)}{RT_{2n}(r)} \quad (1.4-7)$$

determine temperature, pressure, and density at station 2. The calculations are carried out in the same manner as station 1 calculations, to determine the gas properties and radii at station 2. The calculations are then carried out at station 1 of the next stage (ie; $n = n-1$ until properties at all stations and positions have been computed for all stages.)

1.5 Having solved for gas properties at all stations and positions and having flow areas and gas angles the relative stagnation temperature, and the relative Mach numbers for stators and rotors can be calculated.

$$T_{02rel} = T_2 + \frac{w^2}{2g_0 C_p J} \quad (1.5-1)$$

$$M_{nozzle\ inlet} = \left[\left(\frac{T_{01n}}{T_{1n}} - 1 \right) \frac{2}{\gamma-1} \right]^{\frac{1}{2}} \quad (1.5-2)$$

$$M_{nozzle\ exit} = \left[\left(\frac{T_{02n}}{T_{2n}} - 1 \right) \frac{2}{\gamma-1} \right]^{\frac{1}{2}} \quad (1.5-3)$$

$$M_{rotor\ inlet} = \left[\left(\frac{T_{02nrel}}{T_{2n}} - 1 \right) \frac{2}{\gamma-1} \right]^{\frac{1}{2}} \quad (1.5-4)$$

$$M_{rotor\ exit} = \left[\left(\frac{T_{03nrel}}{T_{3n}} - 1 \right) \frac{2}{\gamma-1} \right]^{\frac{1}{2}} \quad (1.5-5)$$

These calculations are carried out at each stage. The following parameters have been determined at all stations of the turbine; temperature, pressure, density, flow area, absolute and relative angles, Reaction, relative Mach numbers, and the axial velocity V_x . These properties will be used to determine the mechanical aspects of the Turbine design and be used in performance estimation.

Mechanical Design of an Axial Flow Turbine

2.1 In this chapter the mechanics of the program and assumptions made to design nozzle and rotor blades, and the turbine disc will be discussed.

2.2 Having the gas properties and flow angles throughout the turbine the blade angles can be calculated for the turbine. The criteria used in establishing the blade angles is the loss data provided by figure (2-2) and (2-3) based upon Ainley Mathieson method of performance estimation. As can be seen for reaction blades in figure (2-2) the incidence angles can vary from -15 to $+15$ without much of an increase in profile loss γ_p . In actual construction this large variation can allow for a decrease in twist in the blade from hub to tip normally required in free vortex blades. For the computer program, selection of blade angle, at the inlet of the blade is for zero incidence at design point.

Figure (2-3) shows how blade exit angle varies with gas angle there is also a correction for Mach number. For Mach numbers less than 0.5 the values given by figure (2-3) will be used, for values greater than Mach 1 the blade angles will be equal to gas angles while for values between Mach 0.5-1.0 a linear variation will be assumed as is shown in figure (2-4). The calculation of these blade angles are carried out in subroutine Blade, which also will be used to determine gas angles when blade angles are known for off-design calculations in chapter 3. There is an additional correction for blades

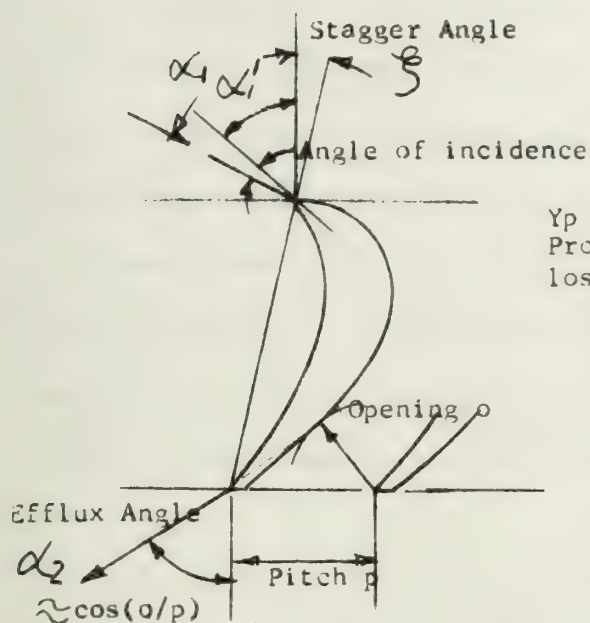


Figure 2-1. Conventional Blade Profile

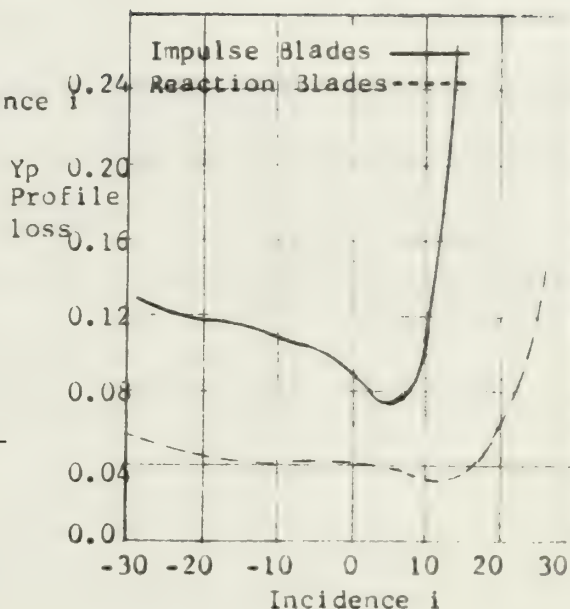


Figure 2-2. Variation in profile loss with incidence angle (From D.G.Ainley and G.C.R.Mathieson(1))

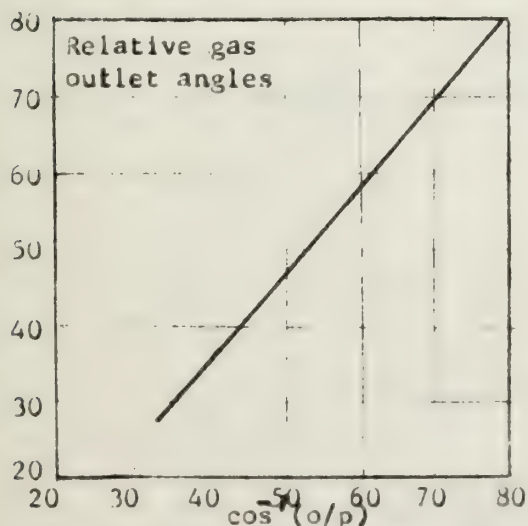


Figure 2-3. Relation between gas and blade angles (From D.G.Ainley and G.C.R.Mathieson(2))

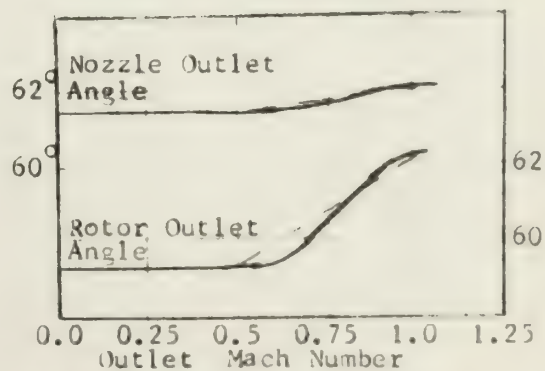


Figure 2-4. Angle variation with mach number (From D.G.Ainley and G.C.R.Mathieson(2))

with a curved back trailing edge in reference (2) but this will not be applied in the computer program.

2.3 The computer program can now determine the ideal pitch to chord ratio P/C , blade chord (C), aspect ratio H/C , bending and centrifugal stresses, number of blades, 1st bending frequency, and weight of blades. As will be shown in this section all these parameters are interrelated and they all help determine each other.

2.3a Centrifugal Stress Calculations

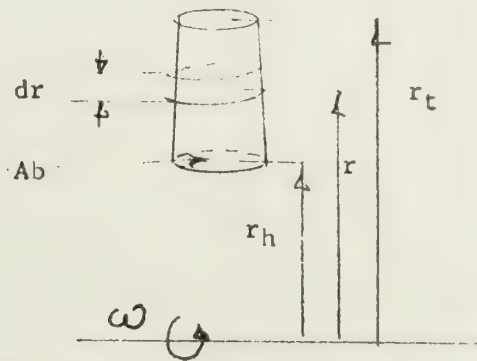


Figure 2-5. Rotating Turbine blade

$$dF = dmr \omega^2 = dmr (2\pi N/60)^2 \quad (2.3-1)$$

$$A(r) = Ab f(r) \quad (2.3-2)$$

$$dm = Ab f(r) dr \frac{\rho}{g_0} \quad (2.3-3)$$

$$dF = (2\pi N/60)^2 \frac{\rho}{g_0} A(r) r dr$$

If $A(r) = \text{constant}$

$$F_c = (2\pi N/60)^2 \frac{Ab \rho}{g_0} \int_{r_{hub}}^{r_{tip}} r dr \quad (2.3-4)$$

$$\sigma_c = \frac{F_c}{Ab} = (2\pi N/60)^2 \frac{\rho}{g_0} \left[\frac{r_{tip}^2 - r_{hub}^2}{2} \right]$$

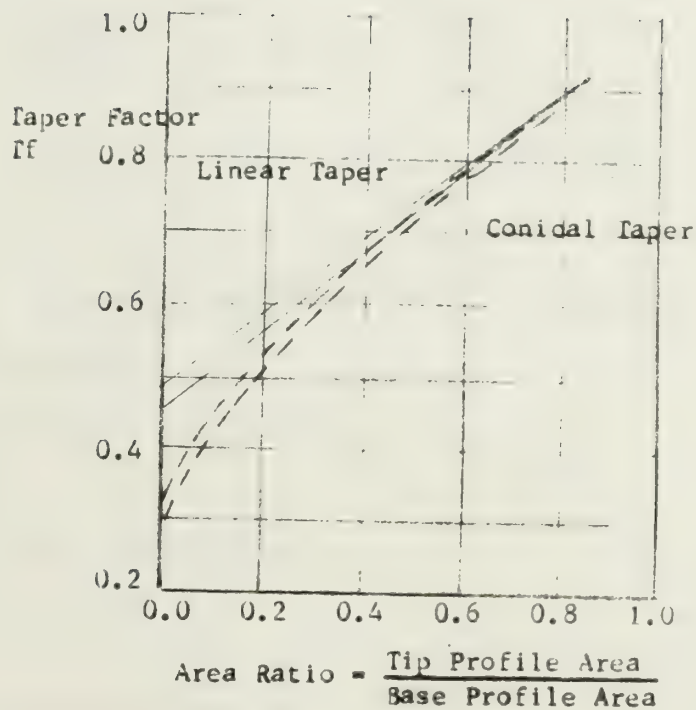


Figure 2-6. Variation of taper factor with blade profile area ratio (From Emmert, Current Design Practices for Gas Turbine Elements.(9))

$$c = 4.51 \text{ Tf } \rho \text{ Ab } \left(\frac{\Omega}{1000} \right)^2 \quad (2.3-5)$$

where; c = average tensile stress(psi)

Tf = taper factor(from figure (2-6))

ρ = specific mass of the blade

A = annular flow area of blade ring (sq. in.)

Ω = shaft speed (rpm)

For the computer program Tf will be based on linear variation with tip to hub area ratio (areara) as an input, normally this will be between 0.25 - 0.333

If a linear taper is assumed equation (2.3-1) can be integrated directly assuming;

$$A(r) = A_b(1 - \alpha r) \quad (2.3-6)$$

$$\begin{aligned} F_c &= \left(\frac{2\pi\Omega}{60} \right)^2 \frac{\rho_b}{g_o} A_b \int_{r_{hub}}^{r_{tip}} (1 - \alpha r) r dr \\ &= \left(\frac{2\pi\Omega}{60} \right)^2 \frac{\rho_b}{g_o} A_b \left[\frac{(r_{tip}^2 - r_{hub}^2)}{2} + \frac{\alpha}{3} (r_{hub}^3 - r_{tip}^3) \right] \end{aligned} \quad (2.3-7)$$

$$\epsilon_c = \frac{F}{A_b} = \left(\frac{2\pi\Omega}{60} \right)^2 \frac{\rho_b}{g_o} \left[\frac{(r_{tip}^2 - r_{hub}^2)}{2} + \frac{\alpha}{3} (r_{hub}^3 - r_{tip}^3) \right] \quad (2.3-8)$$

and α can be evaluated

$$A_{tip} = \alpha \text{ area ratio } (A_b)$$

$$\alpha = \frac{1 - \text{area ratio}}{r_{tip}} \quad (2.3-9)$$

2.3b Determination of ideal pitch to chord ratio. This value for P/C ratio is taken from figure (2-7) which is a correlation of Ainley's optimum spacing data. For stators the inlet gas angle to enter figure (2-7) is $-\alpha_1$ and the relative efflux angle is α_2 . For rotors β_2 is the inlet gas angle while $-\beta_3$ is the relative efflux angle. This calculation is done in subroutine STOCRA and the input angles are taken at the mean radius.

2.3c Determination of non-dimensional section modulus (SM) and non-dimensional area at the base of the blade (Area/C^2). An input to get these values is the turning angle at the base of the blade, ie (hub for rotor and tip for stator). Figure (2-8) taken from reference (7) illustrates how the non dimensional base area is obtained. The non-dimensional section modulus is obtained from figure (2-9) which is

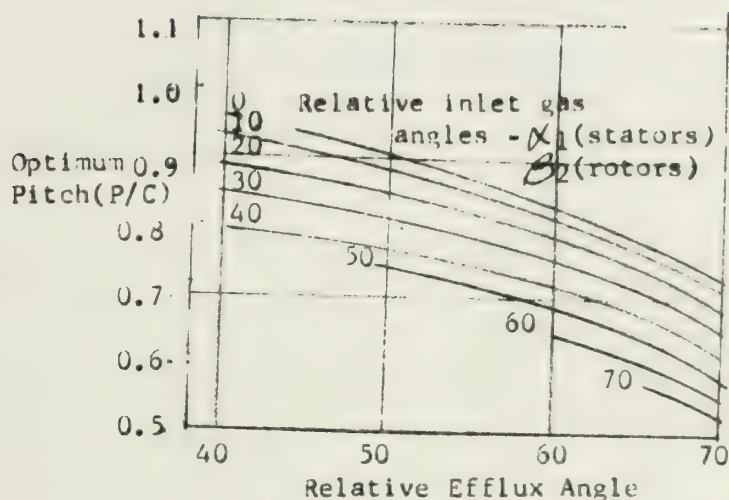


Figure 2-7. Optimum spacing of turbine blades. (From a correlation of Ainley's loss data (2))

an unpublished relationship provided by Ainley for use in reference (6).

A modification was made to this relationship based upon calculations performed by Professor A.D. Carmichael to make the section modulus larger to conform to modern turbine blades. The SM was multiplied by 2.35 for use in the computer program.

2.3d Determining bending stresses on turbine blades. As can be seen in figure (2-10) and equation (2.3-10) a detailed calculation would be needed to determine the gas bending stresses on a turbine blade but because angle ϕ is small and M_w is by far the greater bending moment an approximation proposed in reference (6) is used in the preliminary design.

$$(\mathcal{S}_{gb})_{\max} \approx \frac{m V_x (\tan \alpha_2 - \tan \alpha_3) h}{2 g o n Z} \quad (\text{rotor}) \quad (2.3-11)$$

where the angles are taken at mean radius and the following are the definition of the parameters;

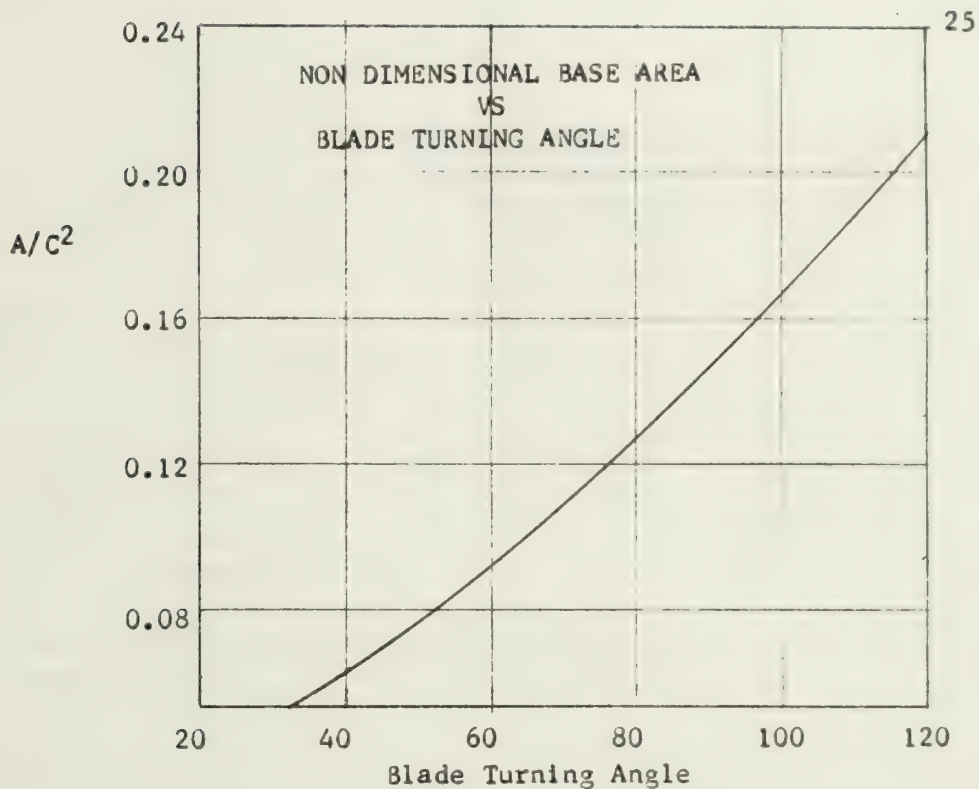


Figure 2-8. Approximate Areas of Turbine Blades vs Turning angle. (From paper by R.E.Dundas in Gas Turbine Handbook(7))

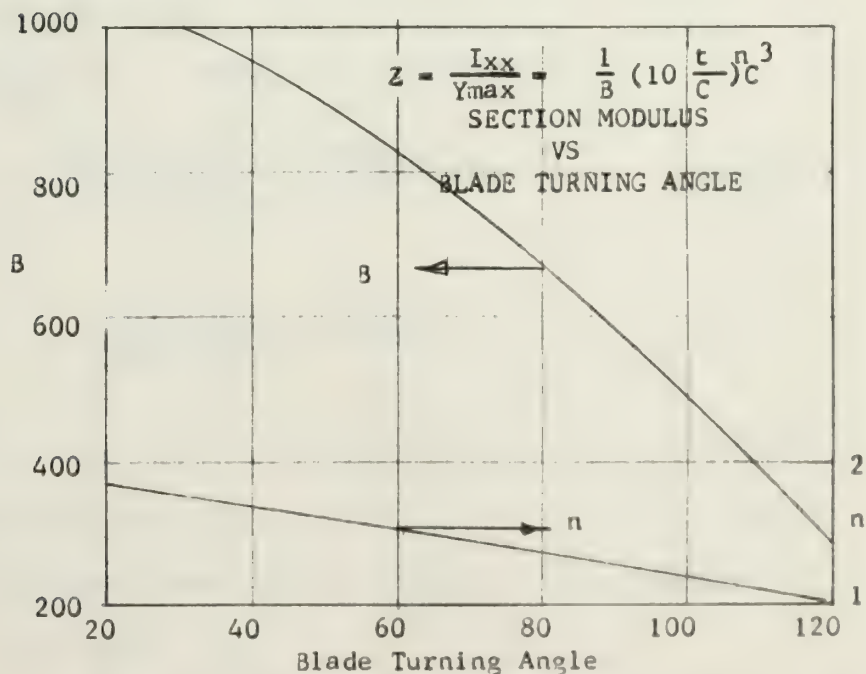


Figure 2-9. Section Modulus as a function of blade turning angle. (From unpublished paper by Ainley for use in reference (6))

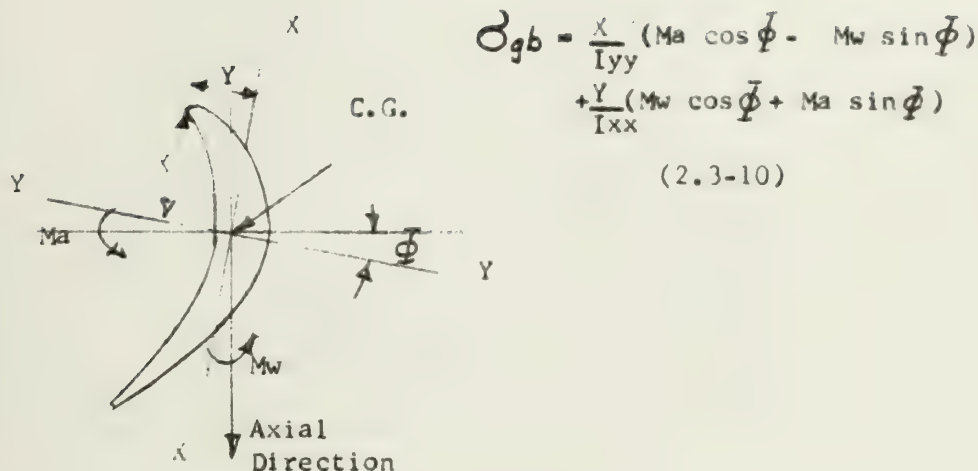


Figure 2-10. Bending moments acting on a turbine blade.

Z = SM C Section Modulus

SM = non dimensional section modulus

n = number of blades

$$= \frac{2\pi r_m}{\text{Pitch}} \quad \text{where } r_m = \text{mean radius}$$

$$(\sigma_{gb})_{\max} = \frac{\dot{m} V_x (\tan \alpha_2 - \tan \alpha_3) h}{2 g_o \frac{2\pi r_m}{p} \text{SM } C^3} \quad (2.3-11a)$$

$$= \frac{\dot{m} V_x (\tan \alpha_2 - \tan \alpha_3) h \frac{p}{C}}{4\pi g_o r_m \text{SM } C^2}$$

At this point the blade chord has not been determined so the bending stress cannot be calculated, but equation (2.3-11a) will be coupled with a natural frequency equation and a cyclic loading equation to calculate the blade chord.

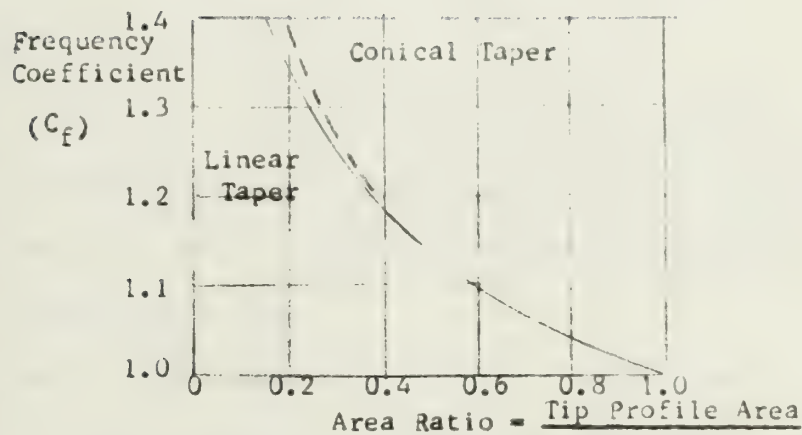


Figure 2-11 Effect of taper on blade natural frequency (From Emmert, Current Design Practices for Gas Turbine Power Elements (9))

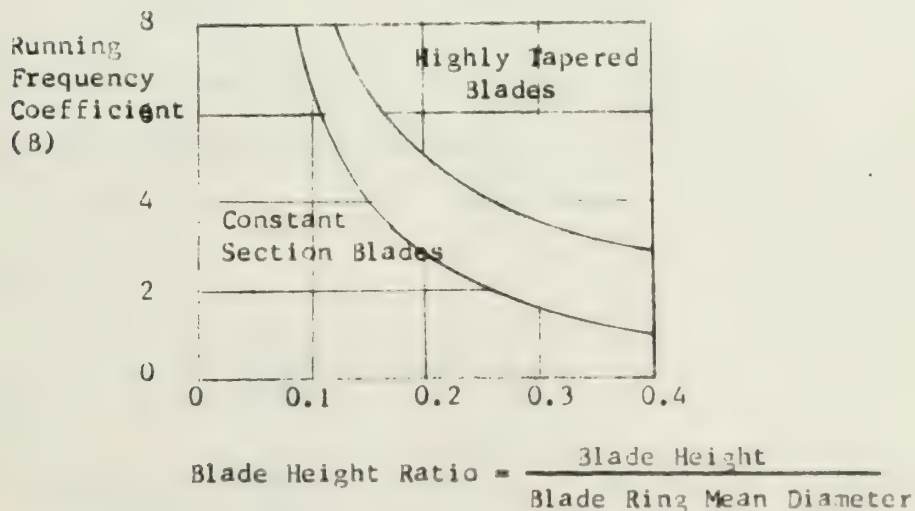


Figure 2-12. Variation of rotation coefficient with blade height ratio. (From Emmert, Current Design Practices for Gas Turbine Power Elements. (9))

2.3e Determination of natural bending frequency of blades from 28 reference (9). The following expression is used in the computer program.

$$f_s = 11.0 C_f \frac{k}{h^2} \sqrt{\frac{E}{\rho}} \quad (2.3-12a)$$

f_s is standing frequency (cps)

C_f frequency correction factor taken from figure (2-11)

k minimum radius of gyration of base profile (in)

h blade height (in)

E modulus of elasticity (psi) $(29 - 30) \times 10^6$ psi

ρ specific weight of blade material (lb/in³)

$$f_s = 11.0 \frac{C_f}{h^2} \sqrt{\frac{IE}{A \rho}} \quad (2.3-12b)$$

$$= 11.0 \frac{C_f}{h^2} \sqrt{\frac{ZtE}{A \rho}} \quad \text{assuming } I \approx Zt$$

A cross sectional area of base of blade (in²)

t maximum thickness of blade (in)

$$= (t/C) C$$

$$A = (A/C^2) C^2$$

$$f_s = 11.0 \frac{C_f}{h^2} \sqrt{\frac{SM C^3 (t/C) E C}{(A/C^2)}}$$

$$= 11.0 \frac{C_f}{h^2} \left[\frac{SM (t/C) E}{(A/C^2)} \right]^{\frac{1}{2}} \text{ chord} \quad (2.3-12c)$$

The running frequency f given by equation (2.3-13) is a function of the natural frequency and the speed of rotation of the blade. The

factor B is given by figure (2-12). A plot of this frequency as a function of operating speed is illustrated in figure (2-13) which is called a Cambell Diagram. Normally for good design practice the running frequency should avoid the 1st four harmonics of the operating speed in its operating range. For the design of the blade chord in the computer program the effect of the rotational speed as indicated in equation (2.3-13) will be neglected but a more stringent criteria using the harmonics of the operating speed to estimate the vibrational stress in the blade will be used.

$$f_r = \sqrt{f_s^2 + B(\Omega/60)^2} \quad (2.3-13)$$

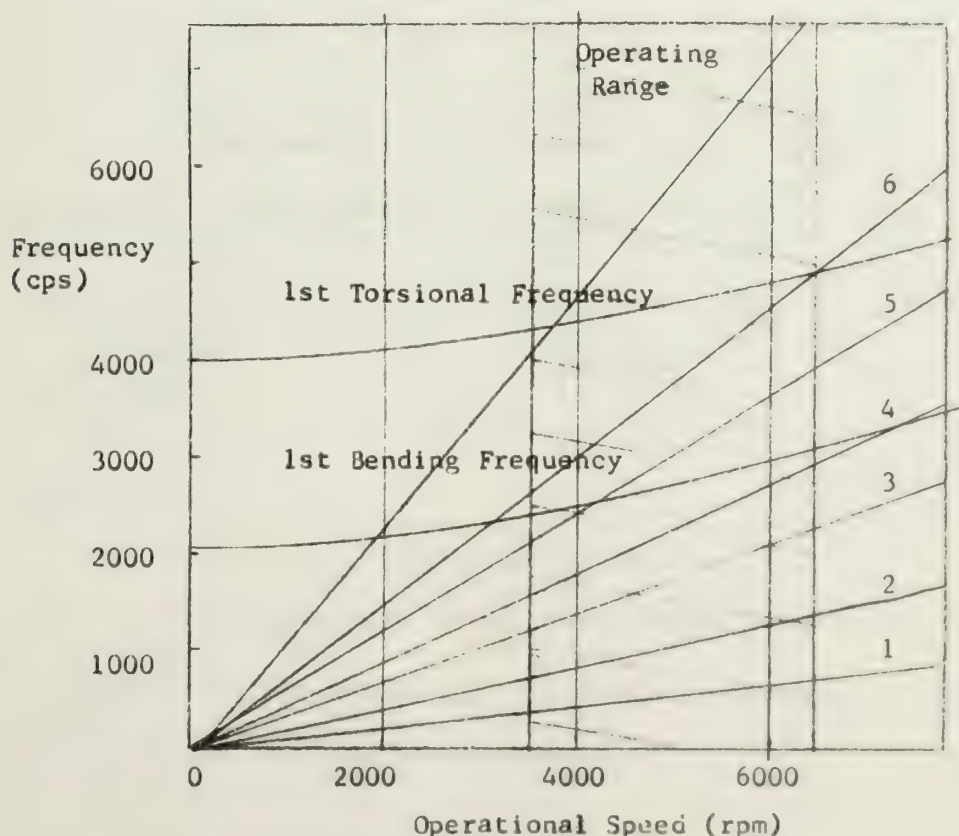


Figure 2-13. Cambell diagram, plot of frequency of vibrations as a function of operating speed.

2.3f Determination of turbine blade characteristics based on cyclic loading requirements by use of a Goodman Diagram of steady and vibrational stresses. In reference (7) a use of the Goodman diagram is used to determine the chord of a compressor blade. This can also be applied to turbine blades. A Goodman diagram is a plot of vibrational stresses as a function of steady stresses. The vibrational stress is taken at 10^7 cycles at the operating temperature.

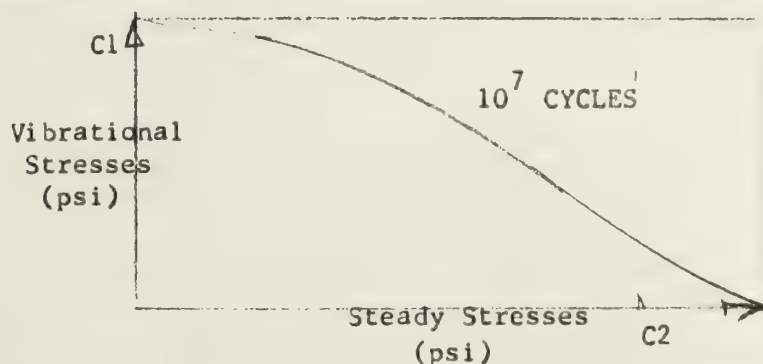


Figure 2-14. Goodman Diagram of Vibrational stresses as a function of steady stresses.

This plot can be approximated as a linear function.

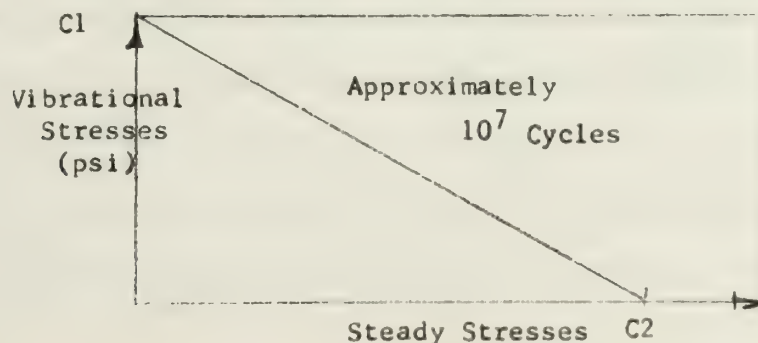


Figure 2-15. Linearized Goodman Diagram

In reference (7) the vibrational stress is proposed as a function of the bending stress as shown in equation (2.3-14)

$$\sigma_{\text{vib}} = 1.3 \text{ Amplification factor } \sigma_{\text{gb}} \quad (2.3-14)$$

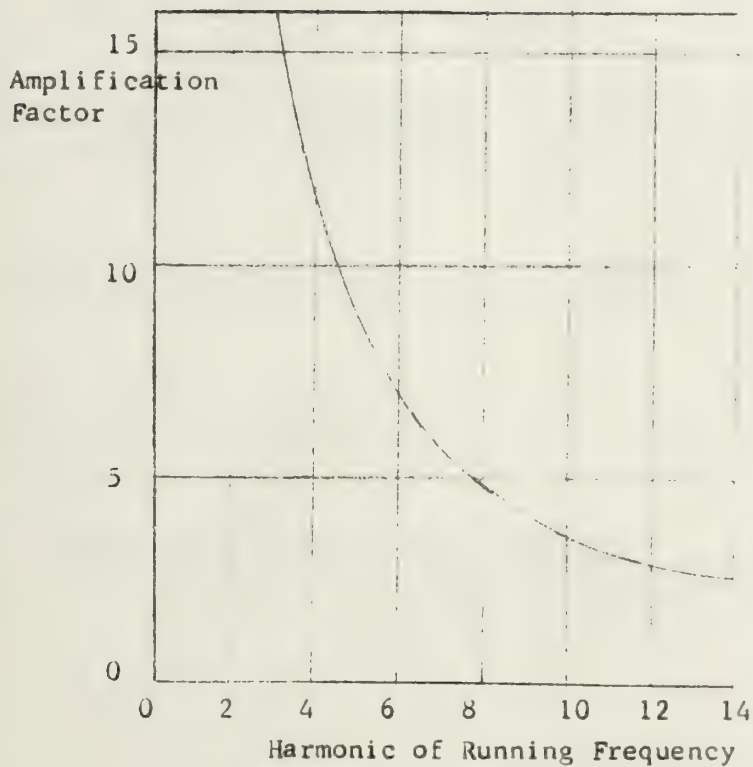


Figure 2-16. Amplification Factor. (From Paper of Trumpler & Owens (19))

where Amplification factor is shown in figure (2-16) as a function of the harmonic of operating speed. The 1.3 is a stress concentration factor specified in reference (7). From personal conversation with Professor **A. Douglas Carmichael** at M.I.T. the amplification factor can be approximated by $41/n$ where n is the harmonic of the operating speed.

$$n = \frac{f}{\left(\frac{\Omega(\text{rpm})}{60} \right)} = \left(\frac{f}{\Omega} \right) 60 \quad (2.3-15)$$

From figure (2-15)

$$\delta_{\text{vib}} = C1 - \frac{C1}{C2} \delta_{\text{steady}} \quad (2.3-16)$$

where $\delta_{\text{steady}} = \delta_{\text{centrifugal}} + \delta_{\text{gas bending}}$

and $C1$ is the upper limit for vibrational stresses and $C2$ is the

upper limit for steady stresses.

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$$\delta_{gb} \frac{1.3\Omega/60}{f} + \frac{C1}{C2} = C1 - \frac{C1}{C2} (\delta_{cf})$$

$$\text{but } \frac{C1}{C2} \ll \frac{1.3 (\Omega/60) 41}{f}$$

$$\text{so } \delta_{gb} \frac{(1.3 \times 41 \times \Omega/60)}{f} \approx C1 - \frac{C1}{C2} (\delta_{cf}) \quad (2.3-17)$$

From equation (2.3-11a), (2.3-12c), and (2.3-17)

$$\frac{\ln Vx(\tan \alpha_2 - \tan \alpha_3)}{4 \pi g_o r_m SM C^3} \left(\frac{P}{C} \right) h \frac{(1.3 \times 41 \times (\Omega/60))}{\frac{11.0 \times C_f}{h^2} \left[\frac{SM(t/C)E}{(A/C^2)} \right]}^{\frac{1}{2}} = C1 - \frac{C1}{C2} \delta_{cf} \quad (2.3-18)$$

$$\text{Stress}_3 = \frac{\ln Vx(\tan \alpha_2 - \tan \alpha_3) h \left(\frac{P}{C} \right)}{4 \pi g_o r_m SM} \quad (2.3-19)$$

$$\text{Freq} = \frac{11.0 \times C_f}{h^2} \left[\frac{SM(t/C)}{(A/C^2)} \right]^{\frac{1}{2}} \quad (2.3-20)$$

$$\frac{1.3 \times 41 (\Omega/60)}{C^3 \text{Freq}} (\text{Stress}_3) = C1 \left(1 - \frac{\delta_{cf}}{C2} \right) \quad (2.3-18a)$$

$$C4 = 1.3 \times 41 (\Omega/60) \text{Stress}_3 / \text{Freq} \quad (2.3-21)$$

$$\frac{C4}{C^3} = C1 \left(1 - \frac{\delta_{cf}}{C2} \right)$$

$$\text{Chord} = C = \left[\frac{C4}{C1 \left(1 - \frac{\delta_{cf}}{C2} \right)} \right]^{1/3} \quad (2.3-22)$$

Now that the chord has been determined the bending stress and natural bending frequency can be determined from equation (2.3-11a) and (2.3-12c) respectively. Having solved for the chord at the base of the blade the chord can be determined at mean and tip assuming a linear taper.

$$Ab_m = \frac{Ab(1 + A_{reara})}{2} \quad (2.3-23)$$

Assuming constant (A/C^2)

$$C_m = \frac{[Ab_m]^{1/2}}{[A/C^2]} \quad (2.3-24)$$

At the opposite end from the base

$$C_t = \frac{[A_{reara}(Ab)]^{1/2}}{[A/C^2]} \quad (2.3-25)$$

where $C_t = C_{hub}$ (for stator)

$C_t = C_{tip}$ (for rotor)

Now that the chord has been determined at the midchord the aspect ratio and pitch can be determined by equation (2.3-26) and (2.3-27).

$$\text{Aspect ratio } (h/C) = h/C_m \quad (2.3-26)$$

$$\text{Pitch}_m = (P/C) C_m \quad (2.3-27)$$

With pitch determined the number of blades can be calculated.

$$\text{Number of blades } (N) = \text{Integer } \frac{2\pi r_m}{\text{Pitch}} \quad (2.3-28)$$

The computer program uses the criteria of having even number of blades for the stator and prime numbers for the rotor to prevent blade interaction of resonant frequencies. The determination of whether a number is prime is done in subroutine TESTP. Thus having solved for the number of blades the pitch can be recalculated and the weight

of the blades. Blades are integrated from the base to end to get volume and is multiply by its specific weight to get the total weight. In the computer program this specific weight is used for both the blades and disc. Also it is assumed they are solid blades with no cooling passages.

$$\begin{aligned}
 \text{Weight of blades} &= N_b \rho b \int_{r_h}^{r_t} A(r) dr & (2.3-29) \\
 &= N_b \rho b A_b \int_r^{r_t} (1 - \alpha r) dr \\
 &= N_b \rho b A_b \left[(r_t - r_h) + \frac{\alpha}{2} (r_h^2 - r_t^2) \right]
 \end{aligned}$$

where α is given by equation (2.3-9)

In this section pitch was determined by a chart of ideal pitch to reduce losses. In actual turbine design the loss data would be coupled with stress analysis of the turbine blade attachment to determine allowable pitch. The type most commonly used is the fir-tree method which is illustrated in figure (2-17). This method of selecting turbine blade chord length is just one of many which are used in industry.

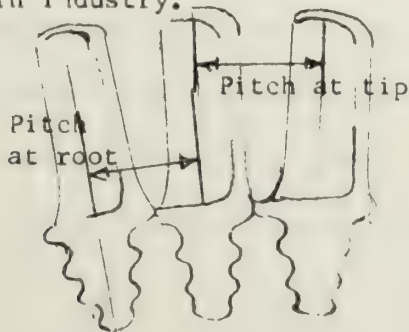


Figure 2-17. Fir-tree blade attachments.

2.4 Turbine Disc Design

The method for design of a disc in this section will be based upon a constant average stress analysis. The stress relations are those proposed in reference (15) and (18). This method will ignore stress concentrations and thermal stresses. In actual disc design these cannot be ignored since thermal stresses may be a major part of the disc stresses and stress concentrations may cause the disk to fail even though the average stress is well below the ultimate tensile stress of the material (UTS). For calculations in the computer program the average tensile stress used is that shown in equation (2.4-1) from reference (7).

$$\sigma_{avg} = \frac{0.75 \text{ UTS}}{\left[\frac{N_b}{N_o} \right]^2} \quad (2.4-1)$$

$$\left[\frac{N_b}{N_o} \right] = \text{Ratio of burst speed to design speed} \quad (2.4-2)$$

The value of this ratio is normally between 1.2 - 1.3. The value of ultimate tensile stress (UTS) is taken at the design operating temperature, higher values producing smaller disc dimensions. This value is an input to the computer program. The computer program has minimum values for some of the dimensions illustrated in figure (2-13) to prevent impossibilities to manufacture and to allow adequate thickness for attachment of blades.

The following equations are based on figure (2-13) taken from reference (15).

$$\sigma_{avg} = \frac{\rho \omega^2 r_o^2 + \frac{N}{2\pi} \sigma_b \frac{A_b}{W_o T_o}}{1 + \frac{L_o r_o}{W_o T_o}} \quad (2.4-3)$$

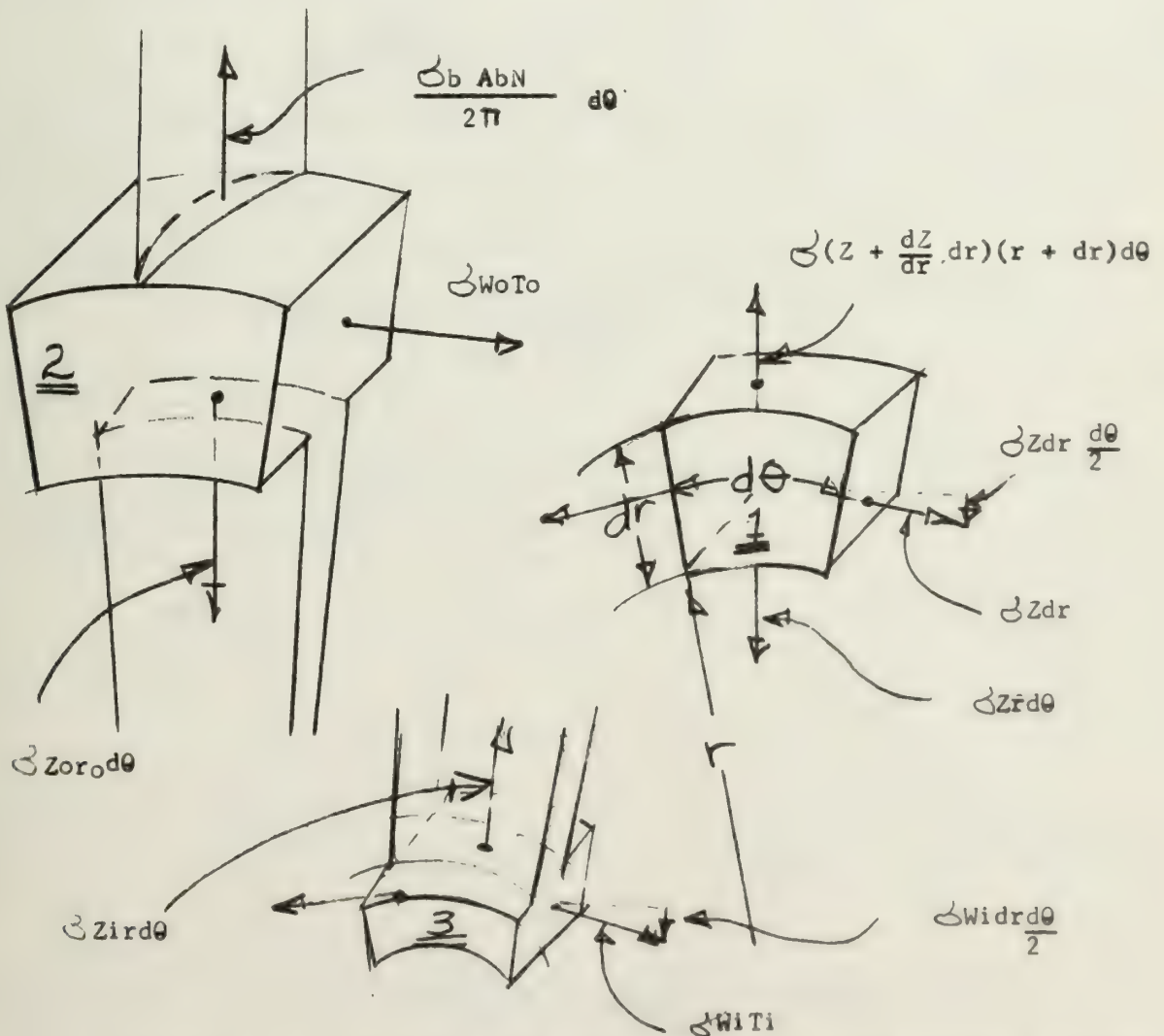
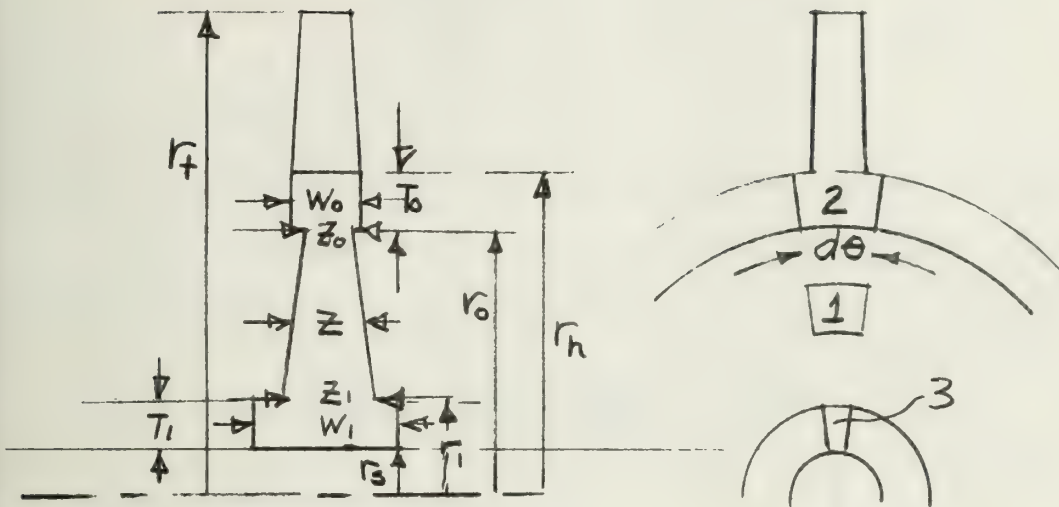


Figure 2-18. Turbine disc stress analysis (From Aircraft Turbines by J.L.Kerrebrock(15))

$$\delta_b = \frac{\rho \omega^2 r_t^2}{2g_o} \left\{ 1 - \left(\frac{r_i}{r_t} \right)^2 - \frac{2\alpha r_t}{3} \left[1 - \left(\frac{r_h}{r_t} \right)^3 \right] \right\} \quad (2.4-4) \quad 37$$

for a linear tapered blade.

For the computer program W_o will be assumed to equal .65 chord and T_o will equal half of W_o unless otherwise specified. With these parameters known Z_o can be solved for by equation (2.4-5). With Z_o

$$Z_o = \left(\frac{W_o T_o}{r_o} \right) \left\{ \left[\left(\frac{r_o}{r_t} \right)^2 + \frac{N_o b}{2\pi W_o T_o} \right] \frac{\rho \omega^2 r_t^2}{2g_o} - 1 \right\} \quad (2.4-5)$$

known $Z(r)$ can be solved for.

$$Z(r) = Z_o e^{\frac{\rho \omega^2 r_t^2}{2g_o} \left[\left(\frac{r_o}{r_t} \right)^2 - \left(\frac{r}{r_t} \right)^2 \right]} \quad (2.4-6)$$

with a minimum of 0.3 inches. The value of r_i is assumed to equal 2 r_{shaft} where r_{shaft} is an input to the computer program. This defines T_i so W_i can be solve for in equation (2.4-7).

$$W_i = \left(\frac{Z_i r_i}{T_i} \right) \left[1 - \left(\frac{r_i}{r_t} \right)^2 \frac{\rho \omega^2 r_t^2}{g_o} \right] \quad (2.4-7)$$

with a minimum of 1.2 W_o .

Having determined the dimensions of the disc the weight of the disc can be determined by equation (2.4-8)

$$\begin{aligned} \text{Disc weight} &= \oint \int_0^{2\pi} \int_{r_s}^{r_o} A(r) dr d\theta \\ &= \oint 2\pi \int A(r) dr \\ &= 2\pi \oint \left[W_i \int_{r_{shaft}}^{r_i} r dr + \int_{r_i}^{r_o} Z(r) r dr + W_o \int_{r_o}^{r_h} r dr \right] \\ &= 2\pi \oint \left\{ \frac{W_i}{2} (r_i^2 - r_{shaft}^2) + \frac{W_o}{2} (r_h^2 - r_o^2) \right. \\ &\quad \left. + 2Z_o r_t^2 e^{\frac{\rho \omega^2 r_o^2}{2g_o}} \left[e^{-\frac{\rho \omega^2 r_t^2}{2g_o} \left(\frac{r_i}{r_t} \right)^2} - e^{-\frac{\rho \omega^2 r_t^2}{2g_o} \left(\frac{r_o}{r_t} \right)^2} \right] \right\} \end{aligned} \quad (2.4-8)$$

Having determined the dimensions of the disc, the burst speed can be computed assuming the disc yields prior to bursting. For this analysis the disc is divided in half as illustrated in figure (2-19) and the stress is assumed constant over area $2A$. The force acting on this area is composed of the disc load and the blade load. It is assumed that the blade load is evenly distributed on the outer rim. In actual stress analysis of the disc this assumption is an over simplification of the stress, but for preliminary design purposes it will be adequate. A more detailed analysis is covered in reference (7). The force due to the disc is now calculated by taking the force in the direction perpendicular to the area $2A$ caused by the disc load and summing it over the disc half and adding this to the force caused by the blade load. This force is divided by the area $2A$ to get the average tensile stress σ_{ta} . This stress is set equal to the ultimate tensile strength of the material of the disc at its operating temperature. The value of N_b is then calculated by equation (2.4-16)

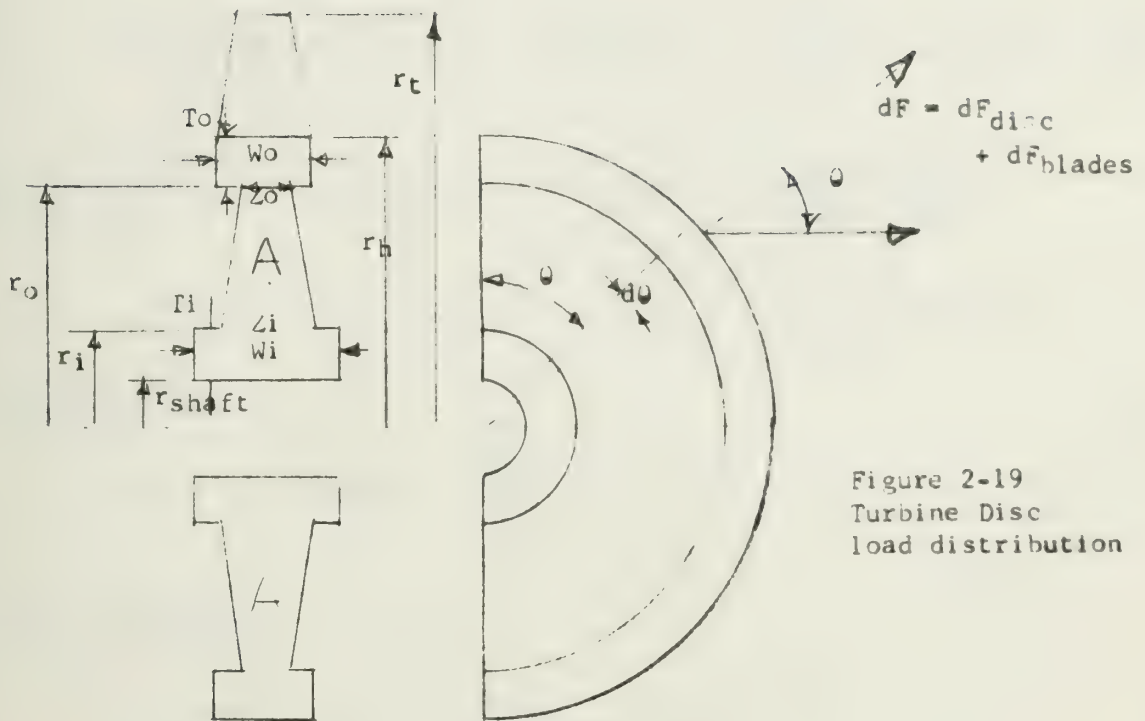


Figure 2-19
Turbine Disc
load distribution

Total area = $2A$

$$A = w_i t_i + \int_{r_i}^{r_o} 2(r) dr + w_o t_o \quad (2.4-9)$$

For disc load

$$dm = \int_{\theta_0}^{\theta} dV \quad (2.4-10)$$

$$dF = (dmr\omega^2)\sin\theta = dmr\left(\frac{2\pi N}{60}\right)^2 \sin\theta \quad (2.4-11)$$

$$\begin{aligned}
 F &= \left(\frac{2\pi N}{60}\right)^2 \int_{\theta_0}^{\theta} \int_{r_{shaft}}^{r_h} 2(r)r^2 \sin\theta d\theta dr \\
 &= \left(\frac{2\pi N}{60}\right)^2 \int_{\theta_0}^{\theta} 2 \int_{r_{shaft}}^{r_h} (r)r^2 dr
 \end{aligned} \quad (2.4-12)$$

$$F = \left(\frac{2\pi N b}{60} \right)^2 \frac{\rho}{g_0} 2 \left[W_i \int_{r_{\text{shaft}}}^{r_i} r^2 dr + \int_{r_i}^{r_o} Z(r) r^2 dr + W_o \int_{r_o}^{r_h} r^2 dr \right]$$

$$= \left(\frac{2\pi N b}{60} \right)^2 \frac{\rho}{g_0} 2 \left[\frac{W_i (r_i^3 - r_{\text{shaft}}^3)}{3} + Z_o \int_{r_i}^{r_o} e^{-a r} r^2 dr + \frac{W_o (r_h^3 - r_o^3)}{3} \right]$$

$Z_o e^{\frac{W_o^2 r_o^2}{2 g_0}} \int_{r_i}^{r_o} e^{-a r} r^2 dr$ is integrated numerically $a = \frac{W_o^2 r_o^2}{2 g_0} \left[\left(\frac{1}{r_t} \right)^2 \right]$

$$\sigma_{ta}(\text{due to disc}) = \frac{F_{\text{disc}}}{2A} \quad (2.4-13)$$

For blade load

$$\sigma_{ta}(\text{due to blades}) = \frac{N \sigma_b (A_b)}{2\pi A} \quad \sigma_b \text{ given by equation (2.4-4)}$$

(2.4-14)

Average disc stress

$$\sigma_{ta}(\text{Disc} + \text{blades}) = \frac{F_{\text{disc}} + F_{\text{blades}}}{2A} \quad (2.4-15)$$

Set $\sigma_{ta} = \text{ULT}$ (ultimate tensile strength of material)

The burst speed N_b can now be calculated. This value will slightly higher than that in equation (2.4-2) due to approximations and arbitrary values selected for some of the disc dimensions.

$$N_b = \left(\frac{30}{\pi} \right) \left[\frac{g_0}{\rho} \frac{\sigma_{\text{ult}} 2A}{2 \int_{r_{\text{shaft}}}^{r_h} Z(r) r^2 dr + \frac{N A_{br}}{2\pi} \left[1 - \left(\frac{r_h}{r_t} \right)^2 - \frac{\alpha_{rt}}{3} \left[1 - \left(\frac{r_h}{r_t} \right)^3 \right] \right]} \right]^{1/2}$$

(2.4-16)

2.5 Having determined the blade and disc dimensions the axial length of the turbine can be calculated. To do this a criteria for spacing between blades must be selected. From reference (6) the values of spacing between blades is 0.2 to 0.5 of the axial chord length. The longer dimension will tend to reduce the flare which would prevent the gas flow to separate from the turbine wall. It would also decrease the chance of inducing vibrational stresses. The shorter dimension would decrease the size and weight of the turbine. For the computer program a value of $\frac{1}{2}$ the average axial chord at mid-height was selected. The computer program will then calculate the axial length and weight of the shaft from station 1 of the 1st stage to station 3 of the last stage. A value of 18° was selected as the stagger angle in the computer program.

Figure 2-20 Axial Spacing between blades

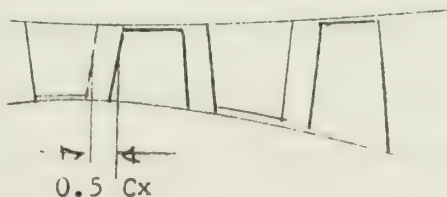
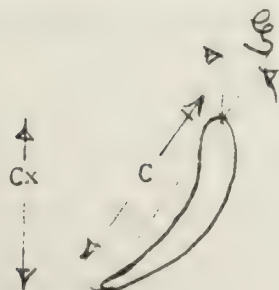


Figure 2-21 Stagger Angle of Turbine blades



Performance Estimation

3.1 The design and off-design characteristics are very important in a complete analysis of a gas turbine since most turbines do not operate only at their design point. Also cycle calculations without actual loss data can not accurately determine the efficiency of a turbine, so this analysis can give a more accurate assessment of the efficiency that can be achieved in the turbine.

The Ainley-Mathieson Method for performance estimation was selected over other methods because it seems to have a wider application and can be easily used for off-design performance calculations. The modifications proposed in reference (8) have been applied. These modifications can reduce the error from $\pm 3\%$ to $\pm 2\%$ for most turbines and are based on modern turbines while the data by Ainley and Mathieson were accumulated over 20 years ago on turbines that existed at that time. The calculation procedure for off-design performance is that proposed by Horlock in reference (12). A description of the Improved Ainley Mathieson Performance Estimation Method will be presented first followed by a description of off-design calculations which use loss coefficients to determine gas properties, isentropic efficiency and static efficiency at design and off-design points.

The computer program used pressure loss coefficient $Y = (P_{01} - P_{02}) / (P_{02} - P_2)$ for the performance analysis. This coefficient is a function of aspect ratio (H/C), pitch to chord ratio (P/C), velocity, Mach number, trailing edge thickness, thickness to chord ratio (t/C), incidence angle, tip clearance, Reynolds number, etc. This coefficient

Loss Coefficient

$$\gamma = \frac{P_{01} - P_{02}}{P_{02} - P_2}$$

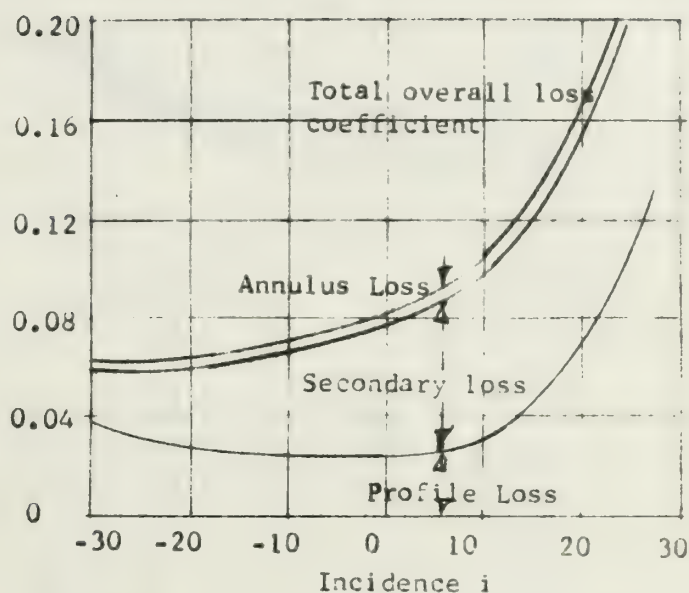


Figure 3-1. Analysis of losses in flow through a row of turbine blades (From D.G. Ainley and G.C.R. Mathieson(1))

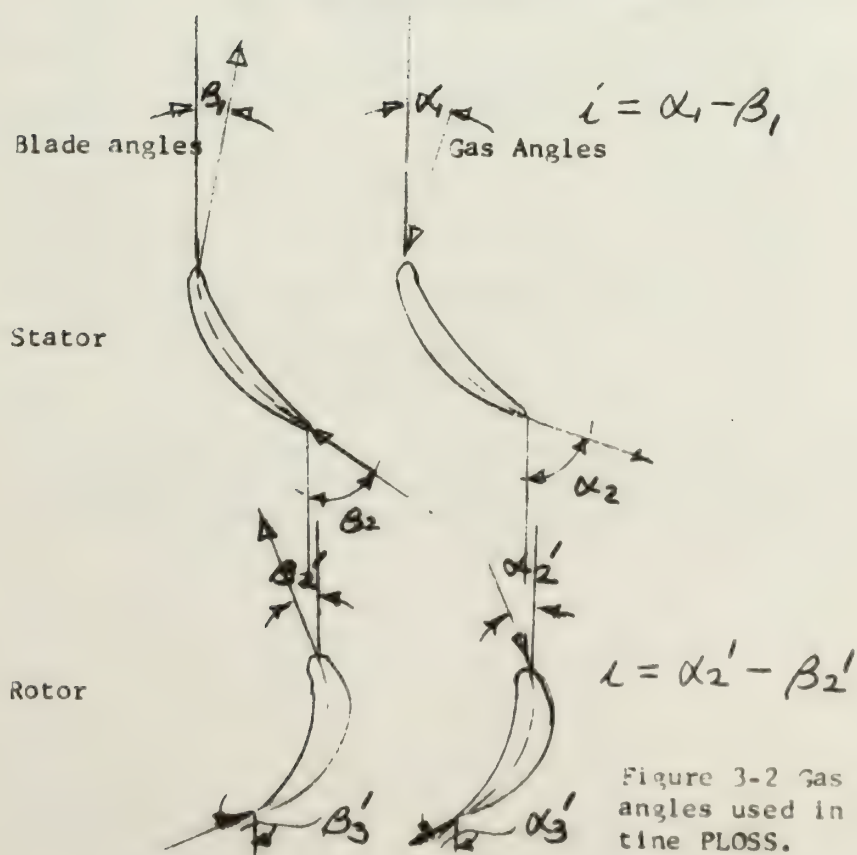
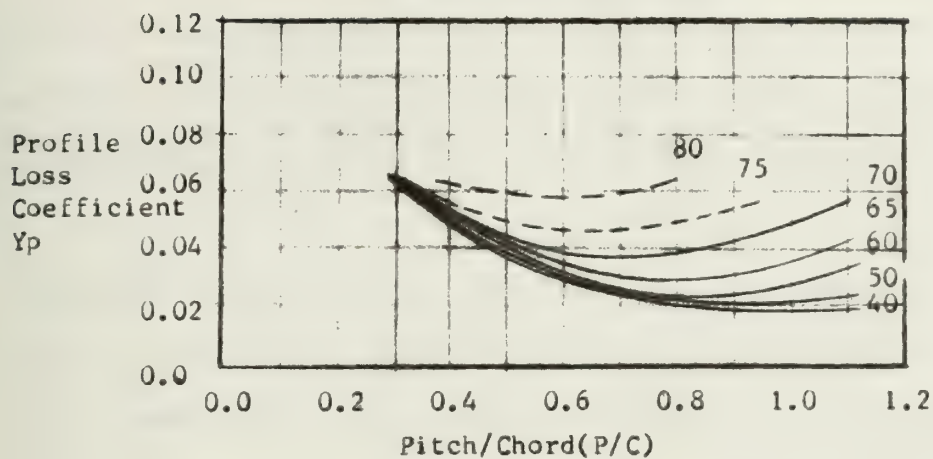
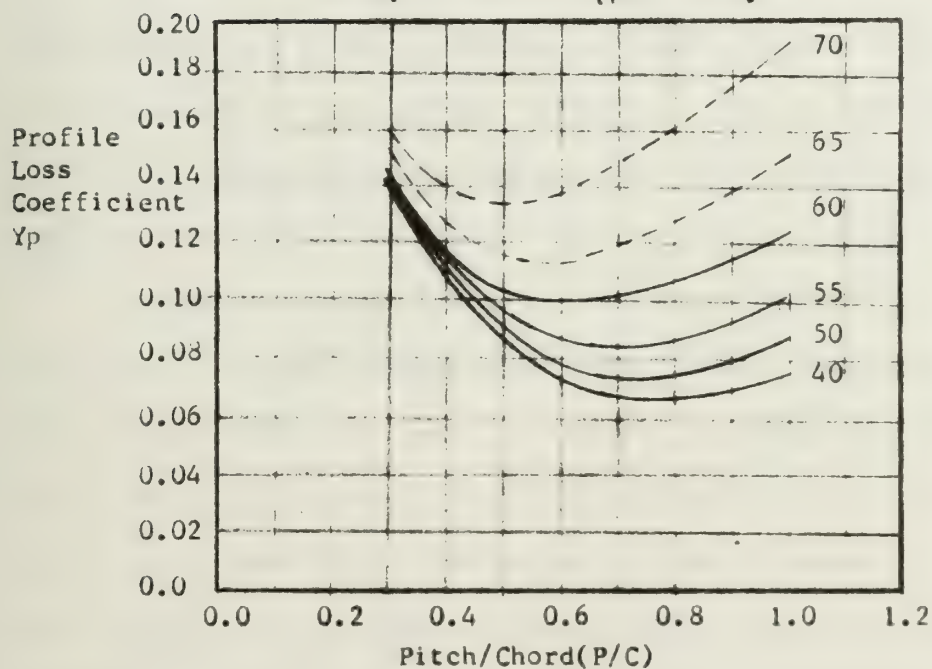


Figure 3-2 Gas and blade angles used in subroutine PLOSS.

Nozzle Blades ($\beta_1 = 0$)

(a)

Impulse Blades ($\beta_1 = \alpha_2$)

(b)

Figure 3-3. Profile-loss coefficients for conventional section blades at zero incidence. $t/c = 20$ per cent; $Re = 2 \times 10^5$; $M = 0.6$. From D.G. Ainley and G.C.R. Mathieson(2))

is broken down into three parts, Y_p (profile loss), Y_s (secondary loss), and Y_k (clearance loss). Figure (3-1) from reference (1) shows the first two, Y_p and Y_s in which the annulus loss is grouped into secondary losses. The modification of the Ainley-Mathieson method mostly affects the secondary and tip clearance loss coefficients, but has an affect on profile loss Y_p for Mach numbers greater than 1, and a Reynolds number correction to the profile and secondary loss coefficients.

3.2 The pressure loss coefficient Y is calculated in subroutine PLOSS. Figure (3-2) shows the sign convention used in this subroutine, where β angles are blade angles and α angles are gas angles. The angles as shown are in the positive direction. The (') implies relative angles. This sign convention and notation applies only to subroutine PLOSS in order to have a standard form for stator and rotor.

Figure (3-3a) and 3-3b) coupled with equation (3.2-1) is used to calculate the profile loss coefficient Y_p for 0 incidence, where $\beta_1 = 0$ values come from figure (3-3a) and $\beta_1 = \alpha_2$ comes from figure (3-3b). The pitch to chord ratio (P/C) is the argument for entering figures (3-3a) and (3-3b). The correction for thickness to chord ratio (t/C) is done in equation (3.2-1) along with the correction for the ratio of inlet blade angle to exit gas angle (β_1/α_2). The value obtained is Y_p for zero incidence ($i=0$).

For off-incidence calculations figures (3-4) a,b,c, and figure (3-5) are used. Figure (3-4) a,b,c determine the stalling incidence so that figure (3-5) can then be used to correct Y_p for off-incidence cases.

$$Y_p = \left\{ Y_p(\beta_1=0) + (\beta_1/\alpha_2)^2 [Y_p(\beta_1=\alpha_2) - Y_p(\beta_1=0)] \right\} \left[\frac{t/C}{0.2} \right] (\beta_1/\alpha_2) \quad (3.2-1)$$

$$\alpha_2(P/C=.75) = \frac{\alpha_2}{(\alpha_2/(\alpha_2(P/C=.75)))} \quad (3.2-2)$$

Figure (3-4b) is entered with the argument of pitch to chord ratio (P/C). The ratio of exit gas angle α_2 to exit angle $\alpha_2(P/C=.75)$ is then determined. It is now possible to determine $\alpha_2(P/C=.75)$. Having determined $\alpha_2(P/C=.75)$, figure (3-4c) is now entered with $\alpha_2(P/C=.75)$ to get the stalling incidence $i_s(P/C=.75)$. Figure (3-4a) is now entered with exit gas angle α_2 and P/C to get $\Delta i_s = i_s - i_s(P/C=.75)$. i_s can then be solved for. Having incidence angle i and i_s the ratio (i/i_s) for entering figure (3-5) to determine ratio $Y_p/Y_p(i=0)$ is known. Having previously solved for $Y_p(i=0)$ in equation (3.2-1) Y_p is now known for off incidence.

$$Y_s = .0334(C/H)(\cos^2 \alpha_2 / \cos^3 \alpha_m) Z \quad (3.2-3)$$

For secondary losses Y_s is computed by equation (3.2-3) from reference (8). Where Z is the Ainley loading parameter from reference (2).

$$Z = \left[\frac{C}{P/C} \right] \frac{\cos^2 \alpha_2}{\cos^3 \alpha_m} \quad (3.2-4)$$

$$\alpha_m = (\alpha_1 - \alpha_2)/2 \quad (\text{for stator}) \quad (3.2-5a)$$

$$\alpha_m = (\alpha_3' - \alpha_2')/2 \quad (\text{for rotor}) \quad (3.2-5b)$$

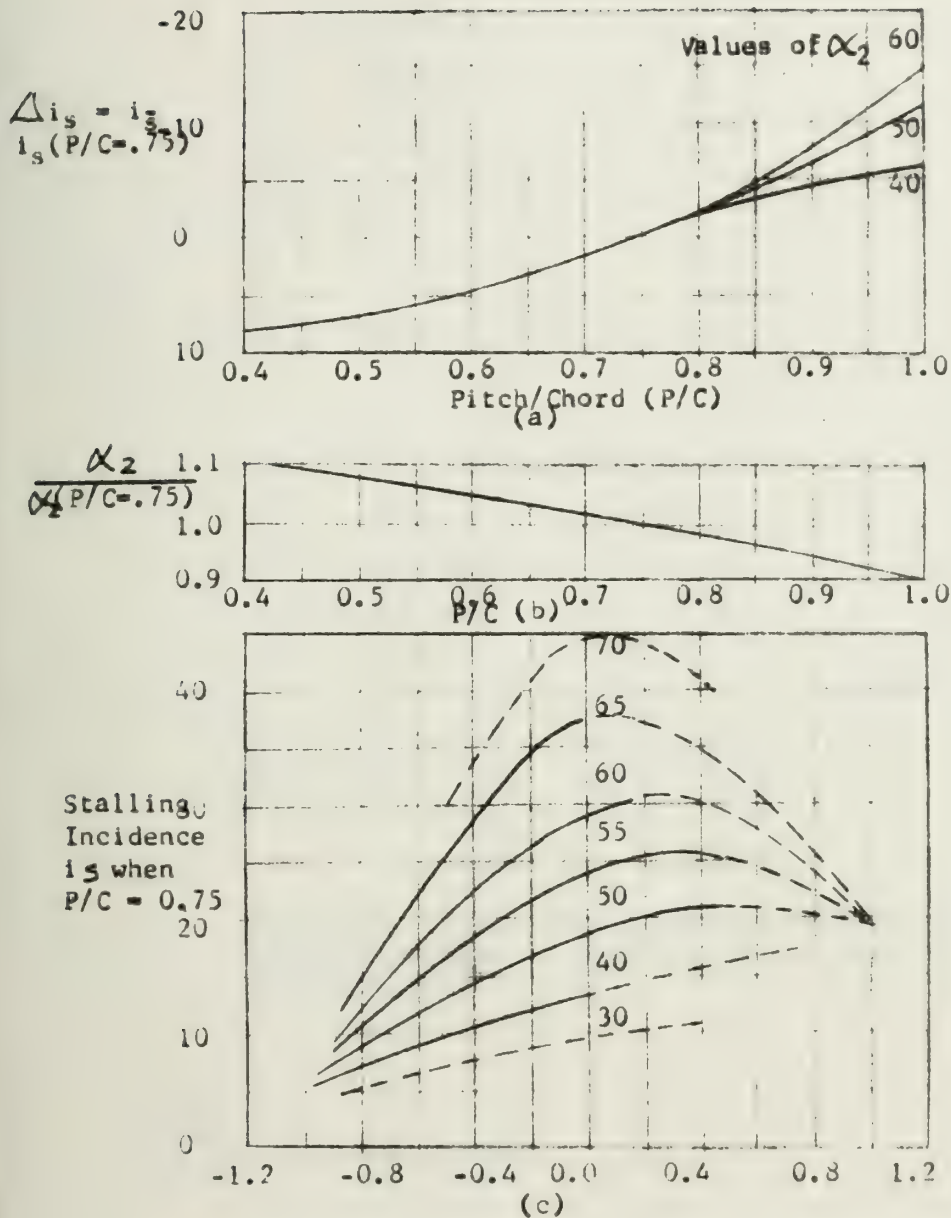


Figure 3-4. Off-design performance of cascades of turbine blades. $Re = 2 \times 10^5$. (a) variation of stalling incidence with pitch-chord ratio; (b) variation of α_2 with p/c; (c) stalling incidence of turbine blade sections when p/c = 0.75. (From D. G. Ainley and G. C. R. Mathieson(2))

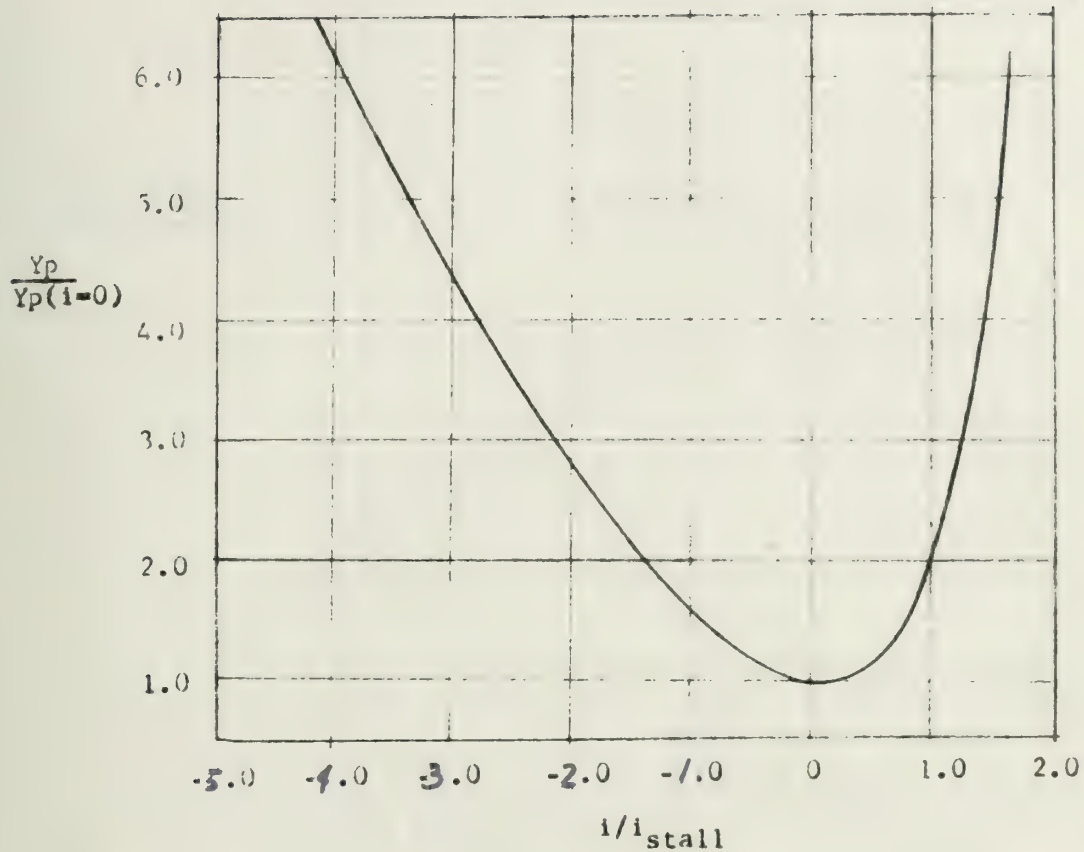


Figure 3-5. Variation of relative profile loss with relative incidence. (From D.G.Ainley and G.C.R. Mathieson(2))

$$\left[\frac{C_L}{P/C} \right] = 2(\tan \alpha_1 + \tan \alpha_2) \quad (3.2-6a)$$

$$\left[\frac{C_L}{P/C} \right] = 2(\tan \alpha_2' + \tan \alpha_3') \quad (3.2-6b)$$

For tip clearance losses γ_k is computed by equation (3.2-7) from reference (3).

$$\gamma_k = B(C/H)(k/C)^{.73} \quad (3.2-7)$$

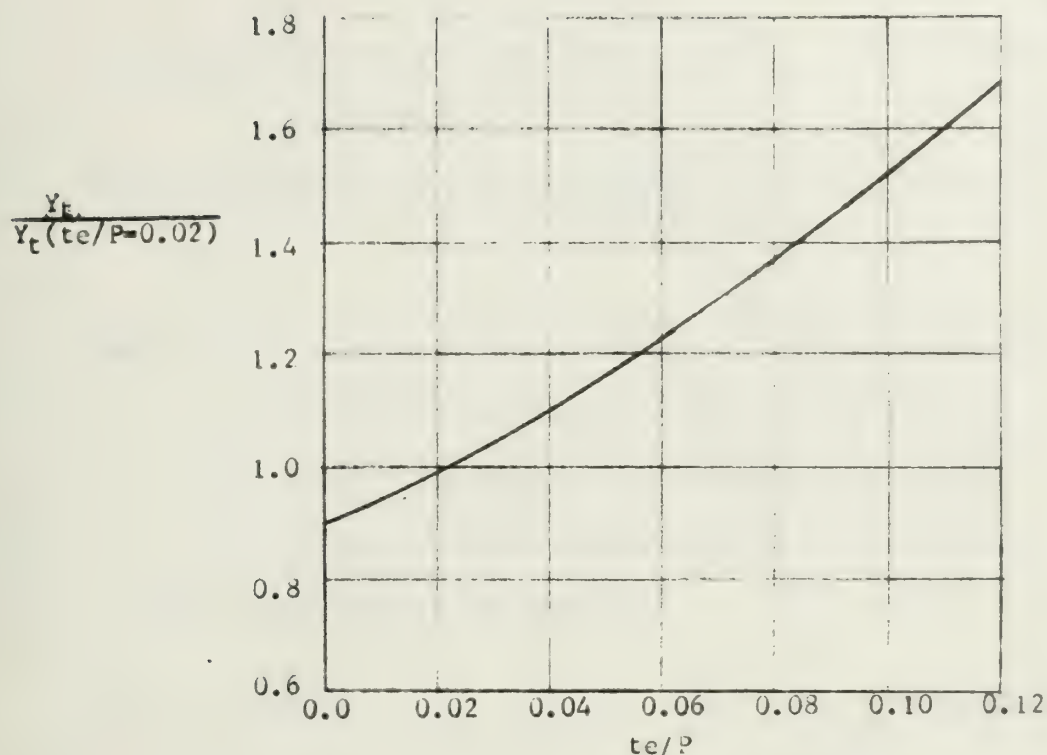


Figure 3-6. Effect of trailing edge thickness on blade coefficients. (From D.G.Ainley and G.C.R.Mathieson(2))

For equation (3.2-7)

$\beta = .37$ for shrouded blades

$\beta = .47$ for plain tip clearance

$\beta = .00$ for stator blades

k = tip clearance

C = chord

H = blade height

Having determined Y_p and Y_s they can be corrected for Mach number and Reynold's number from equation (3.2-8) and (3.2-9) respectively.

$$(Y_p)_{\text{corrected}} = Y_p(1+60(Mn-1)^2) \quad (3.2-8)$$

for $Mn > 1$

$$(Y_p + Y_s)_{\text{corrected}} = (Y_p + Y_s) \left[\frac{Re}{2 \times 10^5} \right]^{-.2} \quad (3.2-9)$$

Now the total loss coefficient Y_t can be computed by equation (3.2-10). This coefficient is now corrected for thickness of trailing edge to pitch ratio (t_e/P) from figure (3-6) to get the total loss coefficient for the blade row. Illustrations of an example calculation can be found in reference (2) and (12).

$$Y_t = Y_p + Y_s + Y_k \quad (3.2-10)$$

3.3 In this section it will be explained how the loss coefficient Y_t solved for in section 3.2 is coupled with compressible flow tables to estimate performance. The method explained is that outlined in reference (12). Another method is explained in reference (2). Calculations will be carried out only at the mean radius which should give a fairly accurate assessment of the turbine performance.

The calculations are started at the inlet of the turbine and worked through each stage where the exit conditions at one blade row is the inlet to the next blade row until the turbine exit of the last stage is reached.

For inlet conditions the values for T_{01} and P_{01} will remain constant, the mass flow and corrected speed $(N/\sqrt{T_{01}})/(N/\sqrt{T_{01}})_{dp}$ will be varied. This will allow the values of efficiency verses pressure ratio and $\left(\frac{\dot{m}\sqrt{T_{01}}}{P_{01}}\right) / \left(\frac{\dot{m}\sqrt{T_{01}}}{P_{01}}\right)_{dp}$ verses pressure ratio to be calculated as illustrated in Figure (3-7).

3.3a Flow through the nozzle.

- (1) Select the mass flow.
- (2) Assume an exit Mach number.

(3) With known blade angles subroutine SFOCRA is entered as in section (2.2) only this time the blade angles will be used to determine the exit gas angles.

(4) From compressible flow tables for the assumed Mach number the values of (P_2/P_{02}) and $(V_2/\sqrt{T_{02}})$ are determined, which will then determine V_2 and T_2

$$V_2 = \sqrt{T_{02}} (V_2/\sqrt{T_{02}}) \quad (3.3-1)$$

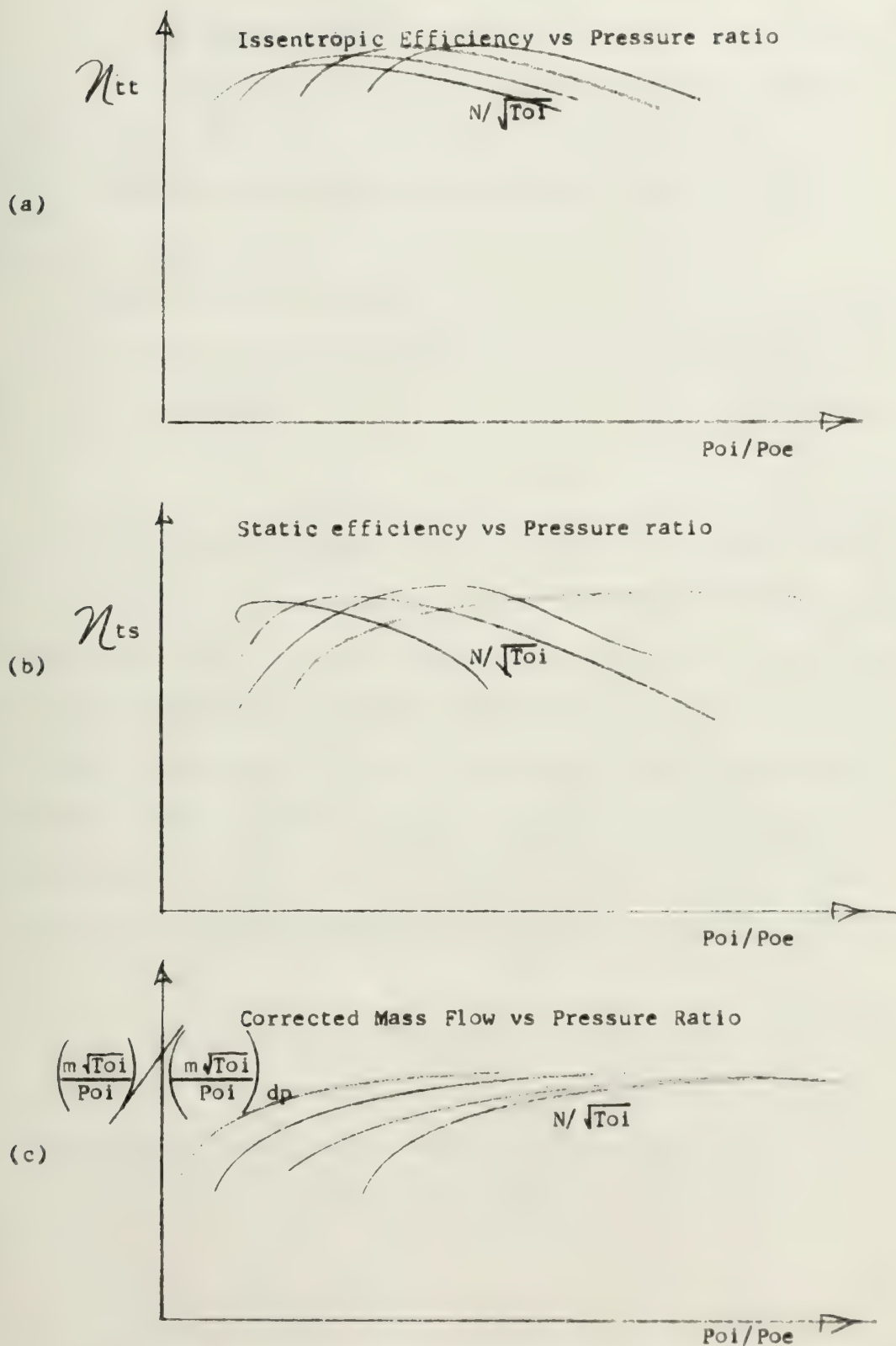


Figure 3-7. Performance Curves for Axial Turbine (a) Isentropic efficiency vs pressure ratio. (b) Static efficiency vs pressure ratio. (c) Corrected mass flow vs pressure ratio.

$$T_2 = T_{02}(P_2/P_{02})^{\frac{\gamma-1}{\gamma}} \quad (3.3-2) \quad 53$$

(5) Subroutine PLOSS is now entered to get Y_t (pressure loss coefficient).

(6) P_{02} can be determined from known values of Y_t and (P_2/P_{02}) from equation (3.3-3)

$$P_{02} = P_{01}/(Y_t(1-P_2/P_{02}) + 1) \quad (3.3-3)$$

(7) The mass flow parameter Q_2 can now be determined.

$$Q_2 = \frac{\dot{m}\sqrt{T_0}}{A_2 P_0} \quad (3.3-4)$$

$$\text{where } A_2 = A_{n2} \cos \alpha_2 \quad (3.3-5)$$

(8) From the compressible flow tables for the value of Q_2 the Mach number can be determined, and if this value is close to the assumed Mach number the inlet conditions to the rotor can be solved for the new values of V_2 , f_2 , P_2 , Mach number, and T_2 will be used to iterate the process until the assumed Mach number approximately equals the calculated value. Also as the Mach number approaches (1) a calculation will be performed to see if the row is choked. (ie; maximum flow for given parameters). This will be covered in section 3.3c.

Now that the exit Mach number has been determined the exit conditions of the nozzle can be determined, which will be the inlet conditions to the rotor. Where V_2 is determined from equation (3.3-1), T_2 from equation (3.3-2) and P_2 from compressible flow data.

$$P_2 = P_{02}(P_2/P_{02}) \quad (3.3-6)$$

$$f_2 = \frac{P_2}{RT_2} \quad (3.36a)$$

3.3b Flow through the rotor

(1) With α_2 and V_2 known the relative rotor angle β_2 can be calculated.

$$\tan \beta_2 = \tan \alpha_2 - \frac{U_{m2}}{V_2 \cos \alpha_2} \quad (3.3-7)$$

$$\text{where } U_m = r_{m2} \omega$$

and determine the incidence angle (i,) where β_2' is the blade angle and $i = \beta_2 - \beta_2'$

(2) The relative velocity w_2 and relative total temperature, can then be calculated.

$$w_2 = (V_2 \cos \alpha_2)^2 (1 + \tan^2 \beta_2) \quad (3.3-8)$$

$$T_{02rel} = T_2 + \frac{w_2^2}{2g_0 J C_p} \quad (3.3-9)$$

(3) $w_2 / \sqrt{T_{02rel}}$ can be calculated, which will be used to enter the gas tables to get P_2 / P_{02rel} which will allow P_{02rel} to be calculated.

(4) The same procedure of assuming exit Mach number and reiterating until the assumed Mach number equals the calculated one is followed. P_{03rel} , β_3 , w_3 , M_{3rel} , and Q_{3rel} will then be known. From this relative Mach number (P_3 / P_{03rel}) can be determined, which will enable P_3 to be solved for. The absolute parameters α_3 , V_3 , P_{03} , T_3 and T_{03} can then be calculated.

$$\tan \alpha_3 = \tan \beta_3 + \frac{U_m}{w_3 \cos \beta_3} \quad (3.3-10)$$

$$\text{where } U_m = r_{m3} \omega$$

$$V_3 = (W_3 \cos \beta_3)^2 (1 + \tan^2 \alpha_3) \quad (3.3-11)$$

$$T_3 = T_{03rel} - \frac{W^2}{2g_0 J C_p} \quad (3.3-12)$$

$$P_{03} = P_3 (T_{03}/T_3)^{\frac{\gamma}{\gamma-1}} \quad (3.3-13)$$

(5) If this is the last stage, efficiency η can be calculated by equations (3.3-14) and (3.3-15). If not the exit conditions will determine the inlet conditions to the nozzle of the next stage. This process will be iterated until the parameters have been determined for every stage in the turbine..

$$\eta_{tt} = \frac{(T_{0i} - T_{0e})}{T_{0i} \left[1 - (P_{0e}/P_{0i})^{\frac{\gamma-1}{\gamma}} \right]} \quad (3.3-14)$$

$$\eta_{ts} = \frac{(T_{0i} - T_{0e})}{T_{0i} \left[1 - (P_e/P_{0i})^{\frac{\gamma-1}{\gamma}} \right]} \quad (3.3-15)$$

3.3c Determination of whether a row of turbine blades have choked is carried out when the Mach number approaches (1). For the computer program this process will be carried out when the Mach number is greater than 0.9.

(1) Select a slightly higher value of $Q \left(\frac{\dot{m} \sqrt{T_0}}{A P_0} \right)$ calling it Q' .

(2) With this Q' enter the compressible flow tables to get corresponding values of P/P_0 and solve for P_{0in}/P_{0out} assuming Y_t remains constant.

(3) Now calculate the new mass flow \dot{m}' . If this is less than \dot{m} the flow has choked. Y_t and Q will be assumed to remain constant with the flow going supersonic at the exit of the blade row. If \dot{m}' is not less than \dot{m} calculations will be resumed as was illustrated in section (3.3a) and (3.3b).

(4) Enter compressible flow tables with Q for supersonic flow to get corresponding values of $(P_{out}/P_{o_{out}})$, $V_{out} / \sqrt{T_{o_{out}}}$, and M_{out} . From equation (3.3-3) determine P_{out} . With equation (3.3-4) A_{out} can be determined. Since $A_{out} = A_{annulus} \cos \alpha$, α can be solved for in equation (3.3-16). The calculations are then resumed in the same manner as in section (3.3a) and (3.3b). The validity of this method depends on whether Q and Y_t remain constant after the turbine blade row has choked.

$$\alpha = \cos^{-1} \left(\frac{\dot{m} \sqrt{T_{o_{out}}}}{A_{annulus} P_{o_{out}}} \right) \quad (3.3-16)$$

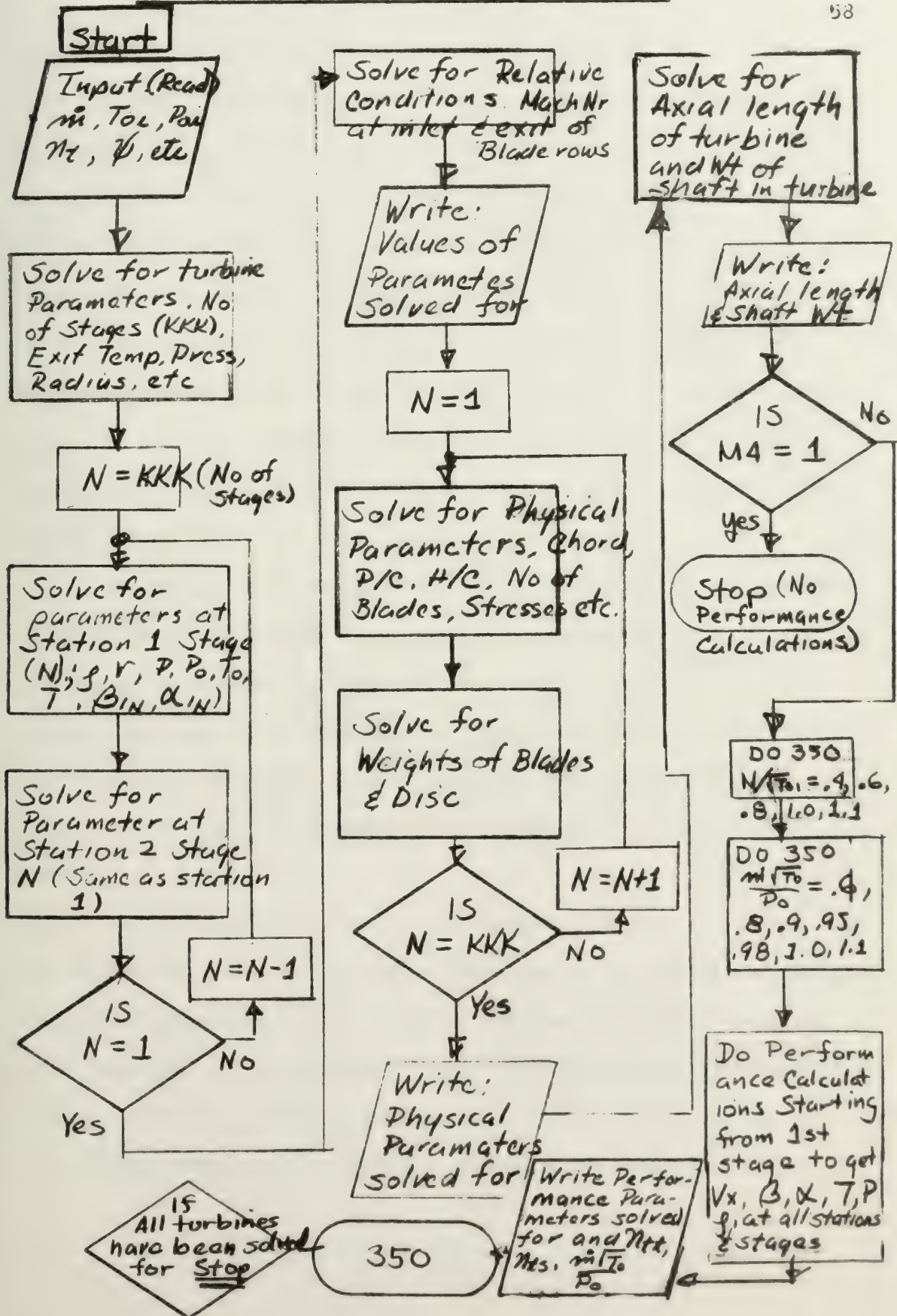
Chapter 4

Operation of Computer Program

4.1 Chapters 1, 2, and 3 described how the calculations are performed in the computer program. This chapter will show how the flow of information is passed through the computer program to get the desired results. This information is not needed to use the computer program since that is covered in Appendix (2).

4.2 The steps of calculations in the computer are in the same sequential order as illustrated in chapters 1 through 3. This is shown in the simplified block diagram figure (4-1)

4.3 The method used in the computer program to interpolate curves is a 4 point Lagrange Polynomials with a worse case of a linear interpolation due to break down of Lagrange Polynomials for monatomic increasing or decreasing function. Subroutine FIG and BK are used to do this interpolating. Subroutine FIG selects the 4 data points closest to the argument X for passing to subroutine BK which actually does the interpolating to get the ordinate Y as an output. If the argument exceeds the limits of the data points the value closest to the argument X is used to get the ordinate Y. The reason this method was chosen over finding formulas by least squares or similar methods is this method can be adapted to changes in data without solving for new formulas or coefficients. The Lagrange Polynomials uses a monatomicly increasing base to get the ordinate Y. If the base is not monatomicly increasing such as

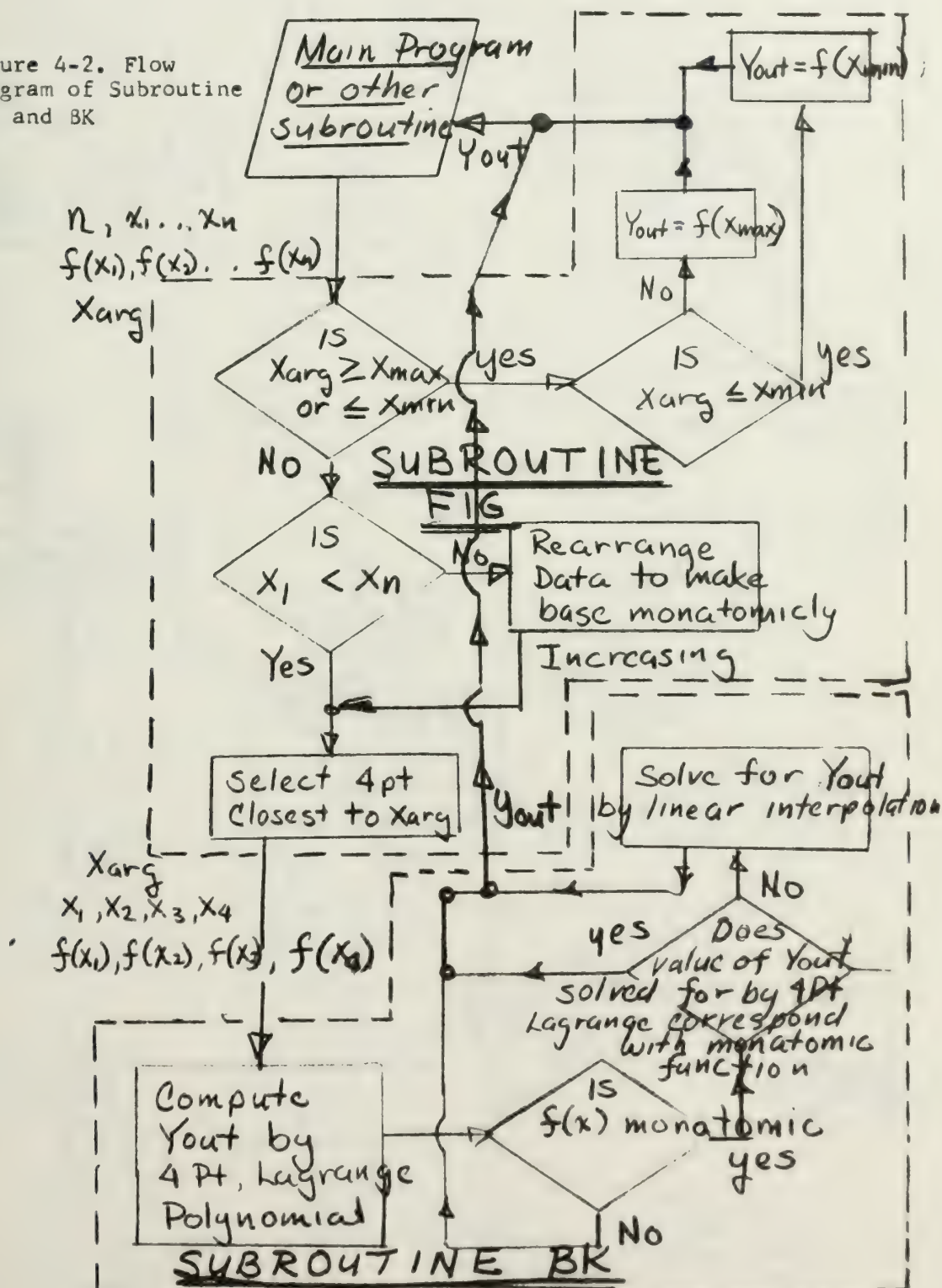


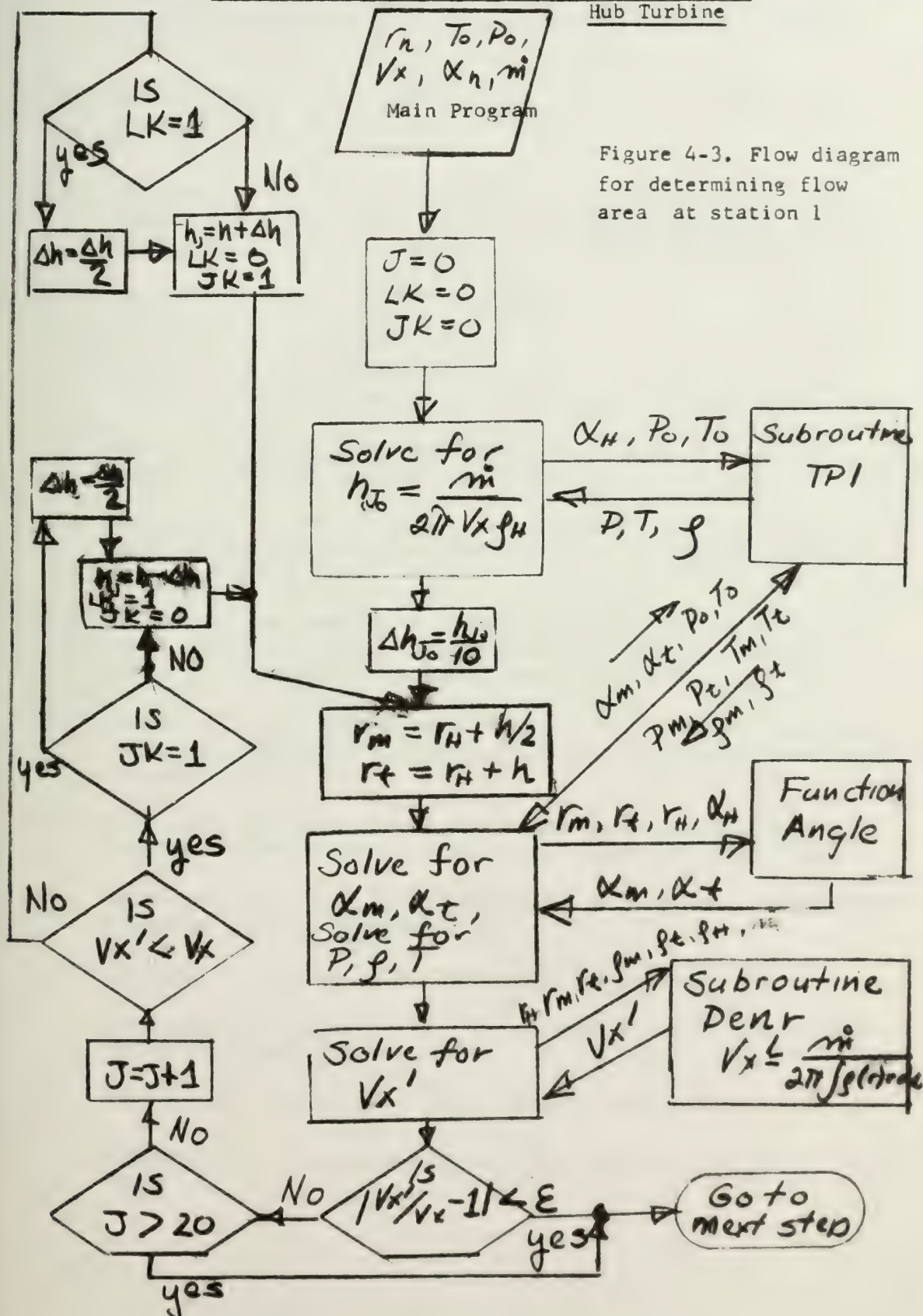
values of Mach Number vs P_s/P_o , subroutine FIG rearranges the base so that the base is monatomically increasing and the ordinate values are in corresponding relationship with the base. For values of base such as $(\dot{m}\sqrt{T_{oi}})/A_{Poi}$ which both increase and decrease the data points are divided into two parts with the part that increases as one set of data points and the part that decreases as another. The ordinate does not have to be a monatomic function. The problem of increasing and decreasing bases can be encountered in the performance estimations because the base and ordinate are interchanged in some of the steps. Figure 4-2 illustrates how subroutine FIG and 3K are used to interpolate data.

4.4 Figure (4-3) illustrates the method the computer program uses to determine the flow area at station 1. In most cases for a value of epsilon (ϵ) = .001 the convergence is reached in less than 10 iterations. This method is used for solving for the exit flow area and also for station 2. The method for solving the other parameters were will documented in chapters 1 thru 3 and will not be repeated here.

4.5 From output generated by the computer program, it is felt that the computer program can be very useful from early preliminary design to start of detailed design, since it has the versatility to accept more input data as the designer firmly sets the characteristics of the Turbine. As for cooling requirements it is felt that compensations in the input data could account for losses attributed to cooling.

Figure 4-2. Flow
Diagram of Subroutine
FIG and BK





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APPENDIX I

Symbols and Notation used in this Text.

SYMBOLS USED IN TEXT

A	Area
C	Chord of turbine blade
C _p	Specific heat (btu/lbm [°] R)
g ₀	Constant in Newtons Law 32.174 (ftlbm)/(lbfsec ²)
h	Blade height
h ₀	Stagnation Enthalpy
H/C	Aspect ratio
i	Incidence angle
I	Moment of inertia
J	Energy conversion factor 778.16 (ftlbf)/(btu)
k	radius of gyration
M	Mach Number
M _{crit}	$V_x / \left(\frac{2}{\gamma+1} g_0 R T_0 \right)^{1/2}$
\dot{m}	Mass Flow
N	Number of Stages
n	Number of Stage
P ₀	Stagnation Pressure
P	Static Pressure
P/C	Pitch to Chord ratio
R	Gas constant (ftlbf)/(lbm [°] R) $\frac{\gamma-1}{\gamma} J C_p$
Q	Corrected mass flow $\dot{m} \sqrt{T_0} / P_0 A$
T ₀	Stagnation Temperature
T	Static Temperature

To	Thickness of Outer Rim
Ti	Thickness of Inner Rim
SM	Non Dimensional Section Modulus
U	Blade Speed
V	Velocity
Vx	Axial Velocity
W	Relative Velocity
Wo	Width of Outer Disc Rim
Wi	Width of Inner Disc Rim
Y	Pressure Loss
R	Radius
r	Radius
t	Thickness of Turbine Blade
t/C	Thickness to Chord Ratio
Z	Section Modulus
Zi	Thickness of Mid Section of Turbine Disc at Inner Radius
Zo	Thickness of Mid Section of Turbine Disc at Outer Radius
Z(r)	Thickness of Disc as a Function of Radius
α	Absolute Gas Angles (except section 3.2)
β	Relative Gas Angles (except section 3.2)
ϵ	Stagger Angle
γ	Ratio of Specific Heats Gamma
η	Efficiency
σ	Stress
ρ	Density
ψ	Psi Loading Coefficient $= \frac{g_o J \Delta h_o}{U_h^2}$

R	Reaction
Ω	Speed (rpm)
ω	Speed (rad/sec)

Subscripts

i	Inlet
e	Exit
t	Total
s	Stage
p	Polytropic
s	Static
t	Tip
m	Mean
h	Hub
c	constant

APPENDIX II

User's Guide To the Computer Program with Comparison of
Turbine Designed with this Computer Program to
one which was Designed by Airesearch. Sample
Input and Output of Turbine
Designed by Computer
Program

II-1 The computer program can be used to calculate physical and thermodynamic properties of an axial flow turbine.. It has an optional ability to do performance calculations. The optional phase is very expensive to operate since it does both design and off design calculations. From operating the computer program and comparing the cycle calculations with performance calculations the cycle calculations are within 3% of performance calculations at design point. For the early preliminary phases of turbine design it is recommended that just the cycle calculations and mechanical design phases be carried out with the computer program punching output cards to have data to perform the performance calculations at a later time

A second program which reads in the punched output cards in the same order as they were punched in the first program is provided to do only performance calculations. This allows the performance estimations to be carried out with running the complete program again.

For future references program #1 will refer to the complete computer program while program #2 will refer to the program which only does performance estimations. The complete computer program can not be operated in computers which have less than 210 k of memory.

II-2 Pages 71 to 76 show how the computer cards are to be punched. The 1st card is a label which will be printed out in the same content as it was entered on the computer card. Cards 2 thru 4 are the input data cards of known turbine parameters. Cards 5 thru NNN are the compressible flow tables and viscosity of the gases with corresponding temperatures. Starting from card 5, 1st the Mach Numbers are read in, 2nd the corrected mass flow $\dot{m}\sqrt{T_0}/P_{0A}$, 3rd the pressure ratio P/P_0 ,

4th the corrected velocity $V/\sqrt{T_0}$, 5th the temperatures which correspond to the viscosity data, 6th the viscosity data. Page 77 is a set of compressible flow tables and viscosity data for air.

II-3 Pages 73 thru 82 are comparison data of computer program designed turbine with one which was designed by Airesearch reference (20). As can be seen from the data some of the parameters are different because of lack of knowledge of some of the design parameters which was used in the design of the Airesearch Turbine, also some simplifications are incorporated into the computer program for solutions of output parameters.

II-4 A list of sample output and input is provided from page 83 to 93 for turbine which was compared with Airesearch turbine. As can be seen on page 91 of output data, the performance data is very similar to the cycle calculations at design point. This justifies the argument of doing only cycle calculations and mechanical calculations when off design data is not required. Figure II-3 is a plot of the data points from performance calculations. Where some of the curves are discontinuous is caused by a limit of 10 iterations for solution at a blade row..

Input Data Cards

* (Must be specified)

(If value of parameter is not known place value to right of #
on computer card)

Card 1

Column 1-80

Alphanumeric label (ie, "Gas Tur-
bine number 1")

Blank Card

Card 2

Column 1*

M1 (Integer)

0 compressible flow data to be
entered.

1 No compressible flow data to be
entered.

Column 2 *

M2 (Integer).

0 Power in horse power specified.

1 Pressure ratio specified.

Column 3 and 4

Blank

Column 5 *

M (Integer) Type of turbine.

1 Constant hub

2 Constant mean

3 Constant tip

Note

Where instructions inform user to place dummy values on cards these values are used to inform computer that they are not specified and to perform optional calculations to determine the parameter.

Column 6-10 (leave blank if M1 = 1)	KLM (Integer right justified) Number of compressible flow elements to be entered. (ie number of Mach Numbers to be entered) (Maximum 100)
Column 11-15 (leave blank if M1 = 1)	KTM (Integer right justified) Number of Temperatures at which values of (viscosity) $\times 10^7 \frac{\text{lbm}}{\text{sec ft.}}$ is defined.
Column 16-20 *	Gamma γ (F5.3)
Column 21-25 *	CP (Btu/lbm °R) (F5.3)
Column 26-35 (lbm/sec ft)	Power (Horse Power) (F10.1) if M2 = 0 this must be specified # 9000.
Column 36-45 *	W (mass flow) (lbm/sec) (F 10.3)
Column 46-50 *	TC (thickness to chord ratio) (F5.3), normally about # 0.2
Column 51-55 *	TET (thickness of trailing edge to pitch ratio) (F5.3) normally about # 0.02
Column 56-60 *	TIPCLA (Tip clearance) (in) (F5.3) normally about # 0.015

Card 3

Column 1-5 *

 ρ PSI (F5.3)

Normally between 1.5 and 2.9

2.0

Column 6-10 *

RTE (Max Tip radius in ft at exit)

(F5.3)

Column 11-15 *

Ratio T (R_{te}/R_{he}) Ratio of Tip to
hub at exit (F5.3) # 1.5

Column 16-20

 R_{hub} (exit) Reaction at hub (F5.3)

normally between .00 to 0.4 # .05

Column 21-30 *

OMEGA (RPM) (F 10.0)

Column 31-32

KLM1 (INTEGER) Right justified if

Compressible flow data provided

the value at which $\frac{\dot{m} \sqrt{T_{0c}}}{P_{0c} A}$

is a maximum in (for

sample problem KLM1 = 50) (Maximum 70)

Column 33-39

Blank

Column 40 *

M4 (Integer)

0 performance calculations to be per-
formed.1 No Performance Calculations to be
performed.

Column 41-45 *

SPECWT (Specific Weight of turbine
disc and blades (lb/in³) (F5.3)

normally about 0.3

Column 46-50 *

Y MOD (Young's Modulus psi) (f 15.0)
Normally about # 30000000.

Column 61-70 *

C1 (Value of allowable blade stress
in psi) (F 10.0) Normally about
50000.

Column 71-80 *

C2 (Value of allowable blade steady
stress in PSI) (F 10.0) Normally
about # 100000.

Card 4

Column 1-5 *

ENT (Value of total efficiency
desired. # .9

Column 6-15

Radius of Shaft in ft (F 10.5) # - 0.1
if not known

Column 16-20

P ratio (Pressure Ratio) (F5.3)
if M2 = 1 this must be specified # 1.5

Column 21-30 *

ULTSRE (Ultimate Tensile Strength of
Material of Disc (psi) Value for disc
stress is calculated from this. (F10.0)

$$\sigma_{avg} = \frac{.75 \text{ ULTSRE}}{2} \quad (1.2)$$

Normally about # 130000.

Column 31-35

ALPHAH (Exit angle in radians) (F5.3)
Normally between (0.0 to - .3) # - .1

Column 36-45

EMACH = $\frac{V_x}{\sqrt{\frac{2}{\gamma+1} g_o R T_o}}$ Normally Between
.2-1.0 #-.5

0 Normal Blades

1 Shrouded Rotor Blades. #0

Column 47-48 *

NKL (Number of turbines to be designed by computer program. (Integer right justified) All sets of cards except for last turbine design will have 0 in column 48. The last set of cards will have the number of sets that were entered. This allows the computer to stop after completing the last turbine design.

Column 49-50

N (Number of stage Integer right justified) (Maximum 10) If this value is not known, enter 0 and computer will solve for number of stages. #0

Column 51-55 *

AREARA (F5.4)

Tip to base area ratio of turbine blade normally between .25 to .35

.25

Column 56-60

T00 (Gives radial thickness of outer rim of disc. by $T_o = W_o/T00$ normally between 2 to 4 (F5.3) # - 5.

Cards 5-NNN

These cards are optional. If performance calculations are to be performed they must be furnished. If more than one turbine is to be designed compressible flow data must only be furnished for 1 st turbine and the value of M1 on card 2 for each following turbines will be "1". If the compressible flow parameters are different a set must be furnished. A set of compressible flow data and viscosity data for air are provided with the computer program.

Card 5 - N	RMAC (Mach Numbers) (20F4.2) same number as KLM
Card (N + 1)-K	WTAP (10F8.5) $\frac{\dot{m}\sqrt{T_o}}{P_o A} \frac{1 \text{bm}^{\circ} R_1}{\text{sec lbf}}$ Same numbers as KLM
Card (K+1) -J	PSPT (Ps/Po) (10F8.5) same number as KLM
Card (J+1)-L	VELTOT ($V/\sqrt{T_o}$) (10F8.5) $\frac{(\text{ft/sec})}{\sqrt{T_o}}$ same number as KLM
Card (L + 1) - LL	TEM ($^{\circ}R$) (20F5.0) same number as KTM temperature values that cor- respond with viscosity data
Card (LL + 1)-NNN	V (Viscosity of fluid (lbm.sec/ft) \times 10^7) (20F5.0)

Compressible Flow Tables for Combustion Products

 $\frac{\text{lbm}}{\text{sec}} \sqrt{\frac{\text{ft}}{\text{sec}}}$
 $(\text{ft/sec}) \sqrt{\frac{\text{ft}}{\text{sec}}}$

reference (2)

MACH NO	W*SQRT(T)/(A*P)	PS/PT	V/SQRT(T)
0.02000	0.01811	0.99973	0.96151
0.04000	0.03596	0.99893	1.91184
0.06000	0.05388	0.99760	2.86589
0.08000	0.07171	0.99574	3.82740
0.10000	0.08945	0.99335	4.77028
0.12000	0.10706	0.99040	5.72284
0.14000	0.12448	0.98700	6.67094
0.16000	0.14154	0.98317	7.61008
0.17000	0.15885	0.97870	8.56787
0.20000	0.17562	0.97380	9.50329
0.22000	0.19226	0.96835	10.45287
0.24000	0.20849	0.96260	11.36668
0.26000	0.22446	0.95630	12.31328
0.28000	0.24049	0.94950	13.26734
0.30000	0.25612	0.94200	14.22884
0.32000	0.27135	0.93440	15.14564
0.34000	0.28590	0.92650	16.04750
0.36000	0.30026	0.91800	16.99411
0.38000	0.31435	0.90920	17.90344
0.40000	0.32763	0.90030	18.82024
0.42000	0.34118	0.89080	19.75192
0.44000	0.35396	0.88100	20.63144
0.46000	0.36627	0.87060	21.54077
0.48000	0.37821	0.86000	22.45010
0.50000	0.38948	0.84950	23.33708
0.52000	0.40061	0.83860	24.27625
0.54000	0.41108	0.82720	25.16321
0.56000	0.42114	0.81570	26.05019
0.58000	0.43067	0.80400	26.95207
0.60000	0.43979	0.79220	27.80922
0.62000	0.44838	0.78020	28.65892
0.64000	0.45669	0.76780	29.54590
0.66000	0.46421	0.75550	30.41052
0.68000	0.47159	0.74270	31.29750
0.70000	0.47796	0.73070	32.09502
0.72000	0.48433	0.71800	32.94472
0.74000	0.49010	0.70520	33.80190
0.76000	0.49560	0.69140	34.65160
0.78000	0.50030	0.67910	35.47894
0.80000	0.50459	0.66650	36.29883
0.82000	0.50815	0.65500	37.05164
0.84000	0.51170	0.64080	37.93861
0.86000	0.51479	0.62760	38.76595
0.88000	0.51734	0.61410	39.60822
0.90000	0.51948	0.60250	40.39828
0.92000	0.52123	0.58900	41.14365
0.94000	0.52257	0.57550	42.03061
0.96000	0.52337	0.56500	42.70889
0.98000	0.52391	0.55100	43.41698
1.00000	0.52418	0.54000	44.16234
1.02000	0.52391	0.52800	44.98222
1.04000	0.52337	0.51700	45.69031
1.06000	0.52257	0.50400	46.36113
1.08000	0.52136	0.49100	47.17357
1.10000	0.51989	0.48000	47.85184
1.12000	0.51801	0.46820	48.59720
1.14000	0.51599	0.45650	49.34256
1.16000	0.51358	0.44530	50.02827
1.18000	0.51076	0.43330	50.74382
1.20000	0.50795	0.42250	51.39229

for $\gamma = 1.333$

$$g = 37.174 \frac{\text{ft}}{\text{sec}}$$

$$C_p = 0.274 \frac{\text{Btu}}{\text{lbm}^\circ\text{R}}$$

Viscosity of
Air from Reference (14)

Temperature (°R)	Viscosity $\times 10^7 \frac{\text{lbm}}{\text{sec ft}}$
---------------------	---

400	100
600	135
800	166
1000	192
1200	218
1400	242
1600	269
1800	284
2000	302
2200	320
2400	338

COMPARISON OF COMPUTER PROGRAM TURBINE DESIGN WITH
AIRESEARCH 9000 HP FREE VORTEX AXIAL FLOW TURBINE

78

Inlet Conditions

Gas	Air
Poi(psi)	312.5
Toi($^{\circ}$ R)	1767.8
m(lb/sec)	119.421

Design Parameters

ψ	1.8
α_e	0.0

Assumed Design Parameters

η_{tt}	0.9168
$C_p(\text{btu/lb}^{\circ}\text{R})$.2715
Gamma γ	1.333

Results

Figure II-1 is a comparison of the turbine wheel dimensions.

Figure II-2 is a comparison of design point performance calculations of total and static efficiencies as a function of tip clearance.

COMPARISON OF OUTPUT DATA FROM COMPUTER PROGRAM WITH
AIRESEARCH GAS TURBINE

Parameter	Airesearch	Computer Program
Toe($^{\circ}$ R)	1571.4	1554.0
M_{crit} at exit	-	.292
Poe (psi)	186.0	177.3

Velocity Diagrams

hub(station 1)

radius(ft)	.4383	.429
β_1 (deg)	-62.02	-63.96
α_1 (deg)	0.0	0.0

tip(station 1)

radius(ft)	.530	.429
β_1 (deg)	-67.15	-70.93
α_1 (deg)	0.0	0.0

Hub(station 3)

radius(ft)	.4383	.429
β_3 (deg)	-60.95	-63.96
α_3 (deg)	0.0	0.0

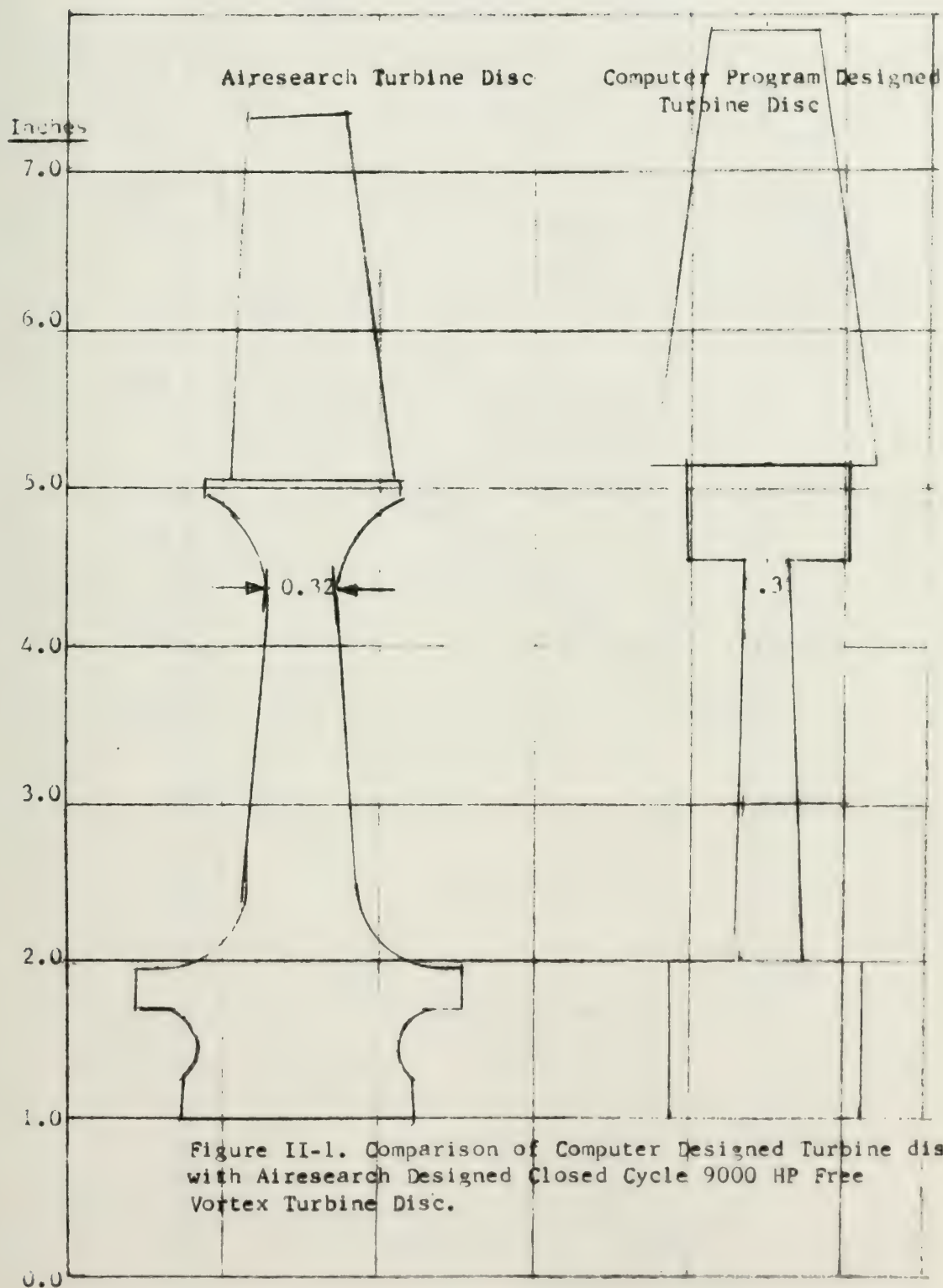
Tip(station 3)

radius(ft)	.6215	.636
β_3 (deg)	-68.26	-72.99
α_3 (deg)	0.0	0.0
δ_{cf} (psi)	37,400.	37,396.

Wheel Weight(lbf)	18.7	21.46
-------------------	------	-------

Conclusions:

Most of the discrepancies in parameter values can be attributed to different input values for the computer model. This is caused by not knowing the assumptions or definitions that were used in designing the Airsearch Gas Turbine. From this limited comparison it is felt that the computer model can be used for designing marine gas turbines since it has the versatility to change parameters which are attributed to specific turbines. For high performance turbines (aircraft) it should be compared with known aircraft turbines to verify the validity of the computer program and adjust the variable parameters to match them.



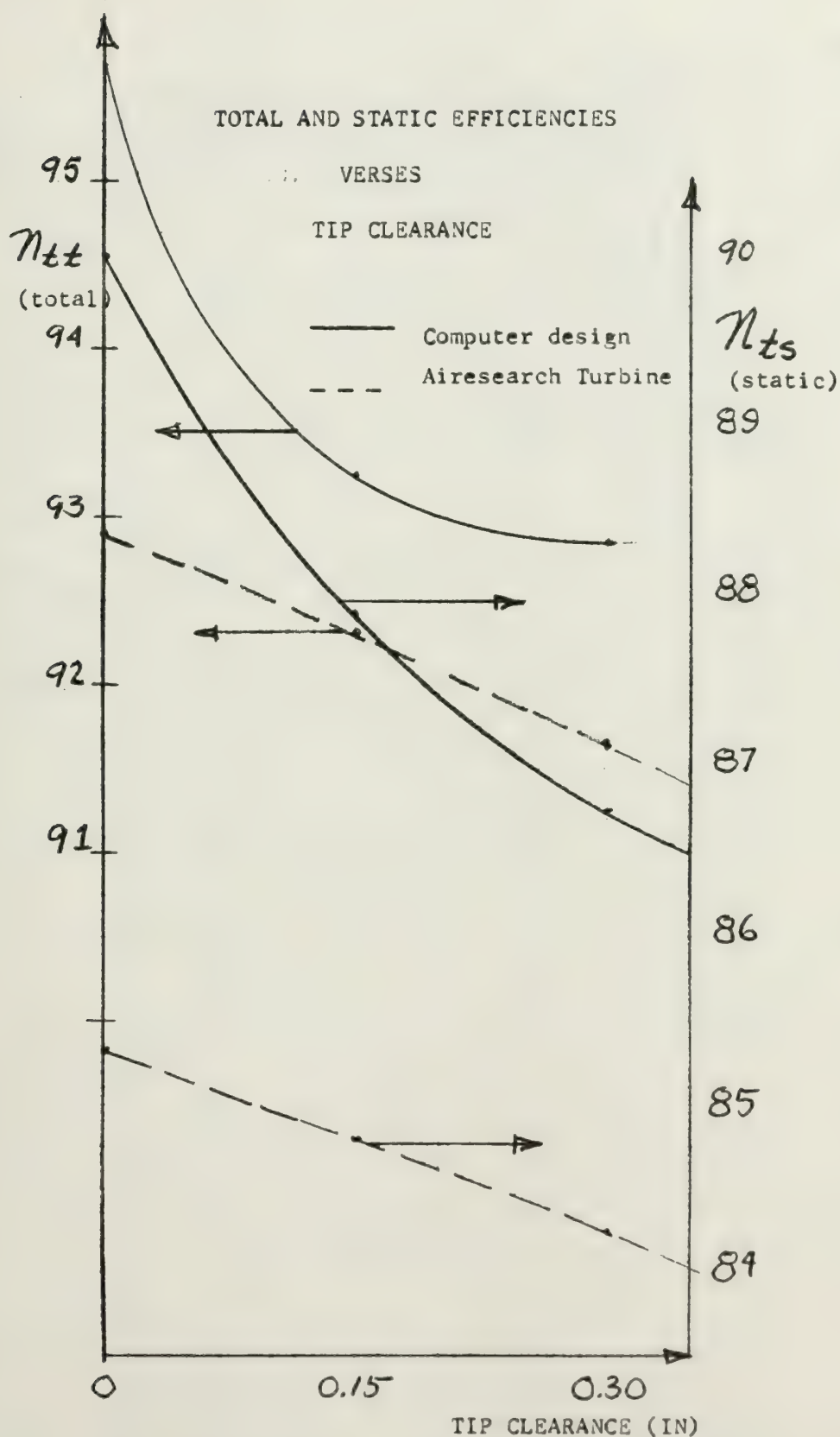


Figure II-2, Comparison of Efficiencies verses Tip Clearance of Performance Calculations.

Input Data Cards to Run 9000 HP turbine with Performance
 Calculations and 136713 HP turbine with Performance Calculations

Card

SENTRY 4 Control Card

Turbine #1

9000 HP SINGLE STAGE GAS TURBINE ENGINE

00 1 60 111.333.2715000. 119.42 .2 .02 .015 1767.8 312.48

1.8 .62161.416.1 20000. 00 0.257 29000000. 20000. 93000.

.9168.04333 1.640130000. .000 .292 0 0 1.25 .1

0.020.040.060.080.100.120.140.160.170.200.220.240.260.280.300.320.340.360.380.400

M 0.420.440.460.480.500.520.540.560.580.600.620.640.660.680.700.720.740.760.780.800

0.820.840.860.880.900.920.940.960.980.100.101.041.061.081.101.121.141.161.181.200

0.01811 0.03596 0.05348 0.07171 0.08945 0.10706 0.12448 0.14154 0.15885 0.17562

0.19226 0.20849 0.22446 0.24049 0.25612 0.27135 0.28590 0.30026 0.31435 0.32763

0.34118 0.35386 0.36627 0.37821 0.38948 0.40061 0.41108 0.42114 0.43067 0.43979

0.44838 0.45569 0.46421 0.47159 0.47796 0.48433 0.49010 0.49560 0.50030 0.50450

0.50815 0.51170 0.51479 0.51734 0.51948 0.52123 0.52257 0.52337 0.52391 0.52414

0.52391 0.52337 0.52257 0.52136 0.51989 0.51801 0.51599 0.51358 0.51076 0.50795

0.99973 0.99893 0.99760 0.99574 0.99335 0.99040 0.98700 0.98317 0.97870 0.97380

0.96835 0.96260 0.95630 0.94930 0.94200 0.93440 0.92650 0.91800 0.90920 0.90030

0.89080 0.88100 0.87050 0.85900 0.84650 0.83800 0.82720 0.81570 0.80400 0.79220

0.774020 0.76780 0.75550 0.74270 0.73070 0.71800 0.70520 0.69140 0.67910 0.66650

0.65500 0.64080 0.62760 0.61410 0.60250 0.58900 0.57550 0.56500 0.55100 0.54000

0.52800 0.51700 0.50400 0.49100 0.48000 0.46820 0.45650 0.44530 0.43330 0.42250

0.96151 1.91184 2.85589 3.82740 4.77028 5.72284 6.67094 7.61008 8.56787 9.50320

10.4528711.3666812.3132813.2673414.2288415.1458416.0475016.9941117.9034418.82020

19.7519220.6314421.5407722.4501023.3370424.2762525.1632126.0501926.9520727.80920

28.6589229.5459030.4105231.2975032.0950232.7447233.8019034.6516035.4789436.29883

37.0516437.9386138.7654539.5042240.3942841.1436542.0306142.7088943.4169844.16234

44.9822245.6903146.3511347.1732747.8518448.2972049.3425650.0282750.7438251.39220

2400. 600. 800. 1000. 1200. 1400. 1600. 1800. 2000. 2200. 2400. Temperature

1100. 135. 155. 175. 214. 242. 264. 284. 302. 320. 338. Viscosity x 10⁷

13673 HP SINGLE STAGE AIRCRAFT GAS TURBINE ENGINE Turbine #2

10 2 60 111.333.274 13573. 100. .2 .02 .015 2960. AR.2

2.8 1.034 2.0 0.0 13555. 00 2.257 29000000. 45000. 93000.

0.9 .1 6. 130000. -.5 -.25 0 2 1.25 .1

Sample Input Data Cards

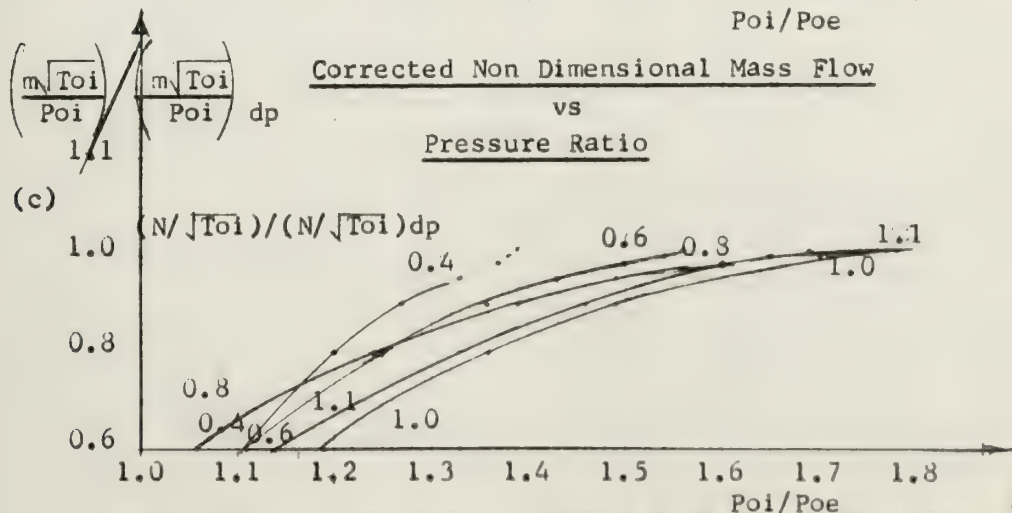
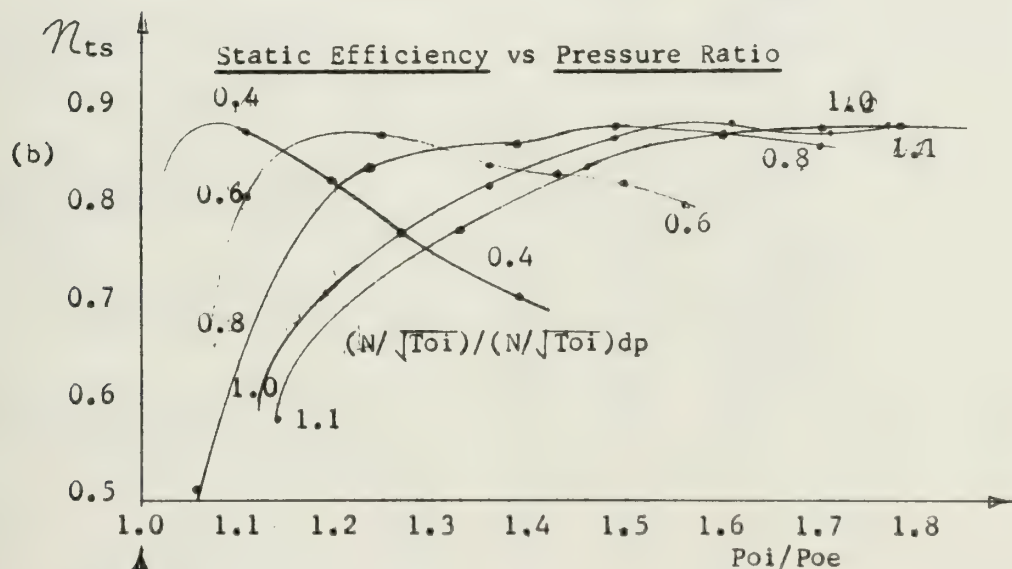
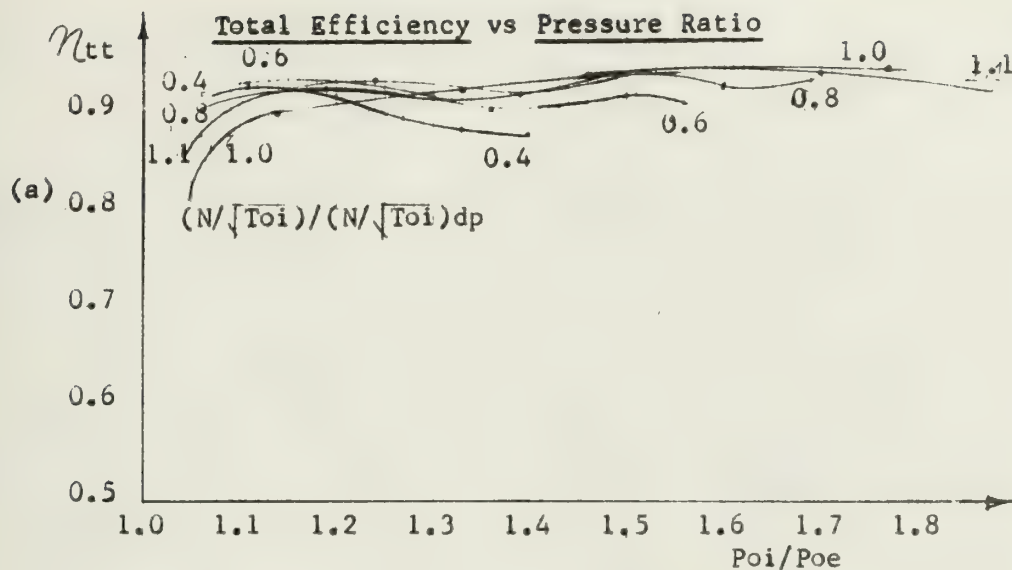


Figure II-3. Performance Curves from data calculated with Computer Program. (a) Isentropic Efficiency (b) Static Efficiency. (c) Corrected mass flow.

9000 HP SINGLE STAGE GAS TURBINE ENGINE

MASS FLOW (LBS/S) POWER (HP) INLET TEMP (°R) EXIT TEMP (°R) INLET PRESS (PSI) EXIT PRESS (PSI) ISSN EFFIC POLY EFFIC
 119.82 9600 1768 1554 312.480 177.270 0.917 0.911

GREGG (AREA) GAMMA CP (BTU/LB-°R) REACTION HUB PSI W (FT/S) CRIT MACH NO AT EXIT
 20000 1.333 0.2715 0.100 1.500 439.2 0.2920

| LOCATION | STATION | STAGE | TOTAL TEMP | STATIC TEMP | TOTAL PRESS | STATIC PRESS | DENSITY | ALPHA | REA | REACTION | RADIUS |
|----------|---------|-------|------------|-------------|-------------|--------------|---------|-------|--------|----------|--------|
| 1 | 1 | 1 | 1767.80 | 1752.61 | 341.88 | 332.56 | 0.4794 | 0.00 | -52.98 | 0.100 | 0.279 |
| 2 | 1 | 1 | 1767.80 | 1752.61 | 341.88 | 332.56 | 0.4794 | 0.00 | -52.98 | 0.100 | 0.279 |
| 3 | 1 | 1 | 1767.80 | 1752.61 | 341.88 | 332.56 | 0.4794 | 0.00 | -52.98 | 0.100 | 0.279 |
| 4 | 1 | 1 | 1767.80 | 1752.61 | 341.88 | 332.56 | 0.4794 | 0.00 | -52.98 | 0.100 | 0.279 |
| 5 | 1 | 1 | 1767.80 | 1752.61 | 341.88 | 332.56 | 0.4794 | 0.00 | -52.98 | 0.100 | 0.279 |
| 6 | 1 | 1 | 1767.80 | 1752.61 | 341.88 | 332.56 | 0.4794 | 0.00 | -52.98 | 0.100 | 0.279 |
| 7 | 1 | 1 | 1767.80 | 1752.61 | 341.88 | 332.56 | 0.4794 | 0.00 | -52.98 | 0.100 | 0.279 |
| 8 | 1 | 1 | 1767.80 | 1752.61 | 341.88 | 332.56 | 0.4794 | 0.00 | -52.98 | 0.100 | 0.279 |
| 9 | 1 | 1 | 1767.80 | 1752.61 | 341.88 | 332.56 | 0.4794 | 0.00 | -52.98 | 0.100 | 0.279 |
| 10 | 1 | 1 | 1767.80 | 1752.61 | 341.88 | 332.56 | 0.4794 | 0.00 | -52.98 | 0.100 | 0.279 |

LOCATION STAGE RELATIVE TOTAL TEMP FOR ROTOR REAL MACH NO FOR STATOR RELATIVE MACH NO FOR ROTOR

| INLET | EXIT | INLET | EXIT |
|---------|---------|--------|--------|
| 1613.25 | 1613.25 | 0.5819 | 0.4493 |
| 1613.25 | 1613.25 | 0.5819 | 0.4493 |
| 1613.25 | 1613.25 | 0.5819 | 0.4493 |
| 1613.25 | 1613.25 | 0.5819 | 0.4493 |
| 1613.25 | 1613.25 | 0.5819 | 0.4493 |
| 1613.25 | 1613.25 | 0.5819 | 0.4493 |
| 1613.25 | 1613.25 | 0.5819 | 0.4493 |
| 1613.25 | 1613.25 | 0.5819 | 0.4493 |
| 1613.25 | 1613.25 | 0.5819 | 0.4493 |
| 1613.25 | 1613.25 | 0.5819 | 0.4493 |

LOC STAGE STATOR ANGIN GAS ANGIN STATOR ANGIN GAS ANGIN ROTOR ANGIN GAS ANGIN

| | | | | | | | | | |
|---|---|-----|-----|------|------|------|------|-------|-------|
| 1 | 1 | 0.0 | 0.0 | 74.7 | 74.7 | 54.6 | 54.6 | -65.7 | -65.7 |
| 2 | 1 | 0.0 | 0.0 | 71.6 | 71.6 | 51.2 | 51.2 | -70.1 | -70.1 |
| 3 | 1 | 0.0 | 0.0 | 68.6 | 68.6 | 47.8 | 47.8 | -73.1 | -73.1 |

THE DATA EXCEEDED THE LIMITS OF THE PROGRAM AND S/C IS ASSUMED TO EQUAL .7

ECR STAGE NO BLADES TOTAL BLADE WTS

| | | | |
|---|---|----|------|
| 1 | 1 | 40 | 2.51 |
| 2 | 1 | 43 | 7.52 |
| 3 | 1 | 43 | 7.52 |

ROV STAGE CHORD1 CHORD2 CHORD3 HEIGHT 1-BEND FWD HARMONIC P/C H/C BENDING STRESS CENTRIFUGAL STRESS

| | | | | | | | | | | | |
|---|---|-------|-------|-------|-------|--------|------|-------|-------|--------|---------|
| 1 | 1 | 0.752 | 1.189 | 1.504 | 2.376 | 4076.6 | 14.0 | 0.728 | 1.097 | 5264.4 | |
| 2 | 1 | 1.574 | 1.205 | 0.787 | 2.849 | 3637.9 | 10.9 | 0.772 | 2.289 | 2436.5 | 37895.5 |

WTORIN WTORIN WTORIN WTORIN WTORIN WTORIN WTORIN WTORIN WTORIN WTORIN WTORIN WTORIN

| | | | | | | | | |
|------|------|------|-------|-------|-------|-------|--------|------|
| 4.65 | 3.34 | 5.55 | 13.54 | 21.46 | 0.667 | 0.429 | 28723. | 1.40 |
|------|------|------|-------|-------|-------|-------|--------|------|

DISC DIMENSIONS FOR ESAT TO RHUE (INCHES)

| | | | |
|--------|------|--------|-------|
| R (1) | 1.00 | 2 (1) | 1.329 |
| R (2) | 2.00 | 2 (2) | 1.228 |
| R (3) | 2.00 | 2 (3) | 0.959 |
| R (4) | 2.26 | 2 (4) | 0.446 |
| R (5) | 2.53 | 2 (5) | 0.433 |
| R (6) | 2.74 | 2 (6) | 0.448 |
| R (7) | 3.06 | 2 (7) | 0.403 |
| R (8) | 3.32 | 2 (8) | 0.397 |
| R (9) | 3.55 | 2 (9) | 0.370 |
| R (10) | 3.85 | 2 (10) | 0.363 |
| R (11) | 4.11 | 2 (11) | 0.326 |
| R (12) | 4.38 | 2 (12) | 0.315 |
| R (13) | 4.60 | 2 (13) | 0.310 |
| R (14) | 4.84 | 2 (14) | 1.423 |
| R (15) | 5.15 | 2 (15) | 1.023 |

STAGE 1 STATION 1 DISTANCE FROM STATION 1, STAGE 1

| | | |
|---|---|--------|
| 1 | 1 | 0.0000 |
| 2 | 1 | 1.7671 |
| 3 | 1 | 3.5342 |

LENGTH (IN) RADIAL INLET (IN) RADIAL EXIT (IN) RADIAL FWD (IN) WTOSHAF (LB) WTOT DTSCS+BLADES (LB)

| | | | | | | |
|------|------|------|------|------|------|-------|
| 3.53 | 5.15 | 7.28 | 5.15 | 8.23 | 3.21 | 28.37 |
|------|------|------|------|------|------|-------|

Output: from cycle and mechanical design of turbine

Output: From Performance Calculations

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED AIRSLEY MATTHEWSON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT FOR N/SORT (T01) = 0.40 P01/P02 = 1.11

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP (R) | TOTAL PRES (PSI) | STATIC PRES (PSI) | DENSITY (LB/FT**3) | ALPHA | BETA |
|---------|--------------|-------------------|-----------------|------------------|-------------------|--------------------|-------|------|
| 1 | 1 | 1767.8 | 1753.6 | 312.5 | 302.6 | 0.471 | 0.0 | 0.0 |
| 2 | 1 | 1767.8 | 1631.3 | 306.5 | 293.2 | 0.368 | 0.455 | 71.2 |
| 3 | 1 | 1553.8 | 1539.6 | 177.3 | 170.9 | 0.303 | 0.442 | 0.0 |
| STAGE | VX STATION 1 | 1767.8 | 1753.6 | 312.5 | 302.6 | 0.471 | 0.0 | 0.0 |
| STAGE | VX STATION 2 | 1767.8 | 1631.3 | 306.5 | 293.2 | 0.368 | 0.455 | 71.2 |
| STAGE | VX STATION 3 | 1553.8 | 1539.6 | 177.3 | 170.9 | 0.303 | 0.442 | 0.0 |
| 1 | 439. | 213. | 181. | | | | | |

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED AIRSLEY MATTHEWSON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT FOR N/SORT (T01) = 0.40 P01/P02 = 1.20

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP (R) | TOTAL PRES (PSI) | STATIC PRES (PSI) | DENSITY (LB/FT**3) | ALPHA | BETA |
|---------|--------------|-------------------|-----------------|------------------|-------------------|--------------------|-------|------|
| 1 | 1 | 1767.8 | 1753.6 | 312.5 | 302.6 | 0.471 | 0.0 | 0.0 |
| 2 | 1 | 1767.8 | 1631.3 | 306.5 | 293.2 | 0.368 | 0.430 | 71.2 |
| 3 | 1 | 1553.8 | 1539.6 | 177.3 | 170.9 | 0.303 | 0.411 | 0.0 |
| STAGE | VX STATION 1 | 1767.8 | 1753.6 | 312.5 | 302.6 | 0.471 | 0.0 | 0.0 |
| STAGE | VX STATION 2 | 1767.8 | 1631.3 | 306.5 | 293.2 | 0.368 | 0.430 | 71.2 |
| STAGE | VX STATION 3 | 1553.8 | 1539.6 | 177.3 | 170.9 | 0.303 | 0.411 | 0.0 |
| 1 | 439. | 302. | 282. | | | | | |

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED AIRSLEY MATTHEWSON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT FOR N/SORT (T01) = 0.40 P01/P02 = 1.27

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP (R) | TOTAL PRES (PSI) | STATIC PRES (PSI) | DENSITY (LB/FT**3) | ALPHA | BETA |
|---------|--------------|-------------------|-----------------|------------------|-------------------|--------------------|-------|------|
| 1 | 1 | 1767.8 | 1753.6 | 312.5 | 302.6 | 0.471 | 0.0 | 0.0 |
| 2 | 1 | 1767.8 | 1631.3 | 306.5 | 293.2 | 0.368 | 0.471 | 71.2 |
| 3 | 1 | 1553.8 | 1539.6 | 177.3 | 170.9 | 0.303 | 0.389 | 0.0 |
| STAGE | VX STATION 1 | 1767.8 | 1753.6 | 312.5 | 302.6 | 0.471 | 0.0 | 0.0 |
| STAGE | VX STATION 2 | 1767.8 | 1631.3 | 306.5 | 293.2 | 0.368 | 0.471 | 71.2 |
| STAGE | VX STATION 3 | 1553.8 | 1539.6 | 177.3 | 170.9 | 0.303 | 0.389 | 0.0 |
| 1 | 439. | 356. | 309. | | | | | |

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED AIRSLEY MATTHEWSON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT FOR N/SORT (T01) = 0.40 P01/P02 = 1.33

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP (R) | TOTAL PRES (PSI) | STATIC PRES (PSI) | DENSITY (LB/FT**3) | ALPHA | BETA |
|---------|--------------|-------------------|-----------------|------------------|-------------------|--------------------|-------|------|
| 1 | 1 | 1767.8 | 1753.6 | 312.5 | 302.6 | 0.471 | 0.0 | 0.0 |
| 2 | 1 | 1767.8 | 1631.3 | 306.5 | 293.2 | 0.368 | 0.398 | 71.2 |
| 3 | 1 | 1553.8 | 1539.6 | 177.3 | 170.9 | 0.303 | 0.372 | 0.0 |
| STAGE | VX STATION 1 | 1767.8 | 1753.6 | 312.5 | 302.6 | 0.471 | 0.0 | 0.0 |
| STAGE | VX STATION 2 | 1767.8 | 1631.3 | 306.5 | 293.2 | 0.368 | 0.398 | 71.2 |
| STAGE | VX STATION 3 | 1553.8 | 1539.6 | 177.3 | 170.9 | 0.303 | 0.372 | 0.0 |
| 1 | 439. | 383. | 341. | | | | | |

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED AIRSLEY MATTHEWSON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT FOR N/SORT (T01) = 0.40 P01/P02 = 1.37

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP (R) | TOTAL PRES (PSI) | STATIC PRES (PSI) | DENSITY (LB/FT**3) | ALPHA | BETA |
|---------|--------------|-------------------|-----------------|------------------|-------------------|--------------------|-------|------|
| 1 | 1 | 1767.8 | 1753.6 | 312.5 | 302.6 | 0.471 | 0.0 | 0.0 |
| 2 | 1 | 1767.8 | 1631.3 | 306.5 | 293.2 | 0.368 | 0.397 | 71.2 |
| 3 | 1 | 1553.8 | 1539.6 | 177.3 | 170.9 | 0.303 | 0.357 | 0.0 |
| STAGE | VX STATION 1 | 1767.8 | 1753.6 | 312.5 | 302.6 | 0.471 | 0.0 | 0.0 |
| STAGE | VX STATION 2 | 1767.8 | 1631.3 | 306.5 | 293.2 | 0.368 | 0.397 | 71.2 |
| STAGE | VX STATION 3 | 1553.8 | 1539.6 | 177.3 | 170.9 | 0.303 | 0.357 | 0.0 |
| 1 | 439. | 407. | 367. | | | | | |

1 439. 407. 367.

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE
CALCULATIONS OF IMPROVED AINSLEY MATHIPSON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT
FOR N/SORT(COI)= 0.40 PO1/PO2= 1.38

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP(R) | TOTAL PRES(PST) | STATIC PRES(PST) | DENSITY (LB/FT**3) | ALPHA | BETA |
|---------|--------------|-------------------|----------------|-----------------|------------------|--------------------|-------|-------|
| 1 | 1 | 1757.8 | 1753.6 | 312.5 | 302.6 | 0.471 | 0.0 | -68.0 |
| 2 | 1 | 1757.8 | 1753.6 | 306.5 | 229.8 | 0.368 | 0.381 | 71.2 |
| 3 | 1 | 1553.8 | 1619.0 | 177.3 | 210.9 | 0.303 | 0.355 | 0.0 |
| STAGE | VX STATION 1 | 1757.8 | 1753.6 | 312.5 | 302.6 | 0.471 | 0.0 | -68.0 |
| STAGE | VX STATION 2 | 1757.8 | 1753.6 | 306.5 | 229.8 | 0.368 | 0.381 | 71.2 |
| STAGE | VX STATION 3 | 1553.8 | 1619.0 | 177.3 | 210.9 | 0.303 | 0.355 | 0.0 |

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE
CALCULATIONS OF IMPROVED AINSLEY MATHIPSON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT
FOR N/SORT(COI)= 0.40 PO1/PO2= 1.39

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP(R) | TOTAL PRES(PST) | STATIC PRES(PST) | DENSITY (LB/FT**3) | ALPHA | BETA |
|---------|--------------|-------------------|----------------|-----------------|------------------|--------------------|-------|-------|
| 1 | 1 | 1757.8 | 1753.6 | 312.5 | 302.6 | 0.471 | 0.0 | -68.0 |
| 2 | 1 | 1757.8 | 1753.6 | 306.5 | 229.8 | 0.368 | 0.381 | 71.2 |
| 3 | 1 | 1553.8 | 1619.0 | 177.3 | 210.9 | 0.303 | 0.355 | 0.0 |
| STAGE | VX STATION 1 | 1757.8 | 1753.6 | 312.5 | 302.6 | 0.471 | 0.0 | -68.0 |
| STAGE | VX STATION 2 | 1757.8 | 1753.6 | 306.5 | 229.8 | 0.368 | 0.381 | 71.2 |
| STAGE | VX STATION 3 | 1553.8 | 1619.0 | 177.3 | 210.9 | 0.303 | 0.355 | 0.0 |

FOR N/SORT(COI) EQUAL TO 0.40

| PO1/PO2 | TOTAL EFFICIENCY | STATIC EFFICIENCY | N*SOPT(COI)/PO1 |
|---------|------------------|-------------------|-----------------|
| 1.11 | 0.4282 | 0.4796 | 0.600 |
| 1.20 | 0.3948 | 0.4282 | 0.600 |
| 1.27 | 0.3921 | 0.4282 | 0.600 |
| 1.31 | 0.3804 | 0.4282 | 0.600 |
| 1.37 | 0.3664 | 0.4282 | 0.600 |
| 1.34 | 0.3759 | 0.4282 | 0.600 |
| 1.39 | 0.3783 | 0.4282 | 0.600 |

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE
CALCULATIONS OF IMPROVED AINSLEY MATHIPSON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT
FOR N/SORT(COI)= 0.40 PO1/PO2= 1.11

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP(R) | TOTAL PRES(PST) | STATIC PRES(PST) | DENSITY (LB/FT**3) | ALPHA | BETA |
|---------|--------------|-------------------|----------------|-----------------|------------------|--------------------|-------|-------|
| 1 | 1 | 1757.8 | 1753.6 | 312.5 | 302.6 | 0.471 | 0.0 | -68.0 |
| 2 | 1 | 1757.8 | 1753.6 | 306.5 | 229.8 | 0.368 | 0.381 | 71.2 |
| 3 | 1 | 1553.8 | 1619.0 | 177.3 | 210.9 | 0.303 | 0.355 | 0.0 |
| STAGE | VX STATION 1 | 1757.8 | 1753.6 | 312.5 | 302.6 | 0.471 | 0.0 | -68.0 |
| STAGE | VX STATION 2 | 1757.8 | 1753.6 | 306.5 | 229.8 | 0.368 | 0.381 | 71.2 |
| STAGE | VX STATION 3 | 1553.8 | 1619.0 | 177.3 | 210.9 | 0.303 | 0.355 | 0.0 |

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE
CALCULATIONS OF IMPROVED AINSLEY MATHIPSON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT
FOR N/SORT(COI)= 0.60 PO1/PO2= 1.25

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP(R) | TOTAL PRES(PST) | STATIC PRES(PST) | DENSITY (LB/FT**3) | ALPHA | BETA |
|---------|--------------|-------------------|----------------|-----------------|------------------|--------------------|-------|-------|
| 1 | 1 | 1757.8 | 1753.6 | 312.5 | 302.6 | 0.471 | 0.0 | -68.0 |
| 2 | 1 | 1757.8 | 1753.6 | 306.5 | 229.8 | 0.368 | 0.381 | 71.2 |
| 3 | 1 | 1553.8 | 1619.0 | 177.3 | 210.9 | 0.303 | 0.355 | 0.0 |
| STAGE | VX STATION 1 | 1757.8 | 1753.6 | 312.5 | 302.6 | 0.471 | 0.0 | -68.0 |
| STAGE | VX STATION 2 | 1757.8 | 1753.6 | 306.5 | 229.8 | 0.368 | 0.381 | 71.2 |
| STAGE | VX STATION 3 | 1553.8 | 1619.0 | 177.3 | 210.9 | 0.303 | 0.355 | 0.0 |

1 439. 342. 265.

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE
CALCULATIONS OF IMPROVED AINSLEY HATHISON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT
FOR W/SOPT (C01) = 0.60 FOI/PO2 = 1.36

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP(°F) | TOTAL PRES(PSSI) | STATIC PRES(PSSI) | DENSITY (LB/FT**3) | ALPHA | BETA |
|---------|--------------|-------------------|-----------------|------------------|-------------------|--------------------|-------|-------------------|
| 1 | 1 | 1767.8 1767.8 | 1753.6 1753.6 | 312.5 312.5 | 302.6 302.6 | 0.471 0.471 | 0.0 | 0.0 -68.0 |
| 2 | 1 | 1767.8 1767.8 | 1631.3 1631.3 | 306.5 306.5 | 290.2 290.2 | 0.368 0.411 | 71.2 | 71.0 20.2 48.2 |
| 3 | 1 | 1553.8 1553.8 | 1539.6 1539.6 | 177.3 177.3 | 170.9 170.9 | 0.303 0.374 | 0.0 | -22.1 -69.4 -68.9 |
| STAGE | VX STATION 1 | VX STATION 2 | VX STATION 3 | | | | | |
| 1 | 439. | 356. | 320. | | | | | |

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE
CALCULATIONS OF IMPROVED AINSLEY HATHISON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT
FOR W/SOPT (C01) = 0.60 FOI/PO2 = 1.43

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP(°F) | TOTAL PRES(PSSI) | STATIC PRES(PSSI) | DENSITY (LB/FT**3) | ALPHA | BETA |
|---------|--------------|-------------------|-----------------|------------------|-------------------|--------------------|-------|-------------------|
| 1 | 1 | 1767.8 1767.8 | 1753.6 1753.6 | 312.5 312.5 | 302.6 302.6 | 0.471 0.471 | 0.0 | 0.0 -68.0 |
| 2 | 1 | 1767.8 1767.8 | 1631.3 1631.3 | 306.5 306.5 | 290.2 290.2 | 0.368 0.398 | 71.2 | 71.0 20.2 48.7 |
| 3 | 1 | 1553.8 1553.8 | 1539.6 1539.6 | 177.3 177.3 | 170.9 170.9 | 0.303 0.357 | 0.0 | -31.1 -69.4 -69.0 |
| STAGE | VX STATION 1 | VX STATION 2 | VX STATION 3 | | | | | |
| 1 | 439. | 353. | 350. | | | | | |

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE
CALCULATIONS OF IMPROVED AINSLEY HATHISON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT
FOR W/SOPT (C01) = 0.60 FOI/PO2 = 1.50

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP(°F) | TOTAL PRES(PSSI) | STATIC PRES(PSSI) | DENSITY (LB/FT**3) | ALPHA | BETA |
|---------|--------------|-------------------|-----------------|------------------|-------------------|--------------------|-------|-------------------|
| 1 | 1 | 1767.8 1767.8 | 1753.6 1753.6 | 312.5 312.5 | 302.6 302.6 | 0.471 0.471 | 0.0 | 0.0 -68.0 |
| 2 | 1 | 1767.8 1767.8 | 1631.3 1631.3 | 306.5 306.5 | 290.2 290.2 | 0.368 0.387 | 71.2 | 71.1 20.2 51.4 |
| 3 | 1 | 1553.8 1553.8 | 1539.6 1539.6 | 177.3 177.3 | 170.9 170.9 | 0.303 0.341 | 0.0 | -38.5 -69.4 -69.1 |
| STAGE | VX STATION 1 | VX STATION 2 | VX STATION 3 | | | | | |
| 1 | 439. | 407. | 395. | | | | | |

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE
CALCULATIONS OF IMPROVED AINSLEY HATHISON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT
FOR W/SOPT (C01) = 0.60 FOI/PO2 = 1.56

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP(°F) | TOTAL PRES(PSSI) | STATIC PRES(PSSI) | DENSITY (LB/FT**3) | ALPHA | BETA |
|---------|--------------|-------------------|-----------------|------------------|-------------------|--------------------|-------|-------------------|
| 1 | 1 | 1767.8 1767.8 | 1753.6 1753.6 | 312.5 312.5 | 302.6 302.6 | 0.471 0.471 | 0.0 | 0.0 -68.0 |
| 2 | 1 | 1767.8 1767.8 | 1631.3 1631.3 | 306.5 306.5 | 290.2 290.2 | 0.368 0.391 | 71.2 | 71.1 20.2 53.1 |
| 3 | 1 | 1553.8 1553.8 | 1539.6 1539.6 | 177.3 177.3 | 170.9 170.9 | 0.303 0.332 | 0.0 | -41.7 -69.4 -69.2 |
| STAGE | VX STATION 1 | VX STATION 2 | VX STATION 3 | | | | | |
| 1 | 439. | 407. | 403. | | | | | |

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE
CALCULATIONS OF IMPROVED AINSLEY HATHISON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT
FOR W/SOPT (C01) = 0.60 FOI/PO2 = 1.56

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP(°F) | TOTAL PRES(PSSI) | STATIC PRES(PSSI) | DENSITY (LB/FT**3) | ALPHA | BETA |
|---------|--------------|-------------------|-----------------|------------------|-------------------|--------------------|-------|-------------------|
| 1 | 1 | 1767.8 1767.8 | 1753.6 1753.6 | 312.5 312.5 | 302.6 302.6 | 0.471 0.471 | 0.0 | 0.0 -68.0 |
| 2 | 1 | 1767.8 1767.8 | 1631.3 1631.3 | 306.5 306.5 | 290.2 290.2 | 0.368 0.376 | 71.2 | 71.2 20.2 54.0 |
| 3 | 1 | 1553.8 1553.8 | 1539.6 1539.6 | 177.3 177.3 | 170.9 170.9 | 0.303 0.336 | 0.0 | -43.2 -69.4 -69.2 |
| STAGE | VX STATION 1 | VX STATION 2 | VX STATION 3 | | | | | |
| 1 | 439 | 436 | 414 | | | | | |

414.

436.

439.

FOR W/SORT(TOI) TOTAL TO 0.60

PO1/PO2 TOTAL EFFICIENCY STATIC EFFICIENCY *SORT(TOI)/PO1

| | | | |
|------|--------|--------|-------|
| 1.11 | 0.9281 | 0.3130 | 0.600 |
| 1.25 | 0.9236 | 0.809 | 0.809 |
| 1.36 | 0.8984 | 0.000 | 0.000 |
| 1.43 | 0.7052 | 0.360 | 0.360 |
| 1.50 | 0.6426 | 0.988 | 0.988 |
| 1.54 | 0.6052 | 0.9093 | 1.000 |
| 1.56 | 0.6076 | 0.8058 | 1.010 |

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED AINSLEY MATHIESON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT
FOR W/SORT(TOI) = 0.80 PO1/PO2 = 1.06

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP(°) | TOTAL PRES(PST) | STATIC PRES(PST) | DENSITY(LR/FT**3) | ALPHA | BETA |
|---------|--------------|-------------------|----------------|-----------------|------------------|-------------------|-------|------|
| 1 | 1 | 1767.8 | 1753.6 | 312.5 | 302.6 | 0.471 0.471 | 0.0 | 0.0 |
| 2 | 1 | 1767.8 | 1753.6 | 306.5 | 299.3 | 0.368 0.455 | 71.2 | 70.9 |
| 3 | 1 | 1553.8 | 1745.1 | 177.3 | 294.6 | 0.303 0.446 | 0.0 | 68.9 |
| STAGE | VX STATION 1 | VX STATION 2 | VX STATION 3 | | | | | |
| 1 | 439. | 213. | 180. | | | | | |

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED AINSLEY MATHIESON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT
FOR W/SORT(TOI) = 0.80 PO1/PO2 = 1.24

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP(°) | TOTAL PRES(PST) | STATIC PRES(PST) | DENSITY(LR/FT**3) | ALPHA | BETA |
|---------|--------------|-------------------|----------------|-----------------|------------------|-------------------|-------|------|
| 1 | 1 | 1767.8 | 1753.6 | 312.5 | 302.6 | 0.471 0.471 | 0.0 | 0.0 |
| 2 | 1 | 1767.8 | 1753.6 | 306.5 | 299.3 | 0.368 0.430 | 71.2 | 70.9 |
| 3 | 1 | 1553.8 | 1692.0 | 177.3 | 252.2 | 0.303 0.402 | 0.0 | 42.9 |
| STAGE | VX STATION 1 | VX STATION 2 | VX STATION 3 | | | | | |
| 1 | 439. | 302. | 265. | | | | | |

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED AINSLEY MATHIESON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT
FOR W/SORT(TOI) = 0.80 PO1/PO2 = 1.39

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP(°) | TOTAL PRES(PST) | STATIC PRES(PST) | DENSITY(LR/FT**3) | ALPHA | BETA |
|---------|--------------|-------------------|----------------|-----------------|------------------|-------------------|-------|------|
| 1 | 1 | 1767.8 | 1753.6 | 312.5 | 302.6 | 0.471 0.471 | 0.0 | 0.0 |
| 2 | 1 | 1767.8 | 1753.6 | 306.5 | 299.3 | 0.368 0.411 | 71.2 | 71.0 |
| 3 | 1 | 1553.8 | 1640.1 | 177.3 | 225.1 | 0.303 0.369 | 0.0 | 16.3 |
| STAGE | VX STATION 1 | VX STATION 2 | VX STATION 3 | | | | | |
| 1 | 439. | 340. | 323. | | | | | |

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED AINSLEY MATHIESON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT
FOR W/SORT(TOI) = 0.80 PO1/PO2 = 1.44

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP(°) | TOTAL PRES(PST) | STATIC PRES(PST) | DENSITY(LR/FT**3) | ALPHA | BETA |
|---------|--------------|-------------------|----------------|-----------------|------------------|-------------------|-------|------|
| 1 | 1 | 1767.8 | 1753.6 | 312.5 | 302.6 | 0.471 0.471 | 0.0 | 0.0 |
| 2 | 1 | 1767.8 | 1753.6 | 306.5 | 299.3 | 0.368 0.398 | 71.2 | 71.0 |
| 3 | 1 | 1553.8 | 1611.3 | 177.3 | 204.7 | 0.303 0.349 | 0.0 | -2.6 |
| STAGE | VX STATION 1 | VX STATION 2 | VX STATION 3 | | | | | |
| 1 | 439 | 383 | 365 | | | | | |

1 439. 383. 365.

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED AUSELEY MATHESON METHOD 1ST COLUMN IS CYCLP CALCULATIONS AT DESIGN POINT
FOR N/SORT(TO1)= 0.80 PO1/PO2= 1.60

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP(P) | TOTAL PRES(PSSI) | STATIC PRES(PSSI) | DENSITY (LB/FT**3) | ALPHA | BETA |
|---------|--------------|-------------------|----------------|------------------|-------------------|--------------------|-----------|-------------|
| 1 | 1 | 1757.8 1767.8 | 1753.6 1753.6 | 312.5 312.5 | 302.6 302.6 | 0.471 0.471 | 0.0 0.0 | -68.0 0.0 |
| 2 | 1 | 1767.8 1767.8 | 1631.3 1631.3 | 306.5 306.5 | 226.2 233.7 | 0.368 0.387 | 71.2 71.1 | 20.2 35.0 |
| 3 | 1 | 1553.8 1556.8 | 1539.6 1574.1 | 177.3 195.6 | 170.9 189.4 | 0.303 0.328 | 0.0 -16.2 | -69.4 -59.2 |
| STAGE | VX STATION 1 | VX STATION 2 | VX STATION 3 | | | | | |
| 1 | 439. | 467. | 460. | | | | | |

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED AUSELEY MATHESON METHOD 1ST COLUMN IS CYCLP CALCULATIONS AT DESIGN POINT
FOR N/SORT(TO1)= 0.80 PO1/PO2= 1.64

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP(P) | TOTAL PRES(PSSI) | STATIC PRES(PSSI) | DENSITY (LB/FT**3) | ALPHA | BETA |
|---------|--------------|-------------------|----------------|------------------|-------------------|--------------------|-----------|-------------|
| 1 | 1 | 1767.8 1767.8 | 1753.6 1753.6 | 312.5 312.5 | 302.6 302.6 | 0.471 0.471 | 0.0 0.0 | -68.0 0.0 |
| 2 | 1 | 1767.8 1767.8 | 1631.3 1631.3 | 306.5 306.5 | 226.2 228.9 | 0.368 0.381 | 71.2 71.1 | 20.2 19.1 |
| 3 | 1 | 1553.8 1575.2 | 1539.6 1566.2 | 177.3 190.0 | 170.9 182.9 | 0.303 0.320 | 0.0 -22.0 | -69.4 -69.2 |
| STAGE | VX STATION 1 | VX STATION 2 | VX STATION 3 | | | | | |
| 1 | 439. | 427. | 416. | | | | | |

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED AUSELEY MATHESON METHOD 1ST COLUMN IS CYCLP CALCULATIONS AT DESIGN POINT
FOR N/SORT(TO1)= 0.80 PO1/PO2= 1.69

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP(P) | TOTAL PRES(PSSI) | STATIC PRES(PSSI) | DENSITY (LB/FT**3) | ALPHA | BETA |
|---------|--------------|-------------------|----------------|------------------|-------------------|--------------------|-----------|-------------|
| 1 | 1 | 1767.8 1767.8 | 1753.6 1753.6 | 312.5 312.5 | 302.6 302.5 | 0.471 0.471 | 0.0 0.0 | -68.0 0.0 |
| 2 | 1 | 1767.8 1767.8 | 1631.3 1631.7 | 306.5 306.5 | 226.2 225.5 | 0.368 0.376 | 71.2 71.2 | 20.2 42.7 |
| 3 | 1 | 1553.8 1565.1 | 1539.6 1567.8 | 177.3 184.6 | 170.9 176.6 | 0.303 0.311 | 0.0 -26.3 | -69.4 -69.3 |
| STAGE | VX STATION 1 | VX STATION 2 | VX STATION 3 | | | | | |
| 1 | 439. | 436. | 430. | | | | | |

FOR N/SORT(TO1) EQUAL TO 0.80

| PO1/PO2 | TOTAL EFFICIENCY | STATIC EFFICIENCY | N/SORT(TO1)/PO1 |
|---------|------------------|-------------------|-----------------|
| 1.64 | 0.9762 | 0.9117 | 0.600 |
| 1.24 | 0.9305 | 0.8431 | 0.800 |
| 1.39 | 0.9178 | 0.8662 | 0.900 |
| 1.40 | 0.9342 | 0.8429 | 0.950 |
| 1.60 | 0.9269 | 0.8705 | 0.980 |
| 1.64 | 0.9327 | 0.8700 | 1.000 |
| 1.69 | 0.9304 | 0.9625 | 1.010 |

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED AUSELEY MATHESON METHOD 1ST COLUMN IS CYCLP CALCULATIONS AT DESIGN POINT
FOR N/SORT(TO1)= 1.30 PO1/PO2= 0.89

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP(P) | TOTAL PRES(PSSI) | STATIC PRES(PSSI) | DENSITY (LB/FT**3) | ALPHA | BETA |
|---------|--------------|-------------------|----------------|------------------|-------------------|--------------------|-----------|-------------|
| 1 | 1 | 1767.8 1767.8 | 1753.6 1753.6 | 312.5 312.5 | 302.6 302.5 | 0.471 0.471 | 0.0 0.0 | -68.0 0.0 |
| 2 | 1 | 1767.8 1767.8 | 1631.3 1716.4 | 306.5 313.3 | 226.2 289.3 | 0.368 0.455 | 71.2 70.9 | 20.2 -67.4 |
| 3 | 1 | 1553.8 1790.5 | 1539.6 1740.9 | 177.3 319.1 | 170.9 291.6 | 0.303 0.457 | 0.0 76.2 | -69.4 -68.9 |
| STAGE | VX STATION 1 | VX STATION 2 | VX STATION 3 | | | | | |
| 1 | 439 | 213 | 175 | | | | | |

175. 213. 175.

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED AINSLEY METHIESON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT FOR $N/SORT^{(TOT)} = 1.00$ $P01/P02 = 1.19$

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP(°) | TOTAL PRES(PST) | STATIC PRES(PST) | DENSITY (LB/FT**3) | ALPHA | BETA |
|---------|--------------|-------------------|----------------|-----------------|------------------|--------------------|-------|------|
| 1 | 1 | 1767.9 | 1767.8 | 312.5 | 302.6 | 0.471 | 0.471 | 0.0 |
| 2 | 1 | 1767.8 | 1767.8 | 306.5 | 220.2 | 0.368 | 0.430 | 71.2 |
| 3 | 1 | 1553.9 | 1590.1 | 177.3 | 170.9 | 0.303 | 0.406 | 0.0 |
| STAGE | VX STATION 1 | VX STATION 2 | VX STATION 3 | | | | | |
| 1 | 439. | 302. | 265. | | | | | |

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED AINSLEY METHIESON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT FOR $N/SORT^{(TOT)} = 1.00$ $P01/P02 = 1.36$

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP(°) | TOTAL PRES(PST) | STATIC PRES(PST) | DENSITY (LB/FT**3) | ALPHA | BETA |
|---------|--------------|-------------------|----------------|-----------------|------------------|--------------------|-------|------|
| 1 | 1 | 1767.8 | 1767.8 | 312.5 | 302.6 | 0.471 | 0.471 | 0.0 |
| 2 | 1 | 1767.8 | 1767.8 | 306.5 | 220.2 | 0.368 | 0.411 | 71.2 |
| 3 | 1 | 1553.9 | 1590.1 | 177.3 | 170.9 | 0.303 | 0.369 | 0.0 |
| STAGE | VX STATION 1 | VX STATION 2 | VX STATION 3 | | | | | |
| 1 | 439. | 302. | 265. | | | | | |

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED AINSLEY METHIESON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT FOR $N/SORT^{(TOT)} = 1.00$ $P01/P02 = 1.49$

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP(°) | TOTAL PRES(PST) | STATIC PRES(PST) | DENSITY (LB/FT**3) | ALPHA | BETA |
|---------|--------------|-------------------|----------------|-----------------|------------------|--------------------|-------|------|
| 1 | 1 | 1767.8 | 1767.8 | 312.5 | 302.6 | 0.471 | 0.471 | 0.0 |
| 2 | 1 | 1767.8 | 1767.8 | 306.5 | 220.2 | 0.368 | 0.398 | 71.2 |
| 3 | 1 | 1553.9 | 1590.1 | 177.3 | 170.9 | 0.303 | 0.347 | 0.0 |
| STAGE | VX STATION 1 | VX STATION 2 | VX STATION 3 | | | | | |
| 1 | 439. | 302. | 265. | | | | | |

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED AINSLEY METHIESON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT FOR $N/SORT^{(TOT)} = 1.00$ $P01/P02 = 1.61$

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP(°) | TOTAL PRES(PST) | STATIC PRES(PST) | DENSITY (LB/FT**3) | ALPHA | BETA |
|---------|--------------|-------------------|----------------|-----------------|------------------|--------------------|-------|------|
| 1 | 1 | 1767.8 | 1767.8 | 312.5 | 302.6 | 0.471 | 0.471 | 0.0 |
| 2 | 1 | 1767.8 | 1767.8 | 306.5 | 220.2 | 0.368 | 0.387 | 71.2 |
| 3 | 1 | 1553.9 | 1590.1 | 177.3 | 170.9 | 0.303 | 0.326 | 0.0 |
| STAGE | VX STATION 1 | VX STATION 2 | VX STATION 3 | | | | | |
| 1 | 439. | 302. | 265. | | | | | |

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED AINSLEY METHIESON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT FOR $N/SORT^{(TOT)} = 1.00$ $P01/P02 = 1.71$

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP(°) | TOTAL PRES(PST) | STATIC PRES(PST) | DENSITY (LB/FT**3) | ALPHA | BETA |
|---------|--------------|-------------------|----------------|-----------------|------------------|--------------------|-------|------|
| 1 | 1 | 1767.8 | 1767.8 | 312.5 | 302.6 | 0.471 | 0.471 | 0.0 |
| 2 | 1 | 1767.8 | 1767.8 | 306.5 | 220.2 | 0.368 | 0.381 | 71.2 |
| 3 | 1 | 1553.9 | 1590.1 | 177.3 | 170.9 | 0.303 | 0.311 | 0.0 |
| STAGE | VX STATION 1 | VX STATION 2 | VX STATION 3 | | | | | |
| 1 | 439. | 302. | 265. | | | | | |

from Cycle Calculation $Vx_1 = Vx_2 = Vx_3 = 493$

428

427

439

1 439. 477. 428.

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED AINSLEY MATHIESON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT
FOR M/SORT(TOI) = 1.00 P01/P02 = 1.77

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP(B) | TOTAL PRES(PST) | STATIC PRES(PST) | DENSITY (LB/FT**3) | ALPHA | BETA |
|---------|--------------|-------------------|----------------|-----------------|------------------|--------------------|-------|------|
| 1 | 1 | 1767.8 | 1753.6 | 312.5 | 302.6 | 0.471 | 0.0 | 0.0 |
| 2 | 1 | 1767.8 | 1631.3 | 306.5 | 220.2 | 0.368 | 0.376 | 71.2 |
| 3 | 1 | 1553.8 | 1539.6 | 177.3 | 170.9 | 0.303 | 0.383 | 0.0 |
| STAGE | VX STATION 1 | 1767.8 | 1539.6 | 177.3 | 170.9 | 0.303 | 0.383 | 0.0 |
| STAGE | VX STATION 2 | 1767.8 | 1539.6 | 177.3 | 170.9 | 0.303 | 0.383 | 0.0 |
| STAGE | VX STATION 3 | 1767.8 | 1539.6 | 177.3 | 170.9 | 0.303 | 0.383 | 0.0 |
| STAGE | VX STATION 4 | 1767.8 | 1539.6 | 177.3 | 170.9 | 0.303 | 0.383 | 0.0 |

FOR M/SORT(TOI) EQUAL TO 1.00

| P01/P02 | TOTAL EFFICIENCY | STATIC EFFICIENCY | M*SQRT(TOI)/P01 |
|---------|------------------|-------------------|-----------------|
| 0.99 | 1.3723 | -0.8225 | 0.600 |
| 1.19 | 0.9222 | 0.7128 | 0.800 |
| 1.36 | 0.9126 | 0.8223 | 0.900 |
| 1.40 | 0.9200 | 0.9709 | 0.950 |
| 1.61 | 0.9207 | 0.980 | 0.980 |
| 1.71 | 0.9325 | 0.9792 | 1.000 |
| 1.77 | 0.9412 | 0.8862 | 1.010 |

At-Design Point

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED AINSLEY MATHIESON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT
FOR M/SORT(TOI) = 1.10 P01/P02 = 0.91

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP(B) | TOTAL PRES(PST) | STATIC PRES(PST) | DENSITY (LB/FT**3) | ALPHA | BETA |
|---------|--------------|-------------------|----------------|-----------------|------------------|--------------------|-------|------|
| 1 | 1 | 1767.8 | 1753.6 | 312.5 | 302.6 | 0.471 | 0.0 | 0.0 |
| 2 | 1 | 1767.8 | 1631.3 | 306.5 | 220.2 | 0.368 | 0.455 | 71.2 |
| 3 | 1 | 1553.8 | 1539.6 | 177.3 | 170.9 | 0.303 | 0.465 | 0.0 |
| STAGE | VX STATION 1 | 1767.8 | 1539.6 | 177.3 | 170.9 | 0.303 | 0.465 | 0.0 |
| STAGE | VX STATION 2 | 1767.8 | 1539.6 | 177.3 | 170.9 | 0.303 | 0.465 | 0.0 |
| STAGE | VX STATION 3 | 1767.8 | 1539.6 | 177.3 | 170.9 | 0.303 | 0.465 | 0.0 |
| STAGE | VX STATION 4 | 1767.8 | 1539.6 | 177.3 | 170.9 | 0.303 | 0.465 | 0.0 |

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED AINSLEY MATHIESON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT
FOR M/SORT(TOI) = 1.10 P01/P02 = 1.14

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP(B) | TOTAL PRES(PST) | STATIC PRES(PST) | DENSITY (LB/FT**3) | ALPHA | BETA |
|---------|--------------|-------------------|----------------|-----------------|------------------|--------------------|-------|------|
| 1 | 1 | 1767.8 | 1753.6 | 312.5 | 302.6 | 0.471 | 0.0 | 0.0 |
| 2 | 1 | 1767.8 | 1631.3 | 306.5 | 220.2 | 0.368 | 0.430 | 71.2 |
| 3 | 1 | 1553.8 | 1539.6 | 177.3 | 170.9 | 0.303 | 0.410 | 0.0 |
| STAGE | VX STATION 1 | 1767.8 | 1539.6 | 177.3 | 170.9 | 0.303 | 0.410 | 0.0 |
| STAGE | VX STATION 2 | 1767.8 | 1539.6 | 177.3 | 170.9 | 0.303 | 0.410 | 0.0 |
| STAGE | VX STATION 3 | 1767.8 | 1539.6 | 177.3 | 170.9 | 0.303 | 0.410 | 0.0 |
| STAGE | VX STATION 4 | 1767.8 | 1539.6 | 177.3 | 170.9 | 0.303 | 0.410 | 0.0 |

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE CALCULATIONS OF IMPROVED AINSLEY MATHIESON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT
FOR M/SORT(TOI) = 1.10 P01/P02 = 1.33

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP(B) | TOTAL PRES(PST) | STATIC PRES(PST) | DENSITY (LB/FT**3) | ALPHA | BETA |
|---------|--------------|-------------------|----------------|-----------------|------------------|--------------------|-------|------|
| 1 | 1 | 1767.8 | 1753.6 | 312.5 | 302.6 | 0.471 | 0.0 | 0.0 |
| 2 | 1 | 1767.8 | 1631.3 | 306.5 | 220.2 | 0.368 | 0.411 | 71.2 |
| 3 | 1 | 1553.8 | 1539.6 | 177.3 | 170.9 | 0.303 | 0.373 | 0.0 |
| STAGE | VX STATION 1 | 1767.8 | 1539.6 | 177.3 | 170.9 | 0.303 | 0.373 | 0.0 |
| STAGE | VX STATION 2 | 1767.8 | 1539.6 | 177.3 | 170.9 | 0.303 | 0.373 | 0.0 |
| STAGE | VX STATION 3 | 1767.8 | 1539.6 | 177.3 | 170.9 | 0.303 | 0.373 | 0.0 |
| STAGE | VX STATION 4 | 1767.8 | 1539.6 | 177.3 | 170.9 | 0.303 | 0.373 | 0.0 |

439

356

322

356.

436.

1

322.

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE
 CALCULATIONS OF IMPROVED AINSLEY MATHESSON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT
 FOR N/SORT(TO1)= 1.10 P01/P02= 1.44

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP(R) | TOTAL PRPS(PST) | STATIC PRPS(PST) | DENSITY(LR/P2**3) | ALPHA | BETA |
|--------------------|--------------|-------------------|----------------|-----------------|------------------|-------------------|-------|------|
| 1 | 1 | 1767.8 | 1767.8 | 312.5 | 302.6 | 0.471 | 0.0 | 0.0 |
| 2 | 1 | 1767.8 | 1767.8 | 312.5 | 302.6 | 0.471 | 0.0 | 0.0 |
| 3 | 1 | 1553.8 | 1553.8 | 306.5 | 225.5 | 0.368 | 0.398 | 71.2 |
| STAGE VX STATION 1 | VX STATION 2 | 177.3 | 213.8 | 177.3 | 213.8 | 0.303 | 0.348 | 0.0 |
| 439. | 439. | | | | | | | |

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE
 CALCULATIONS OF IMPROVED AINSLEY MATHESSON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT
 FOR N/SORT(TO1)= 1.10 P01/P02= 1.60

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP(R) | TOTAL PRPS(PST) | STATIC PRPS(PST) | DENSITY(LR/P2**3) | ALPHA | BETA |
|--------------------|--------------|-------------------|----------------|-----------------|------------------|-------------------|-------|------|
| 1 | 1 | 1767.8 | 1767.8 | 312.5 | 302.6 | 0.471 | 0.0 | 0.0 |
| 2 | 1 | 1767.8 | 1767.8 | 312.5 | 302.6 | 0.471 | 0.0 | 0.0 |
| 3 | 1 | 1553.8 | 1553.8 | 306.5 | 225.5 | 0.368 | 0.387 | 71.2 |
| STAGE VX STATION 1 | VX STATION 2 | 177.3 | 213.8 | 177.3 | 213.8 | 0.303 | 0.327 | 0.0 |
| 439. | 439. | | | | | | | |

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE
 CALCULATIONS OF IMPROVED AINSLEY MATHESSON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT
 FOR N/SORT(TO1)= 1.10 P01/P02= 1.70

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP(R) | TOTAL PRPS(PST) | STATIC PRPS(PST) | DENSITY(LR/P2**3) | ALPHA | BETA |
|--------------------|--------------|-------------------|----------------|-----------------|------------------|-------------------|-------|------|
| 1 | 1 | 1767.8 | 1767.8 | 312.5 | 302.6 | 0.471 | 0.0 | 0.0 |
| 2 | 1 | 1767.8 | 1767.8 | 312.5 | 302.6 | 0.471 | 0.0 | 0.0 |
| 3 | 1 | 1553.8 | 1553.8 | 306.5 | 225.5 | 0.368 | 0.381 | 71.2 |
| STAGE VX STATION 1 | VX STATION 2 | 177.3 | 213.8 | 177.3 | 213.8 | 0.303 | 0.312 | 0.0 |
| 439. | 439. | | | | | | | |

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE
 CALCULATIONS OF IMPROVED AINSLEY MATHESSON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT
 FOR N/SORT(TO1)= 1.10 P01/P02= 1.79

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP(R) | TOTAL PRPS(PST) | STATIC PRPS(PST) | DENSITY(LR/P2**3) | ALPHA | BETA |
|--------------------|--------------|-------------------|----------------|-----------------|------------------|-------------------|-------|------|
| 1 | 1 | 1767.8 | 1767.8 | 312.5 | 302.6 | 0.471 | 0.0 | 0.0 |
| 2 | 1 | 1767.8 | 1767.8 | 312.5 | 302.6 | 0.471 | 0.0 | 0.0 |
| 3 | 1 | 1553.8 | 1553.8 | 306.5 | 225.5 | 0.368 | 0.376 | 71.2 |
| STAGE VX STATION 1 | VX STATION 2 | 177.3 | 213.8 | 177.3 | 213.8 | 0.303 | 0.301 | 0.0 |
| 439. | 439. | | | | | | | |

FOR N/SORT(TO1) EQUAL TO 1.10

P01/P02 TOTAL EFFICIENCY STATIC EFFICIENCY N-SORT(TO1)/P01

| | | | |
|------|--------|---------|-------|
| 0.93 | 1.1022 | -1.8246 | 0.600 |
| 1.14 | 0.8002 | 0.6823 | 0.800 |
| 1.37 | 0.9022 | 0.7779 | 0.900 |
| 1.46 | 0.9281 | 0.8044 | 0.950 |
| 1.60 | 0.9466 | 0.8766 | 0.990 |
| 1.70 | 0.9422 | 0.8416 | 1.000 |
| 1.78 | 0.9419 | 0.8546 | 1.010 |

1 439. 356. 322.

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPLETED WITH PERFORMANCE CALCULATIONS OF IMPROVED KINLEY HATHIESON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT
POB M/SORT(M01) = 1.10 PO1/PO2 = 1.46

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP(°F) | TOTAL PRES(PST) | STATIC PRES(PST) | DENSITY (LB/PT**3) | ALPHA | BETA |
|---------|--------------|-------------------|-----------------|-----------------|------------------|--------------------|-------|------|
| 1 | 1 | 1767.8 | 1753.6 | 312.5 | 302.6 | 0.471 | 0.0 | 0.0 |
| 2 | 1 | 1767.8 | 1631.3 | 306.5 | 220.2 | 0.368 | 0.398 | 71.2 |
| 3 | 1 | 1553.8 | 1530.6 | 177.3 | 170.9 | 0.303 | 0.348 | 0.0 |
| STAGE | VX STATION 1 | 1767.8 | 1631.3 | 306.5 | 220.2 | 0.368 | 0.398 | 71.2 |
| STAGE | VX STATION 2 | 1553.8 | 1530.6 | 177.3 | 170.9 | 0.303 | 0.348 | 0.0 |
| STAGE | VX STATION 3 | 1553.8 | 1530.6 | 177.3 | 170.9 | 0.303 | 0.348 | 0.0 |

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPLETED WITH PERFORMANCE CALCULATIONS OF IMPROVED KINLEY HATHIESON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT
POB M/SORT(M01) = 1.10 PO1/PO2 = 1.60

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP(°F) | TOTAL PRES(PST) | STATIC PRES(PST) | DENSITY (LB/PT**3) | ALPHA | BETA |
|---------|--------------|-------------------|-----------------|-----------------|------------------|--------------------|-------|------|
| 1 | 1 | 1767.8 | 1753.6 | 312.5 | 302.6 | 0.471 | 0.0 | 0.0 |
| 2 | 1 | 1767.8 | 1631.3 | 306.5 | 220.2 | 0.368 | 0.387 | 71.2 |
| 3 | 1 | 1553.8 | 1530.6 | 177.3 | 170.9 | 0.303 | 0.327 | 0.0 |
| STAGE | VX STATION 1 | 1767.8 | 1631.3 | 306.5 | 220.2 | 0.368 | 0.387 | 71.2 |
| STAGE | VX STATION 2 | 1553.8 | 1530.6 | 177.3 | 170.9 | 0.303 | 0.327 | 0.0 |
| STAGE | VX STATION 3 | 1553.8 | 1530.6 | 177.3 | 170.9 | 0.303 | 0.327 | 0.0 |

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPLETED WITH PERFORMANCE CALCULATIONS OF IMPROVED KINLEY HATHIESON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT
POB M/SORT(M01) = 1.10 PO1/PO2 = 1.70

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP(°F) | TOTAL PRES(PST) | STATIC PRES(PST) | DENSITY (LB/PT**3) | ALPHA | BETA |
|---------|--------------|-------------------|-----------------|-----------------|------------------|--------------------|-------|------|
| 1 | 1 | 1767.8 | 1753.6 | 312.5 | 302.6 | 0.471 | 0.0 | 0.0 |
| 2 | 1 | 1767.8 | 1631.3 | 306.5 | 220.2 | 0.368 | 0.381 | 71.2 |
| 3 | 1 | 1553.8 | 1530.6 | 177.3 | 170.9 | 0.303 | 0.312 | 0.0 |
| STAGE | VX STATION 1 | 1767.8 | 1631.3 | 306.5 | 220.2 | 0.368 | 0.381 | 71.2 |
| STAGE | VX STATION 2 | 1553.8 | 1530.6 | 177.3 | 170.9 | 0.303 | 0.312 | 0.0 |
| STAGE | VX STATION 3 | 1553.8 | 1530.6 | 177.3 | 170.9 | 0.303 | 0.312 | 0.0 |

FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF CYCLE CALCULATIONS WILL BE COMPLETED WITH PERFORMANCE CALCULATIONS OF IMPROVED KINLEY HATHIESON METHOD 1ST COLUMN IS CYCLE CALCULATIONS AT DESIGN POINT
POB M/SORT(M01) = 1.10 PO1/PO2 = 1.79

| STATION | STAGE | TOTAL TEMPERATURE | STATIC TEMP(°F) | TOTAL PRES(PST) | STATIC PRES(PST) | DENSITY (LB/PT**3) | ALPHA | BETA |
|---------|--------------|-------------------|-----------------|-----------------|------------------|--------------------|-------|------|
| 1 | 1 | 1767.8 | 1753.6 | 312.5 | 302.6 | 0.471 | 0.0 | 0.0 |
| 2 | 1 | 1767.8 | 1631.3 | 306.5 | 220.2 | 0.368 | 0.376 | 71.2 |
| 3 | 1 | 1553.8 | 1530.6 | 177.3 | 170.9 | 0.303 | 0.301 | 0.0 |
| STAGE | VX STATION 1 | 1767.8 | 1631.3 | 306.5 | 220.2 | 0.368 | 0.376 | 71.2 |
| STAGE | VX STATION 2 | 1553.8 | 1530.6 | 177.3 | 170.9 | 0.303 | 0.301 | 0.0 |
| STAGE | VX STATION 3 | 1553.8 | 1530.6 | 177.3 | 170.9 | 0.303 | 0.301 | 0.0 |

FOR M/SORT(M01) EQUAL TO 1.10

| PO1/PO2 | TOTAL EFFICIENCY | STATIC EFFICIENCY | M-SORT(M01)/PO1 |
|---------|------------------|-------------------|-----------------|
| 0.93 | 1.1022 | -1.8246 | 0.600 |
| 1.14 | 0.8062 | 0.5833 | 0.800 |
| 1.33 | 0.9222 | 0.7574 | 0.900 |
| 1.46 | 0.9381 | 0.7664 | 0.950 |
| 1.60 | 0.9466 | 0.7664 | 0.980 |
| 1.70 | 0.9422 | 0.7664 | 1.000 |
| 1.78 | 0.9419 | 0.7664 | 1.010 |

APPENDIX III

Computer Programs developed for the Design
of a Free Vortex Axial Flow Turbine.

Computer Program 1 Complete design

Computer Program 2 Performance Calculations Only


```
// ENDO, REGION=220K, CLASS=C
//MUTID USWR=(M13562, P14715, VME)
```

```
//SRI LOW
```

```
//MAIN TIME=6, LINES=2, CARDS=1
```

```
// EXEC WAITIV
```

```
//C.SYSIN DE *, DCB=BLKSIZE=2000
```

```
$JOB
```

```
ENDC, NOLIST, NOSURCHK, TIME=60
```

```
COMMON PI, GO, PJ, HJ, GAMMA, VX, SOMEGA, DHOS, CP, GAMMA2, GAMMA1, KK
```

```
COMMON/AFEA1/TET, TC, TIPCIA
```

```
DIMENSION SPRES(3,3,12), RADJUS(3,3,12), ALPHA(3,3,12), BETA(3,3,12)
```

```
DIMENSION DEN(3), PAD(3), TTEMP(3,12), TERPS(3,12), STEMP(3,3,12)
```

```
DIMENSION PEACT(3,3,12), RHO(3,3,12), REALMS(3,2,12), REIMP(3,2,12),
```

```
TEMPR(3,2,12), VAXIAL(3,12)
```

```
DIMENSION RBLADT(3,2,12), SBLADE(3,2,12), CHOPDR(3,2,12), SIGMAB(2,12
```

```
1), SIGMAC(12), BLADEH(2,12), PTOCHO(2,12), ASPEC(2,12), TFM(50), V(50)
```

```
DIMENSION BIADWT(2,12), WTDISC(12), P(15), Z(15), NOBLAD(2,12)
```

```
DIMENSION SPEED(5), WDOTP(7), OMEGAB(5), POIS(7), PHOC(3,10), BPTAC(3,1
```

```
10), ALPHAC(3,10), TOTC(3,10), TSC(3,10), POC(3,10), PSC(3,10), PPOC(3,10
```

```
3), VEL(3,10), FVEL(3,10), PMAC(100), WTAP(100), DSPT(100), VEITOT(100),
```

```
101PO2(7), ENTMJ(7), ENSMJ(7), PTOC(3,10), A(6), P(6)
```

```
DIMENSION WTAP11(70), WTAP12(70), RMAC11(70), RMAC12(70), VPOT11(70), V
```

```
1EOT12(70), DSPT11(70), DSPT12(70)
```

```
CHARACTER WORD*80
```

```
DATA SPEED/.4,.6,.8,1.,1.1/
```

```
DATA WDOTP/.6,.8,.9,.95,.98,1.,1.01/
```

```
DATA P/1.6048,1.339,1.1857,1.1,1.04,1./
```

```
DATA A/0.,.2,.4,.6,.8,1./
```

```
IL=5
```

```
KK=6
```

```
E=.001
```

```
HJ=778.16
```

```
PI=3.14159264
```

```
GO=32.174
```

```
LKL=C
```

```
LKL=LKL+1
```

```
READ(LL,217)WORD
```

398

COMPUTER PROGRAM #1

PGM10001
PGM10002
PGM10003
PGM10004
PGM10005
PGM10006
PGM10007
PGM10008
PGM10009
PGM10010
PGM10011
PGM10012
PGM10013
PGM10014
PGM10015
PGM10016
PGM10017
PGM10018
PGM10019
PGM10020
PGM10021
PGM10022
PGM10023
PGM10024
PGM10025
PGM10026
PGM10027
PGM10028
PGM10029
PGM10030
PGM10031
PGM10032
PGM10033
PGM10034
PGM10035
PGM10036

| | | |
|-----|--|----------|
| 217 | FORMAT(A80) | PGM10037 |
| 218 | WRITE(KK,218)WORD | PGM10038 |
| | FORMAT('1',APC) | PGM10039 |
| 219 | READ(LL,219)M1,M2,M,KLM,KTM,GAMMA,CP,POWER,W,TC,TET,TIPCLA,TOI,POI | PGM10040 |
| 220 | FORMAT(211,I3,215,2F5.3,F10.1,F10.3,3F5.3,2F10.1) | PGM10041 |
| 221 | READ(LL,211)PSI,PTE,PATROT,REPCIH,OMEGA,KLM1,M4,SPECWT,YMOD,C1,C2 | PGM10042 |
| | FORMAT(4F5.3,F10.3,I2,I8,F5.3,F15.0,2F10.0) | PGM10043 |
| | READ(LL,212)ENI,BSHAFT,PRATIO,ULTSRF,ALPHAH,EMACH,M3,NKL,N,APARA, | PGM10044 |
| | 1TOO | PGM10045 |
| 222 | FORMAT(F5.3,F10.5,F5.3,F10.0,F5.3,F10.4,I1,I2,I2,F5.4,F5.2) | PGM10046 |
| | POI=POI*144. | PGM10047 |
| | IF(M1.EQ.1)GO TO 214 | PGM10048 |
| | READ(LL,200)(RMAC(I),I=1,KLM) | PGM10049 |
| | READ(LL,201)(WTAP(I),I=1,KLM) | PGM10050 |
| | READ(LL,201)(PSPT(I),I=1,KIM) | PGM10051 |
| | READ(LL,201)(VELTOT(I),I=1,KLM) | PGM10052 |
| 200 | FORMAT(20F4.2) | PGM10053 |
| 201 | FORMAT(10F8.5) | PGM10054 |
| | READ(LL,227)(CFM(I),I=1,KIM) | PGM10055 |
| | READ(LL,227)(V(I),I=1,KIM) | PGM10056 |
| 227 | FORMAT(20F5.C) | PGM10057 |
| | KLM2=KLM-KLM1 | PGM10058 |
| | DO 230 I=1,KLM1 | PGM10059 |
| | II=I+KLM1-1 | PGM10060 |
| | RMAC11(I)=RMAC(I) | PGM10061 |
| | WTAP11(I)=WTAP(I) | PGM10062 |
| | PSPT11(I)=PSPT(I) | PGM10063 |
| | VEOT11(I)=VEITOT(I) | PGM10064 |
| | IF(I.GT.KLM2)GO TO 230 | PGM10065 |
| | RMAC12(I)=RMAC(II) | PGM10066 |
| | WTAP12(I)=WTAP(II) | PGM10067 |
| | PSPT12(I)=PSPT(II) | PGM10068 |
| | VEOT12(I)=VEITOT(II) | PGM10069 |
| 230 | CONTINUE | PGM10070 |
| 214 | SOMEGA=OMEGA*PI/RC. | PGM10071 |
| | GAMMA1=GAMMA/(GAMMA-1.) | PGM10072 |


```

5      LK=0
      TTEMP(3,N)=TCE
      TPRES(3,N)=PCE
      PHI=VX/(SOMEGA*RHE)
      PSI=GO*HJ*DHCS/(SOMEGA*RHE)**2
      IF((PSI.GT.2.0).AND.(REACTH.LT.0.05)) REACTH=0.05
      ALPHAH=ATAN(1.-REACTH-PSI/2.)/PHI
      IF(PSI.LT.2.0) ALPHAH=0.0
      ALPHA(1,3,N)=ALPHAH
      ALPHA(2,3,N)=ANGLE(PHE,ALHHAH,PHE)
      ALPHA(3,3,N)=ANGLE(RHE,ALPHAH,RHE)
      IF(EMACH.GT.0.0) GO TO 600
      DO 4 I=1,3
      CALL TP1(TTEMP(3,N),STEMP(I,3,N),ALPHA(I,3,N),TPRES(3,N),SPRES(I,3
3      1,N),DEN(I))
      RAD(1)=RHE
      RAD(2)=RHF
      RAD(3)=RTE
      CALL DENE(RAD,DEN,W,VXP)
      IF(ABS(VXP/VX-1.)>.5) GO TO 6
      J=J+1
      IF(J.GT.20) GO TO 6
      IF(VXP.GT.2.*VX) GO TO 656
      IF(J.EQ.2) GO TO 655
      IF(ABS(VXP/VX-1.)>.5) GO TO 655
      IF(VXP/VX.GT.1.) GO TO 654
      IF(LK.EQ.1) GO TO 657
      VX=VX-DELTAV
      JK=1
      LK=0
      GO TO 5
      DELTAV=DELTAV/2.
      VX=VX-DELTAV
      JK=1
      LK=0
      GO TO 5

```

657

PGM10100
PGM10110
PGM10111
PGM10112
PGM10113
PGM10114
PGM10115
PGM10116
PGM10117
PGM10118
PGM10119
PGM10120
PGM10121
PGM10122
PGM10123
PGM10124
PGM10125
PGM10126
PGM10127
PGM10128
PGM10129
PGM10130
PGM10131
PGM10132
PGM10133
PGM10134
PGM10135
PGM10136
PGM10137
PGM10138
PGM10139
PGM10140
PGM10141
PGM10142
PGM10143
PGM10144


```

654 IF(JK.EQ.1)GO TO 658
    VY=VX+DELTAV
    LK=1
    JK=0
    GO TO 5
658 DELTAV=DELTAV/2.
    VX=VX+DELTAV
    LK=1
    JK=C
    GO TO 5
655 VX=VXP
    GO TO 5
656 VX=1.5*VX
    J=1
    GO TO 5
659 H=0.9*(PTE-RHE)
    RMF=PTE-H/2.
    RHE=PTE-H
    J=1
    LK=0
    JK=0
    GO TO 5
6 IF(ENACH.LT.C.O)VX=VXP
    IF(ABS(ALPHA*180./PI).GT.25.)GO TO 650
    KKK=N
    DO 8 I=1,3
        RADIUS(I,3,N)=RAD(I)
        RHO(I,3,N)=DFN(I)
        REACT(I,3,N)=REACTI(RADIUS(I,3,N),ALPHA(T,3,N))
        BETA(I,3,N)=ANGLEB(ALPHA(I,3,N),RADIUS(T,3,N))
        GO TO 100
    J=0
    VX=ENACH*SQRT((2./(GAMMA+1.))*GO*BJ*TOE)
    H=.98*PHE
    DELTA=H/10.
    LK=0
    GO TO 100
    PGM10145
    PGM10146
    PGM10147
    PGM10148
    PGM10149
    PGM10150
    PGM10151
    PGM10152
    PGM10153
    PGM10154
    PGM10155
    PGM10156
    PGM10157
    PGM10158
    PGM10159
    PGM10160
    PGM10161
    PGM10162
    PGM10163
    PGM10164
    PGM10165
    PGM10166
    PGM10167
    PGM10168
    PGM10169
    PGM10170
    PGM10171
    PGM10172
    PGM10173
    PGM10174
    PGM10175
    PGM10176
    PGM10177
    PGM10178
    PGM10179
    PGM10180

```


PGM10073
PGM10074
PGM10075
PGM10076
PGM10077
PGM10078
PGM10079
PGM10080
PGM10081
PGM10082
PGM10083
PGM10084
PGM10085
PGM10086
PGM10087
PGM10088
PGM10089
PGM10090
PGM10091
PGM10092
PGM10093
PGM10094
PGM10095
PGM10096
PGM10097
PGM10098
PGM10099
PGM10100
PGM10101
PGM10102
PGM10103
PGM10104
PGM10105
PGM10106
PGM10107
PGM10108

```

RJ=CP*HJ/GAMMA1
J=1
RHE=RTE/EATIOR
RME=(RTE+PHE)/2.
IF(M2.EQ.1)GO TO 215
POWERP=POWER*550./HJ
DHO=POWERB/(W*ENT)
TOE=TOI-DHO/CP
TRATIO=TCE/TOI
ENP=ALOG(TRATIO)/ALOG((TRATIO-1.)/ENT+1.)
GAMMA2=GAMMA1/FNP
ECE=POI*TRATIO*GAMMA2
GO TO 216

215  PRATIO=1./PRATIO
      ENP=(ALOG(1.-ENT*(1.-PRATIO*(1./GAMMA1)))/LOG(PRATIO))*GAMMA1
      GAMMA2=GAMMA1/FNP
      TOR=TOI*PRATIO*(1./GAMMA2)
      DHO=CP*(TOI-TOR)
      POWER=DHO*ENT*W*HJ/550.
      POR=POI*PPATIO
      PHE=SOMPGA*PHE
      P01=POI/144.
      P02=POE/144.
      DHOS=PSI*(RHF*SOMEGA)**2/(GC*HJ)
      VX=.75*SOMEGA*PHE
      DELTAV=VX/1C.
      IP(N.GT.0)GO TO 2
      N=INT(DHO/DHOS)
      N=N+1
      RN=FLOAT(N)
      DHOS=DHO/RN
      IF((ALPHAH.EQ.0.0).AND.(PSI.LT.2.0)) PRACTH=1.-PSI/2.
      IF((PRACTH.LT.0.95).AND.(ALPHAH.EQ.0.0)) PSI=2.*(1.-C.C5)
      IF((PSI.LT.2.0).AND.(ALPHAH.EQ.0.0)) RHF=SQRT(DHOS*GO*HJ/(PSI*SOMEGA
1A**2))
      JK=0

```


| | | |
|-----|--|----------|
| 603 | JK=0 | PGM10181 |
| | J=J+1 | PGM10182 |
| | RAD(1)=RHE | PGM10183 |
| | RAD(2)=RHE+4/2. | PGM10184 |
| | RAD(3)=RHE+H | PGM10185 |
| | IF(PAD(3)/PHF.GT.2.0)GO TO 602 | PGM10186 |
| | PSI=GO*HJ*DHOS/(SOMEGA*PAD(1))*2 | PGM10187 |
| | PHI=VX/(SOMEGA*PAD(1)) | PGM10188 |
| | IF((PSI.GT.2.0).AND.(REACTH.LT.0.05)) REACTH=.05 | PGM10189 |
| | ALPHA=ATAN((1.-PEACTH-PSI/2.)/PHI) | PGM10190 |
| | IF(PSI.LT.2.)ALPHAH=C.0 | PGM10191 |
| | DO 611 I=1,3 | PGM10192 |
| | ALPHA(I,3,N)=ANGLE(PAD(1),ALPHAH,PAD(I)) | PGM10193 |
| 611 | CALL TP1(TTEMP(3,N),STEMP(I,3,N),ALPHA(I,3,N),TDEES(3,N),SPRES(I,3 | PGM10194 |
| | 1,N),DEN(I)) | PGM10195 |
| | CALL DENR(RAD,DEN,W,VXP) | PGM10196 |
| | RME=RAD(2) | PGM10197 |
| | RHE=RAD(1) | PGM10198 |
| | RTE=RAD(3) | PGM10199 |
| | IF(ABS(VXP/VX-1.).LE.E)GO TO 6 | PGM10200 |
| | IF(J.GT.20)GO TO 6 | PGM10201 |
| | IF(VXP/VX.LT.1.)GO TO 604 | PGM10202 |
| | IF(LK.EQ.1)GO TO 607 | PGM10203 |
| | H=H+DELTA | PGM10204 |
| | JK=1 | PGM10205 |
| | LK=0 | PGM10206 |
| | IF(H.GT.0.5*PTE)VX=VXP | PGM10207 |
| | GO TO 603 | PGM10208 |
| 607 | DELTA=DELTA/2. | PGM10209 |
| | H=H+DELTA | PGM10210 |
| | JK=1 | PGM10211 |
| | LK=0 | PGM10212 |
| | IF(H.GT.0.5*PTE)VX=VXP | PGM10213 |
| | GO TO 603 | PGM10214 |
| 604 | IF(JK.EQ.1)GO TO 608 | PGM10215 |
| | H=H-DELTA | PGM10216 |


```

LK=1
JK=0
GO TO 603
DELTA=DELTA/2.
H=H-DELTA
LK=1
JK=0
GO TO 603
RHE=RAD(3)/1.99
GO TO 601
J=3
K=J-2
TEMP(K,N)=ITEMP(J,N)+DHOS/CP
TPRES(K,N)=TPRES(J,N)/(TEMP(J,N)/TEMP(K,N))*GAMMA2
RADIUS(M,K,N)=RADIUS(M,J,N)
IF(N.EQ.1)GO TO 9
ALPHA(M,K,N)=ALPHA(M,J,N)
GO TO 10
9 ALPHA(M,K,N)=0.
10 BETA(M,K,N)=ANGLEB(ALPHA(M,K,N),RADIUS(M,K,N))
CALL TP1(TEMP(K,N),STEMP(M,K,N),ALPHA(M,K,N),TPRES(K,N),SPRES(M,K,N),DEN(M))
H=W/(2.*PI*VX*DEN(M)*RADIUS(M,K,N))
LK=0
JK=0
JJ=0
DELTA=H/10.
JJ=JJ+1
IF(M.EQ.1)GO TO 75
IF(M.EQ.3)GO TO 76
RAD(2)=RME
RAD(1)=RME-H/2.
RAD(3)=RME+H/2.
GO TO 77
75 RAD(1)=RHE
RAD(2)=RHE+H/2.

```

```

PGM10217
PGM10218
PGM10219
PGM10220
PGM10221
PGM10222
PGM10223
PGM10224
PGM10225
PGM10226
PGM10227
PGM10228
PGM10229
PGM10230
PGM10231
PGM10232
PGM10233
PGM10234
PGM10235
PGM10236
PGM10237
PGM10238
PGM10239
PGM10240
PGM10241
PGM10242
PGM10243
PGM10244
PGM10245
PGM10246
PGM10247
PGM10248
PGM10249
PGM10250
PGM10251
PGM10252

```



```

76 RAD(3)=RHE+H
   GO TO 77
   RAD(3)=RTE
   RAD(2)=RTE-H/2.
   RAD(1)=RTE-H
77 DO 11 I=1,3
   ALPHA(I,K,N)=ANGLE(RAD(M),ALPHA(M,K,N),RAD(I))
11 CALL IP1(TTEMP(K,N),STEMP(I,K,N),ALPHA(I,K,N),TOPES(K,N),SPDES(I,K,
   1,N),DEN(I))
   CALL DENE(RAD,DEN,N,VXP)
   IF(ABS(VXP/VX-1.)).LE.E)GO TO 15
   IF(JJ.EQ.20)GO TO 15
   IF(VXP/VX.LT.1.)GO TO 14
   IF(LK.EQ.1)GO TO 17
   H=H+DELTA
   JK=1
   LK=0
   GO TO 13
17 DELTA=DELTA/2.
   H=H+DELTA
   JK=1
   LK=0
   GO TO 13
14 IF(JK.EQ.1)GO TO 18
   H=H-DELTA
   LK=1
   JK=0
   GO TO 13
18 DELTA=DELTA/2.
   H=H-DPLTA
   LK=1
   JK=0
   GO TO 13
15 DO 16 I=1,3
   RHO(I,K,N)=DFN(I)
   RADIUS(I,K,N)=RAD(I)

```

```

PGM10253
PGM10254
PGM10255
PGM10256
PGM10257
PGM10258
PGM10259
PGM10260
PGM10261
PGM10262
PGM10263
PGM10264
PGM10265
PGM10266
PGM10267
PGM10268
PGM10269
PGM10270
PGM10271
PGM10272
PGM10273
PGM10274
PGM10275
PGM10276
PGM10277
PGM10278
PGM10279
PGM10280
PGM10281
PGM10282
PGM10283
PGM10284
PGM10285
PGM10286
PGM10287
PGM10288

```



```

16  RPACT(I,1,N)=REACT(RADIUS(I,K,N),ALPHA(I,K,N))
    BETA(I,K,N)=ANGLEB(ALPHA(I,K,N),RADIUS(I,K,N))
C SOLUTION FOR STATION 2
    ALPHA(M,2,N)=ANGLE2(RADIUS(M,3,N),ALPHA(M,3,N))
    TTMP2(2,N)=TTMP(1,N)
    CALL TPD2(TTMP(2,N),STMP(M,1,N),STMP(M,2,N),SPRES(M,
11,N),SPRES(M,2,N),ALPHA(M,2,N),DEN(M))
    H=W/(2.*PI*VX*DEN(M)*RADIUS(M,K,N))
    DELTA=H/10.
    JK=J
    LK=0
    JJ=0
    JJ=JJ+1
    IF(M.EQ.1)GO TO 78
    IF(M.EQ.3)GO TO 79
    RAD(2)=PME
    RAD(1)=RAD(2)-H/2.
    RAD(3)=RAD(2)+H/2.
    GO TO 80
78  RAD(1)=RHE
    RAD(2)=RHE+H/2.
    RAD(3)=RHE+H
    GO TO 80
79  RAD(3)=RTE
    RAD(2)=RTE-H/2.
    RAD(1)=RTE-H
    DO 25 I=1,3
    ALPHA(I,2,N)=ANGLE(PAD(M),ALPHA(M,2,N),RAD(I))
    CALL TPD2(TTMP(2,N),STMP(I,1,N),STMP(I,2,N),SPRES(I,
11,N),SPRES(I,2,N),ALPHA(I,2,N),DEN(I))
    CALL DENE(PAD,DEN,W,VXP)
    IF(ABS(VXP/VX-1.).LE.E)GO TO 30
    IF(VXP/VX.LT.1.)GO TO 34
    IF(JJ.EQ.20)GO TO 33
    IF(LK.EQ.1)GO TO 27
    H=H+DELTA

```

```

PGM10289
PGM10290
PGM10291
PGM10292
PGM10293
PGM10294
PGM10295
PGM10296
PGM10297
PGM10298
PGM10299
PGM10300
PGM10301
PGM10302
PGM10303
PGM10304
PGM10305
PGM10306
PGM10307
PGM10308
PGM10309
PGM10310
PGM10311
PGM10312
PGM10313
PGM10314
PGM10315
PGM10316
PGM10317
PGM10318
PGM10319
PGM10320
PGM10321
PGM10322
PGM10323
PGM10324

```



```

27 JK=1
    IK=0
    GO TO 33
    DELTA=DELTA/2.
    H=H+DELTA
    JK=1
    LK=0
    GO TO 33
34 IF (JK.EQ.1) GO TO 38
    H=H-DELTA
    LK=1
    JK=0
    GO TO 33
38 DELTA=DELTA/2.
    H=H-DELTA
    LK=1
    JK=0
    GO TO 33
    DO 36 I=1,3
    RHO(I,2,N)=DEN(I)
    RADIUS(I,2,N)=RAD(I)
    REACT(I,2,N)=REACT2(RADIUS(I,2,N),ALPHA(I,2,N))
    BETA(I,2,N)=ANGLF8(BETA(I,2,N),RADIUS(I,2,N))
    IF (N.EQ.1) GO TO 40
    NN=N-1
    DO 41 I=1,3
    ALPHA(I,3,NN)=ALPHA(I,1,N)
    BETA(I,3,NN)=BETA(I,1,N)
    RADIUS(I,3,NN)=RADIUS(I,1,N)
    STEMP(I,3,NN)=STEMP(I,1,N)
    SPRES(I,3,NN)=SPRES(I,1,N)
    RHO(I,3,NN)=RHO(I,1,N)
    TTEMP(3,NN)=TTEMP(1,N)
    TPRES(3,NN)=TPRES(1,N)
    N=N-1
    GO TO 100

```

```

PGM10325
PGM10326
PGM10327
PGM10328
PGM10329
PGM10330
PGM10331
PGM10332
PGM10333
PGM10334
PGM10335
PGM10336
PGM10337
PGM10338
PGM10339
PGM10340
PGM10341
PGM10342
PGM10343
PGM10344
PGM10345
PGM10346
PGM10347
PGM10348
PGM10349
PGM10350
PGM10351
PGM10352
PGM10353
PGM10354
PGM10355
PGM10356
PGM10357
PGM10358
PGM10359
PGM10360

```



```

40 WRITE(KK,43)W,POWER,TOI,TOE,PO1,PO2,EN,END
43 FORMAT('O','MASS FLOW(IB/S) POWER(HP) INLET TEMP(R) EXIT TEMP(R)
1) INLET PRESS(PSSI) EXIT PRESS(PSSI) ISSN EFFIC POLY EFFIC',/,
1,F12.2,F12.0,2F14.0,F14.3,F17.3,F16.3,F12.3)
ENACH=VX/SORT((2./(GAMMA+1.))*RJ*GO*TOE)
WRITE(KK,70)OMEGA,GAMMA,CP,REACTH,PSI,VX,EMICH
70 FORMAT('O','OMEGA(ZEM) GAMMA CP(RIU/LBMR) REACTION HUB PSI VX
1(FI/S) CRIT MACH NO AT EXIT',/,F10.0,F7.3,F12.4,F14.3,F7.3,F11.
11,F15.4)
DO 45 K=1,KK
DO 45 J=1,3
DO 45 I=1,3
IF(J.EQ.3)GO TO 37
REALMS(I,J,K)=SQRT((TIME(J,K)/STEMP(T,J,K)-1.)*2./(GAMMA-1.))
IJ=J+1
STEMPR(I,J,K)=STEMP(I,IJ,K)+(VX/COG(BETA(I,IJ,K)))*2/(2.*CP*HJ*GO
1)
RFLMR(I,J,K)=SQRT((ITEMPR(T,J,K)/STEMP(I,IJ,K)-1.)*2./(GAMMA-1.))
CONTINUE
37 ALPHA(I,J,K)=180.*ALPHA(I,J,K)/PI
BETA(I,J,K)=BETA(I,J,K)*180./PI
CONTINUE
45 WRITE(KK,42)
42 FORMAT('O','LOCATION STATION DENSITY ALPHA BETA REACTION RADIUS')
1AL PPFS STATIC PRESS DENSITY ALPHA BETA REACTION RADIUS)
DO 250 K=1,KK
DO 250 J=1,3
DO 250 I=1,3
IF((J.EQ.3).AND.(K.NE.KKK))GO TO 250
TOTPRE=TPRES(J,K)/144.
STPRE=SPRES(I,J,K)/144.
WRITE(KK,50)T,J,K,TEMP(I,J,K),TOTPRE,STPRE,PHO(I,J,K)
50 1,ALPHA(I,J,K),BETA(I,J,K),REACT(I,J,K),RADIUS(I,J,K)
FORMAT(' ',I5,I6,I8,F14.2,F12.2,F12.2,4X,F9.4,F7.2,3X,F6.2,F
18.3,F8.3)
250 CONTINUE

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PGM10361
PGM10362
PGM10363
PGM10364
PGM10365
PGM10366
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PGM10374
PGM10375
PGM10376
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PGM10378
PGM10379
PGM10380
PGM10381
PGM10382
PGM10383
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PGM10385
PGM10386
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PGM10390
PGM10391
PGM10392
PGM10393
PGM10394
PGM10395
PGM10396

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128 WRITE(KK,128)
   FORMAT('0',I,LOC,STAGE,RELATIVE,LOCAL,TEMP,LOC,POTOP,REAL,M
129 TACH,NO FOR,STATOR,RELATIVE,MACH,NO FOR,POTOP')
   WRITE(KK,129)
   FORMAT(' ',25X,'INLET',10X,'EXIT',7X,'INLET',7X,'EX
1,10X,'EXIT')
   WRITE(KK,130)((I,K,TEMPR(I,1,K),TEMPR(I,2,K),PPALMS(I,1,K),PPALM
1S(I,2,K),RELMP(I,1,K),RELMP(I,2,K),I=1,3),K=1,KKK)
130   FORMAT(' ',I8,I7,F13.2,F16.2,4X,F10.4,F13.4,F12.4,F16.4)
C CALCULATE BLADE ANGLES OF STATOR AND ROTOR
   DO 60 K=1,KKK
   DO 60 J=1,2
   DO 60 I=1,3
   IF(J.EQ.2)GO TO 49
   RRLADE(I,J,K)=BETA(I,2,K)
   SPLADE(I,J,K)=ALPHA(I,J,K)
   GO TO 60
49   CALL BLADE(ALPHA(I,2,K),PPALMS(I,2,K),SBLADE(I,J,K),1)
   BETAOT=-BETA(I,3,K)
   CALL BLADE(BETAOT,PFIMR(I,2,K),BLADE3,1)
   RBLADE(I,2,K)=-BLADE3
   CONTINUE
60   WRITE(KK,145)
145   FORMAT('0',LOC,STAGE,STATOR,ANGIN,GAS,ANGIN,STATOR,ANGOUT,GA
1S,ANGOUT,POTOP,ANGIN,GAS,ANGIN,POTOP,ANGOUT,GAS,ANGOUT')
   WRITE(KK,150)((J,K,SBLADE(J,1,K),ALPHA(J,2,K),SBLADE(J,2,K),ALPHA(
1J,2,K),RBLADE(J,1,K),BETA(J,2,K),RBLADE(J,3,K),BETA(J,3,K),J=1,3),
2K=1,KKK)
150   FORMAT(' ',I2,I6,8F12.1)
   SIGMAT=.75*MULTS*PF/1.2**2
   OMEGAT=SOMEGA
   STEPESC=0.0
   TF=.49+.51*PPRARA
   CALL FIG(6,A,B,ARPA,CF)
   DO 103 K=1,KKK
   DO 103 J=1,2

```



```

L=J+1
HEIGHT=(RADIUS(3,I,K)-RADIUS(1,I,K)+RADIUS(3,J,K)-RADIUS(1,J,K))/2.
1.
IF(J.EQ.1)GO TO 101
SMTANG=RRPLADE(1,1,K)-RRPLADE(1,2,K)
BETA3=-BETA(2,3,K)
CALL STOCRA(BETA(2,2,K),BETA3,PCPAT)
AREA=PI*(RADIUS(2,2,K)+RADIUS(2,3,K))*HEIGHT
STRESC=4.51*TF*SPLEWT*AREA*144.*(OMEGA/1000.)**2
A2=PI*ALPHA(2,2,K)/180.
A3=PI*ALPHA(2,3,K)/180.
IF(J.EQ.2)GO TO 102
SMTANG=SRBLADE(3,2,K)-SRBLADE(3,1,K)
A1=-ALPHA(2,1,K)
A2=ALPHA(2,2,K)
CALL STOCRA(A1,A2,PCPAT)
A2=PI*A2/180.
A3=PI*A1/180.
CALL SECMOD(SMTANG,TC,SM,ARFAC2)
STRESB=W*VX*(TAN(A2)-TAN(A3))*PCPAT*HEIGHT/(SM*2.*PI*(RADIUS(2,J,K)
1)+RADIUS(2,L,K))*GO)
PPEQ=11.0*CF*SORT(YMOD*SM*TC/(SPECWT*AREAC2))/(HEIGHT**2*144.)
C4=1.3*41.*(OMEGA/60.)*STRESB/PRFO
C5=C4/(C1*(1.-STRFSC/C2))
IF(J.EQ.1)C5=C4/C1
CHORD=C5**(1./3.)
FREQ1=PPFQ*CHORD
HARMON=FFEQ1/(OMEGA/60.)
IF(CHORD.LT..6)CHORD=.6
CK1=AREAC2*CHORD**2
CK2=CK1*(1.-7*PEAPA)
CHORDR(2,J,K)=SORT((CK1-CK2/2.)/AREAC2)
IF(J.EQ.1)GO TO 61
CHORDR(1,J,K)=CHORD
CHORDR(3,J,K)=SORT((CK1-CK2)/(AREAC2)
GO TO 62

```

101

102


```

61 CHORDR(3,J,K)=CHORD
   CHORDP(1,J,K)=SQRT((CK1-CK2)/AREA2)
62 CONTINUE
   SIGMAB(J,K)=STRESSB/CHORD**2
   BLADEH(J,K)=HEIGHT*12.
   ASPECR(J,K)=BLADEH(J,K)/CHORDR(2,J,K)
   PTOCHO(J,K)=PCFAT
   IF(J.EQ.2) SIGMAC(K)=STPESC
   PITCH=PCFAT**CHORDR(2,J,K)
   BLADNE=12.*PI*(RADIUS(2,J,K)+RADIUS(2,L,K))/PITCH
   KL=INT(BLADNE)
   IF(J.EQ.2) GO TO 63
   KL=(KL/2)*2
   GO TO 64
63 CALL TESTP(KI,KL)
   IF(KI.EQ.1) GO TO 64
   KL=KL-1
   GO TO 63
64 NOBLAD(J,K)=KL
   PITCH=12.*PI*(RADIUS(2,J,K)+RADIUS(2,L,K))/FLOAT(KL)
   PTOCHO(J,K)=PITCH/CHORDR(2,J,K)
   RHUB=(RADIUS(1,L,K)+RADIUS(1,J,K))/2.
   RTIP=RHUB+HEIGHT
   ALPHAW=(1.-AFEARA)/PTIP
   BIADWT(J,K)=12.*CK1*SPECWT*(ALPHAW/2.)*(RHUB**2-RTIP**2)+PTIP-3HU
1B) *FLOAT(NOBLAD(J,K))
   IF(J.EQ.1) GO TO 98
   WC=0.65*CHORDR(1,J,K)
   IF(WO.LT.0.7) WC=0.7
   IF(RSHAFT.LT.C.01) PSHAFT=PAD(W)/6.
   RI=2.*PSHAFT
   TI=PSHAFT
   TC=WO/24.
   RO=RHUB-TO
   ROPT=RO/RTIP
   PHET=RHUB/RTIP

```



```

PIRT=RI/RTIP
SIGROW=12.*SPECWT*OMEGAT**2*PI*PI**2/(SIGMAT*GO)
ALPPT=ALPHAW*RTIP
ABWOT=FLOAT(NBELAD(J,K))*CK1/(WO*TO*12.)
CBLAD1=(1.-PHRT**2-(2.*ALPPT/3.)*(1.-PHRT**3))
CBLADE=(ABWOT/(4.*PI))*CBLAD1
DFLTAR=(90-PI)/10.
ZO=(WO*TO/RO)*( (ROBT**2+CBLADF)*SIGROW-1.)
IF(ZO.LT.WO/5.)ZO=WO/5.
IF(ZO.LT.0.3)ZO=0.3
ZI=ZO*EXP((SIGROW/2.)*(ROBT**2-PI*PI**2))
WI=(ZI*RI/TI)/(1.-RI*PI**2*SIGROW)
IF(WI.LT.1.2*WO)WI=1.2*WO
WTMID=PI*SPECWT*288.*ZO*RTIP**2*EXP((SIGROW/2.)*ROBT**2)*(EXP(-(SIGROW/2.)*RI*PI**2)-EXP(-(SIGROW/2.)*FORBT**2))/SIGROW
WTIPM=SPECWT*WI*PI*144.*(PI**2-3*SHAPE**2)
WTORIM=SPECWT*WO*PI*144.*(RHUR**2-RO**2)
WTDISC(K)=WTIPIM+WTCPIM*WTMID
WTOT=WTDISC(K)+ELADWT(J,K)
R(1)=RSHAFT
Z(1)=WI
R(2)=RI
Z(2)=WI
E(3)=RI
Z(3)=ZI
R(14)=RO
Z(14)=WO
R(15)=RHUR
Z(15)=WO
SUM=0.
SUM2=0.
DO 84 II=4,13
IJ=II-1
R(II)=R(IJ)+DFLTAR
Z(II)=ZO*EXP((SIGROW/2.)*(ROBT**2-(R(II)/RI*PI)**2))
SUMZ=(Z(II)+Z(IJ))/2.

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PGM10505
 PGM10506
 PGM10507
 PGM10508
 PGM10509
 PGM10510
 PGM10511
 PGM10512
 PGM10513
 PGM10514
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 PGM10516
 PGM10517
 PGM10518
 PGM10519
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 PGM10521
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 PGM10532
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 PGM10534
 PGM10535
 PGM10536
 PGM10537
 PGM10538
 PGM10539
 PGM10540


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84 SUM=SUM+SUMZ*DELTA
   SUM2=SUM2+SUMZ*( (P(IJ)+P(IJ))/2.)*2*DELTA
CONTINTE
AREADT=24.*(WC*TO+WZ*TI+SUM)
FORORI=WC*(RHUB**3-FC**3)
FORIPI=WZ*(RI**3-RSHAPT**3)
FORMID=SUM2
FORCT=(SPECWT/GO)*(288.*(FORIPT/3.+FOROPT/3.+FORMID)+FICAT(NOBLAD(
1J,K))*RTIP**2*CK1*6.*CBLAD1/PI)
BURST=FORCT/PPFADT
SPEEDB=SQRT(ULTSRE/BURST)*30./PI
BSOS=SPEEDB/CMEGA
88 WRITE(KK,89)J,K,NORLAD(J,K),BLADWT(J,K)
89 FORMAT('0','POW STAGE NO BLADES TOTAL BLADE WTS',' ',I3,I7,-11,
1F10.2)
IF(J.EO.1)GO TO 95
WRITE(KK,121)J,K,CHORDP(1,J,K),CHOEDR(2,J,K),CHORD3(3,J,K),BLADEH(
1J,K),PREO1,HARMON,PTOCHO(J,K),ASPECT(J,K),SIGMAP(J,K),SIGMAC(K)
121 FORMAT('0','POW STAGE CHORD1 CHORD2 CHORD3 HEIGHT 1-REND RPE
10 HARMONIC P/C H/C BENDING STRESS CENTRIFUGAL STRESS',' ',I3,
3I7,4F8.3,3F10.1,3F10.1,3F10.3,3F6.3,3F10.1,3F17.1)
WRITE(KK,92)WTORIM,WTRIM,WTHMD,WTDISC(K),WTOI,RTIP,RHUB,SPEEDB,BS
105
92 FORMAT('0','WTORIM WTRIM WTHMD WTDISC WTC RTIP RHUB BURST
1 SPEED BS/OMEGA',' ',F6.2,F8.2,F10.2,F7.2,2F6.3,F10.3,F9.2)
DO 96 I=1,15
R(I)=R(I)*12.
96 WRITE(KK,97)
97 FORMAT('0','DISC DIMENSIONS FROM RSHAPT TO FHUB(INCHES)')
WRITE(KK,93)(I,R(I),I,Z(I),I=1,15)
93 FORMAT(' ','R(I2)=' ,F5.2,' Z(I2)=' ,F5.3)
GO TO 98
95 WRITE(KK,120)J,K,CHORDP(1,J,K),CHOEDR(2,J,K),CHOEDR(3,J,K),BLADEH(
1J,K),PREO1,HARMON,PTOCHO(J,K),ASPECT(J,K),SIGMAP(J,K)
120 FORMAT('0','POW STAGE CHORD1 CHORD2 CHORD3 HEIGHT 1-BEND RPE
10 HARMONIC P/C H/C BENDING STRESS',' ',I3,I7,4F8.3,2F10.1,3F10.

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13,P6.3,P10.1)
CONTINUE
CONTINUE
WRITE(KK,403)
FORMAT(' ',' STAGE STATION DISTANCE FROM STATION 1, STAGE 1')
J=1
K=1
SUM=0.0
WRITE(KK,402)K,J,SUM
FORMAT(' ',I7,I8,F15.4)
J=J+1
IF(J.EQ.1)J1=2
IF(J.EQ.2)J1=1
IF(J.EQ.1)K1=K-1
IF(J.EQ.2)K1=K
IF(J.EQ.1)J3=1
IF(J.EQ.2)J3=2
SUM=SUM+.11875*CHORDR(2,J1,K1)+1.1875*CHORDR(2,J,K)+.11875*CHORDR(
12,J3,K)
WRITE(KK,402)K,J,SUM
IF((J.EQ.2).AND.(K.EQ.KKK))GO TO 401
IF(J.EQ.1)GO TO 415
K=K+1
IF(J.EQ.2)J=0
GO TO 415
SUM=SUM+1.36625*CHORDF(2,2,KKK)+.11875*CHORDR(2,1,KKK)
J=3
WRITE(KK,402)K,J,SUM
WRITE(KK,404)
FORMAT('0',' LENGTH(IN) RADHUBINLET(IN) RADTIPINLET(IN) RADH9BE
1XIT(IN) RADTIFEXIT(IN) WTOPSHAFT(LP) WTOP DISCS+BLADES(LB)')
RAD1H=RADIUS(1,1,1)*12.
RAD1T=RADIUS(3,1,1)*12.
RAD2H=RADIUS(1,3,KKK)*12.
RAD2T=RADIUS(3,3,KKK)*12.
SUM1=0.0

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SUM2=0.0
SHAPEWT=SPECWT*PI*(2*SHAPEWT*12.)*2*SUM
DO 410 K=1,KKK
DO 409 J=1,2
409 SUM1=SUM1+BLADWT(J,K)
410 SUM2=SUM2+SUM1+WIDISC(K)
405 WRITE(KK,405)SUM,PAD1H,PAD1T,PAD2H,PAD2T,SHAPEWT,SUM2
405 FORMAT(' ',P9.2,P15.2,5F16.2)
779 WRITE(7,779)CP,GO,HJ,PJ,VX,TOI,DOI,E,W,KKK
779 FORMAT(F5.3,F10.4,F8.3,F10.4,F10.1,F10.1,F10.3,P5.3,P10.3,I2)
780 WRITE(7,780)GAMMA1,PI,SOMEGA,IET,TC,TIPCLA,GAMMA,KLM1,KLM2
780 FORMAT(F10.5,F10.9,F10.3,4F5.3,I2,2F10.3)
782 WRITE(7,782)((RADIUS(I,J,K),I=1,3),J=1,3),K=1,KKK)
782 FORMAT(8F10.4)
781 WRITE(7,781)((TEMP(J,K),J=1,3),K=1,KKK)
781 FORMAT(8F10.1)
781 WRITE(7,781)((TEMP(2,J,K),J=1,3),K=1,KKK)
781 WRITE(7,781)((TPFFS(J,K),J=1,3),K=1,KKK)
781 WRITE(7,781)((SPFFS(2,J,K),J=1,3),K=1,KKK)
781 WRITE(7,782)((PHO(2,J,K),J=1,3),K=1,KKK)
781 WRITE(7,782)((ALPHA(2,J,K),J=1,3),K=1,KKK)
781 WRITE(7,782)((BETA(2,J,K),J=1,3),K=1,KKK)
781 WRITE(7,782)((PEAIMS(2,J,K),J=1,2),K=1,KKK)
781 WRITE(7,782)((PEIMR(2,J,K),J=1,2),K=1,KKK)
781 WRITE(7,782)((SBLADE(2,J,K),J=1,2),K=1,KKK)
781 WRITE(7,782)((PBLADE(2,J,K),J=1,2),K=1,KKK)
781 WRITE(7,782)((CHORDE(2,J,K),J=1,2),K=1,KKK)
781 WRITE(7,782)((ASEFCE(J,K),J=1,2),K=1,KKK)
781 WRITE(7,782)((PTOCHO(J,K),J=1,2),K=1,KKK)
IF(M4.P0.1)GC TO 399
DO 350 MK=1,5
OMEGAB(MK)=SOMEGA*SEFFD(MK)
DO 350 MJ=1,7
K=1
BETAC(1,K)=0.0
ALPHAC(1,K)=0.0

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PGM10613
PGM10614
PGM10615
PGM10616
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PGM10632
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PGM10636
PGM10637
PGM10638
PGM10639
PGM10640
PGM10641
PGM10642
PGM10643
PGM10644
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PGM10647
PGM10648
113

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TOTC(1,K)=TOI
VAXIAL(1,K)=VX
TSC(1,K)=TOI-VX**2/(2.*CP*HJ*GO)
POC(1,K)=POI
PSC(1,K)=POC(1,K)/(TOTC(1,K)/TSC(1,K))*GAMMA1
RHOC(1,K)=PSC(1,K)/(TSC(1,K)*PJ)
POI=POC(1,K)
TOI=TOI
GANG1=0.0
J=1
L=J+1
JJ=0
U=OMEGAB(MK)*RADIUS(2,L,K)
PHO2=PHO(2,L,K)
RMAC2=.15*REALMS(2,2,K)
IF(J.EQ.2)PMAC2=.15*VFIMB(2,2,K)
BLADE2=SLADE(2,2,K)
IF(J.EQ.2)BLADE2=BLADE(2,2,K)
ANNULA=PI*(RADIUS(3,L,K)**2-DIMS(1,L,K)**2)
JJ=JJ+1
IF(J.EQ.2)BLADE2=-BLADE2
CALL BLADE(GANG2,PMAC2,BLAD2,2)
IF(J.EQ.2)GANG2=-GANG2
IF(J.EQ.2)BLADE2=-BLADE2
CALL FIG(KLM,PMAC,VFI TOT,PMAC2,VFI TOT)
V2=SORT(TC1)=VFI TOT
CALL FIG(KLM,PMAC,PSPT,PMAC2,PSPT)
T2=TOI*PSPT**2*(1./GAMMA1)
IF(PSPT2.LT.O.C)GO TO 390
BLADE1=SLADE(2,1,K)
IF(J.EQ.2)BLADE1=BLADE(2,1,K)
IF(J.EQ.1)KLEAP=1
IF(J.EQ.2)KLEAP=2
IF((J.EQ.2).AND.(M3.EQ.1))KLEAP=2
CALL PLOSS(GANG1,GANG2,PHO2,T2,BLAD2,PTOCHO(J,K),ASPEC(J,K),P4AC
12,V2,CHOPDR(2,J,K),J,K,KLEAP,YT,KTM,TEM,V)

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304
306

300


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316 P02=PO1/(YT*(1.-PSP2)+1.)
    Q2=WDOTP(MJ)*W*SQRT(TO1)/(ANNULA*COS(GANG2*PI/180.)*PO2)
    IF(Q2.GT.WTAP(KLM1))Q2=WTAP(KLM1)*.90
    IF(RMAC2.GT.RMAC(KLM1))GO TO 316
    CALL FIG(KLM1,WTAP11,RMAC11,Q2,RMAC2P)
    CALL FIG(KLM1,WTAP11,VFOT11,Q2,VFOT2P)
    CALL FIG(KLM1,WTAP11,PSP11,Q2,PSP2P)
    GO TO 317
317 CALL FIG(KLM2,WTAP12,RMAC12,Q2,RMAC2P)
    CALL FIG(KLM2,WTAP12,VFOT12,Q2,VFOT2P)
    CALL FIG(KLM2,WTAP12,PSP12,Q2,PSP2P)
    P02=PO1/(YT*(1.-PSP2)+1.)
    P2=PO2*PSP2P
    T2=TO1*PSP2P*(1./GAMMA1)
    RHO2=P2/(T2*PJ)
    V2=SQRT(TO1)*VFOT2P
    IF(ABS(RMAC2P/RMAC2-1.).LT.E)GO TO 302
    IF((RMAC2P/GT.C.9).AND.(JJ.GT.3))GO TO 304
    IF(JJ.EQ.10)GO TO 302
    IF(ABS(RMAC2P/RMAC2-1.).LT.0.03)GO TO 308
    RMAC2=RMAC2P
    GO TO 300
318 RMAC2=RMAC2+(RMAC2P-RMAC2)/2.
    GO TO 300
304 O2PRIM=1.0001*Q2
    IF((RMACP.GT.1.0))GO TO 330
    CALL FIG(KLM1,WTAP11,PSP11,Q2PP1M,PSP1P)
    GO TO 331
330 CALL FIG(KLM2,WTAP12,PSP12,Q2PP1M,PSP1P)
331 P02PO1=1./(YT*(1.-PSP2)+1.)
    WPRIME=Q2PP1M*P02PO1*ANNULA*COS(GANG2*PI/180.)*PO1/SQRT(TO1)
    IF(WPRIME.LT.W*VDOCTP(MJ))GO TO 305
    GO TO 315
305 WRITE(KK,310)J,K,SPEED(MK),PO1/1000.
310 FORMAT(' ','DOWN',I2,' IN STAGE',I3,' FOR N/SQRT(IO1) = ',F4.1,' FO
1R PO1 = ',F8.1,' CHOKED')

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PGM10685
PGM10686
PGM10687
PGM10688
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PGM10699
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PGM10701
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PGM10705
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PGM10708
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PGM10713
PGM10714
PGM10715
PGM10716
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PGM10718
PGM10719
PGM10720

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CALL FIG(KLM2,WTAP12,PSPT12,Q2,PSPT2)
 CALL FIG(KLM2,WTAP12,VEOT12,Q2,VEOT2)
 V2=SQRT(TO1)*VEOT2

PO2=PO1/(YT*(1.-PSPT2)+1.)

P2=PO2*PSPT2

T2=TO1*PSPT2**(1./GAMMA1)

RHO2=P2/(T2*PJ)

GANG2=ARCCOS(W*WDOCTD(MJ)*SQRT(TO1)/(PO2*Q2*ANNUIA))*180./PI

IF(J.EQ.2)GO TO 307

J=J+1

ALPHAC(2,K)=GANG2

BETAC(2,K)=(ATAN(TAN(GANG2*PI/180.)-U/(V2*CCS(GANG2*PI/180.)))*19

10./PI

RHOC(2,K)=RHO2

VWSQ=(V2*CCS(GANG2*PI/180.))*2*(1.+TAN(BETAC(2,K)*PI/180.))*2)

VAXIAL(2,K)=V2*CCS(GANG2*PI/180.)

TOTC(2,K)=TO1

TSC(2,K)=TO1-V2**2/(2.*CP-HJ*GO)

TO1=T2+VWSQ/(2.*CP-HJ*GO)

PSC(2,K)=P2

POC(2,K)=PC2

RVTOTR=SQRT(VWSQ)/SQRT(TO1)

CALL FIG(KLM,VETCT,PSPT,RVTOTE,PSPT2)

PO1=P2/PSPT2

GANG1=BETAC(2,K)

GO TO 306

307

TSC(3,K)=TO1-V2**2/(2.*GO*HJ*CP)

VAXIAL(3,K)=V2*CCS(GANG2*PI/180.)

PSC(3,K)=PO2*PSPT2

RHOC(3,K)=RHC2

BETAC(3,K)=GANG2

U=OMEGAB(MK)*RADIUS(2,3,K)

ALPHAC(3,K)=(ATAN(TAN(GANG2*PI/180.)+U/(V2*CCS(GANG2*PI/180.)))*1

180./PI

VELSQ=(V2*CCS(GANG2*PI/180.))*2*(1.+TAN(ALPHAC(3,K)*PI/180.))*2)

TOTC(3,K)=TSC(3,K)+VELSQ/(2.*GO*HJ*CP)

PGM10721
 PGM10722
 PGM10723
 PGM10724
 PGM10725
 PGM10726
 PGM10727
 PGM10728
 PGM10729
 PGM10730
 PGM10731
 PGM10732
 PGM10733
 PGM10734
 PGM10735
 PGM10736
 PGM10737
 PGM10738
 PGM10739
 PGM10740
 PGM10741
 PGM10742
 PGM10743
 PGM10744
 PGM10745
 PGM10746
 PGM10747
 PGM10748
 PGM10749
 PGM10750
 PGM10751
 PGM10752
 PGM10753
 PGM10754
 PGM10755
 PGM10756


```

POC(3,K)=PSC(3,K)*(TOTC(3,K)/TSC(3,K))*GAMMA
IF(K.EQ.KKK)GO TO 309
L=K
K=K+1
ALPHAC(1,K)=ALPHAC(3,L)
VAXIAL(1,K)=VAXIAL(3,L)
BETAC(1,K)=BETAC(3,L)
TSC(1,K)=TSC(3,L)
POC(1,K)=POC(3,L)
PSC(1,K)=PSC(3,L)
TOTC(1,K)=TOTC(3,L)
RHOC(1,K)=RHOC(3,L)
GANG1=ALPHAC(1,K)
T01=TOTC(1,K)
P01=POC(1,K)
GO TO 308
309 PO1P02(MJ)=PCI/POC(3,KKK)
ENHJ(MJ)=(TOTC(1,1)-TOTC(3,KKK))/(TOTC(1,1)*
1,1)**(1./GAMMA1))
ENSMJ(MJ)=(TOTC(1,1)-TOTC(3,KKK))/(TOTC(1,1)*
1,1)**(1./GAMMA1))
WRITE(KK,311)SPEED(MK),PO1P02(MJ)
FORMAT('0','FOR DESIGN POINT CALCULATIONS AT M
1 CYCLE CALCULATIONS WILL BE COMPARED WITH PERF
2ATIONS OF IMPROVED AINSLEY MATHIESON METHOD AS
3CULATIONS AT DESIGN POINT',/,',FOR N/SORT('01
3,F5.2)
WRITE(KK,312)
FORMAT('0','STATION STAGE TOTAL TEMPERATURE
1AL PRES(P5I) STATIC PRES(P5I) DENSITY(LB/FT
313 FORMAT(' ',I4,I8,F11.1,F8.1,F10.1,F8.1,F9.1,F7
13,4F6.1)
DO 320 K1=1,KKK
DO 320 J1=1,3
TPRES2=TPRES(J1,K1)/144.
TPRES3C=POC(J1,K1)/144.

```



```

320 SPRES2=SPRES(2,J1,K1)/144.
    SPRES=PSC(J1,K1)/144.
    IF(J1.EQ.3).AND.(K1.NE.KKK)GO TO 321
    WRITE(KK,313)J1,K1,TEMP(J1,K1),TOTC(J1,K1),STEMP(2,J1,K1),TSC(J1,
1K1),TPRES2,TPRES,SPRES2,SPRES,PHO(2,J1,K1),RHOC(J1,K1),ALPHA(2,J
11,K1),ALPHAC(J1,K1),BETA(2,J1,K1),BETAC(J1,K1)
    CONTINUE
    WRITE(KK,345)
345 FORMAT(' ','STAGE VX STATION 1 VX STATION 2 VX STATION 3')
    WRITE(KK,344)(K1,(VAXIAL(J1,K1),J1=1,3),K1=1,KKK)
344 FORMAT(' ',I4,3F13.0)
    IF(MJ.NE.7)GO TO 350
    WRITE(KK,326)SPEED(MK)
326 FORMAT('0','FOR V/SOPT(TC1) EQUAL TO ',F5.2)
    WRITE(KK,328)
328 FORMAT('0',' PQ1/PQ2 TOTAL EFFICIENCY STATIC EFFICIENCY M*EORT(
1TO1)/PQ1')
    WRITE(KK,327)(PC1PO2(I),ENTMJ(I),ENSMJ(I),WDOIP(I),I=1,7)
327 FORMAT(' ',F7.2,F15.4,F18.4,F17.3)
350 CONTINUE
399 CONTINUE
    IF(LKL.NE.NKL)GO TO 398
    STOP
    END
    SUBROUTINE PLOSS(ANGIN,ANGOUT,RHO,TEMP,BLADIV,PTUCHO,ASPECB,REMAICH
1,VELOT,CHCED,J,K,KLEAR,YI,KTM,TEMP,U)
    COMMON PI,GO,RJ,HJ,GAMMA,VX,SOMEGA,DHOS,CP,GAMMA2,GAMMA1,KK
    COMMON/AREA1/TEI,TC,TIPCLA
    DIMENSION B180(9),B275(9),A3(9),B370(9),B365(9),B360(9),B350(9),B3
140(9),A5(5),B5(5),A15(5),B1540(5),B1550(5),B1560(5),B1570(5),A7(7)
1,B7(7),A811(11),B865(11),B860(11),B855(11),B850(11),B840(11),A8(9)
3,B8830(9),PCA(7),PCB(7),YF2A(7),YP2B(7),TEY(7),TEY(7),ANG80(7),ANG7
40(6),ANG30(7),YF1A(6),YPN1(7),SI75(7),ANG47(4),SIDA(4),PI(9),TEM(K
5TM),U(KTM),B170(9),B165(9),B160(9),B155(9),B150(9),B140(9)
    DATA A3/.3,.4,.5,.6,.7,.8,.9,1.,1.1/
    DATA B180/.07149,.06468,.0594,.05889,.06128,.06553,.07285,.0834,.0

```


19532/
 DATA B275/.06979,.05872,.05021,.04766,.04817,.05140,.05651,.06383,
 1.074/
 DATA B370/.06004,.05532,.04511,.03966,.03830,.04,.0434,.04036,.057
 187/
 DATA B365/.06809,.05498,.04255,.03574,.03098,.02028,.03115,.03643,
 1.04494/
 DATA B360/.06638,.05362,.04068,.03149,.02689,.02417,.02451,.02879,
 1.03404/
 DATA B350/.06519,.05055,.03940,.0303,.02502,.02264,.02128,.02077,.
 102247/
 DATA B340/.06468,.0497,.03745,.02911,.02383,.02043,.01855,.01804,.
 101872/
 DATA A1/.3,.4,.5,.6,.7,.8,.9,1,.1/
 DATA B170/.16112,.14186,.13640,.13981,.14868,.16198,.17596,.19006,
 1.2077/
 DATA B165/.15618,.13026,.11867,.11628,.12080,.12907,.13981,.15102,
 1.1668/
 DATA B160/.14834,.11799,.10571,.10196,.10401,.10827,.11475,.12361,
 1.1329/
 DATA B155/.14749,.11509,.09787,.08747,.08491,.08730,.09379,.10315,
 1.1166/
 DATA B150/.14408,.11168,.09122,.07843,.0731,.07502,.0798,.08781,.0
 1987/
 DATA B140/.13981,.10656,.0861,.07315,.06591,.0653,.0682,.07398,.03
 1/
 DATA A5/.4,.5,.6,.7,.8/
 DATA B5/8.0576,6.8921,4.6043,1.4532,-2.3022/
 DATA A15/.8,.85,.9,.95,1./
 DATA B1540/-2.3022,-3.3813,-4.6762,-5.5396,-6.4023/
 DATA B1550/-2.3044,-4.4504,-6.6187,-8.9928,-11.3660/
 DATA B1560/-2.3066,-4.7626,-8.0432,-11.3669,-14.777/
 DATA B1570/-2.3088,-5.2227,-9.6403,-14.1726,-19.1367/
 DATA A7/-9,-6,-4,-2,0,.2,.4/
 DATA B7/11.218,25.671,33.437,40.988,44.223,43.576,40.34/
 DATA A81/-0,-.8,-.6,-.4,-.2,0,.2,.4,.6,.8,1./

PGM10829
 PGM10830
 PGM10831
 PGM10832
 PGM10833
 PGM10834
 PGM10835
 PGM10836
 PGM10837
 PGM10838
 PGM10839
 PGM10840
 PGM10841
 PGM10842
 PGM10843
 PGM10844
 PGM10845
 PGM10846
 PGM10847
 PGM10848
 PGM10849
 PGM10850
 PGM10851
 PGM10852
 PGM10853
 PGM10854
 PGM10855
 PGM10856
 PGM10857
 PGM10858
 PGM10859
 PGM10860
 PGM10861
 PGM10862
 PGM10863
 PGM10864


```

DATA B865/10.355,14.453,21.788,28.562,34.30,37.105,37.105,34.732,3
10.503,24.808,10.199/
DATA B866/8.025,11.563,17.397,22.435,26.318,28.093,30.547,29.899,2
17.57,23.384,19.199/
DATA B867/7.982,10.01,14.324,17.915,21.141,23.643,25.283,25.369,24
1.032,21.788,10.1990/
DATA B868/7.162,8.802,11.735,14.367,16.611,18.552,19.976,20.892,20
1.796,20.192,19.631/
DATA B869/6.04,6.903,8.845,10.355,11.865,13.289,14.626,15.532,15.4
181,17.56,18.984/
DATA A8/-9,-8,-6,-4,-2,0,2,4,6/
DATA B830/4.099,5.177,6.472,7.55,8.629,9.406,10.268,10.786,11.045/
DATA PCA/4,5,6,7,8,9,1/
DATA PCB/1,1069,1.0784,1.0483,1.0155,.9867,.9405,.9/
DATA YP2A/-4,-3,-2,-1,0,1,1.5/
DATA YP2B/6.1573,4.2565,2.7395,1.5663,1.,2.0922,4.5599/
DATA TEX/0,02,04,06,08,1,12/
DATA IFY/.91304,1.,1.1087,1.239,1.3826,1.5304,1.6913/
DATA ANG89/40.,50.,60.,65.,73.,75.,80./
DATA ANG70/40.,50.,55.,60.,65.,70./
DATA ANG47/40.,50.,60.,70./
DATA ANG30/30.,40.,50.,55.,60.,65.,70./
U1=U(KTM)
IF(TEMP.GT.TPM(KTM))GO TO 1
CALL FIG(KTM,TFM,U,TEMP,U1)
RF=((RHO*CHOPD*VELOT)/(U1*12.))*1.37
YPI=0.2
A1=ANGIN
IF(J.EQ.1)A1=-ANGIN
A2=ANGOUT
IF(J.EQ.2)A2=-ANGOUT
B1=BLADIN*PI/180.
CINCI=ANGIN-BLADIN
IF(J.EQ.1)CINCI=BLADIN-ANGIN
IF((PTOCHO.LT.C.4).OE.(PTOCHO.GT.1.1))GO TO 25
IF((ABS(ANGOUT).GT.80.).OP.(ABS(ANGOUT).LT.30.))GO TO 25

```

PGM10865
 PGM10866
 PGM10867
 PGM10868
 PGM10869
 PGM10870
 PGM10871
 PGM10872
 PGM10873
 PGM10874
 PGM10875
 PGM10876
 PGM10877
 PGM10878
 PGM10879
 PGM10880
 PGM10881
 PGM10882
 PGM10883
 PGM10884
 PGM10885
 PGM10886
 PGM10887
 PGM10888
 PGM10889
 PGM10890
 PGM10891
 PGM10892
 PGM10893
 PGM10894
 PGM10895
 PGM10896
 PGM10897
 PGM10898
 PGM10899
 PGM10900


```

1 IF (ABS (BLADIN) .LT. 0.5) GO TO 6
2 IF (ABS (ANGOUT) .GT. 7C.) GO TO 2
3 IF (ABS (ANGOUT) .LT. 4C.) GO TO 4
4 CALL FIG (9, AI, BI40, PTOCHO, YPIA (1))
5 CALL FIG (9, AI, BI50, PTOCHO, YPIA (2))
6 CALL FIG (9, AI, BI55, PTOCHO, YPIA (3))
7 CALL FIG (9, AI, BI60, PTOCHO, YPIA (4))
8 CALL FIG (9, AI, BI65, PTOCHO, YPIA (5))
9 CALL FIG (9, AI, BI70, PTOCHO, YPIA (6))
10 CALL FIG (6, ANG70, YPIA, I2, YPI)
11 GO TO 6
12 WRITE (KK, 3) J, K
13 POPMAT (I, I, 'THE ANGLE OUT WAS GREATER THAN 70 DEG AND THE DATA WAS
14 1 FROM 70 DEG FOR ROW', I3, 'STAGE', I3)
15 GO TO 6
16 WRITE (KK, 5) J, K
17 POPMAT (I, I, 'THE ANGLE OUT WAS LESS
18 1 FROM 40 DEG FOR ROW', I3, 'STAGE', I3)
19 CALL FIG (9, AI, BI40, PTOCHO, YPI)
20 IF (ABS (ANGOUT) .LT. 4C.) GO TO 7
21 CALL FIG (9, A3, B340, PTOCHO, YPNA (1))
22 CALL FIG (9, A3, B350, PTOCHO, YPNA (2))
23 CALL FIG (9, A3, B360, PTOCHO, YPNA (3))
24 CALL FIG (9, A3, B365, PTOCHO, YPNA (4))
25 CALL FIG (9, A3, B370, PTOCHO, YPNA (5))
26 CALL FIG (9, A3, B275, PTOCHO, YPNA (6))
27 CALL FIG (9, A3, B180, PTOCHO, YPNA (7))
28 CALL FIG (7, ANG80, YPNA, A2, YPN)
29 GO TO 8
30 WRITE (KK, 5) J, K
31 CALL FIG (9, A3, B340, PTOCHO, YPN)
32 YP = (YPN + (BLADIN / ANGOUT) * 2 * (YPI - YPN)) * (PC / .2) * * (-BLADIN / ANGOUT)
33 IF (ABS (CINCI) .LT. 0.5) GO TO 18
34 IF (PTOCHO.GT. 1.) GO TO 35
35 CALL FIG (7, PCA, PCR, PTOCHO, C75)
36 GO TO 34

```

PGM10901
 PGM10902
 PGM10903
 PGM10904
 PGM10905
 PGM10906
 PGM10907
 PGM10908
 PGM10909
 PGM10910
 PGM10911
 PGM10912
 PGM10913
 PGM10914
 PGM10915
 PGM10916
 PGM10917
 PGM10918
 PGM10919
 PGM10920
 PGM10921
 PGM10922
 PGM10923
 PGM10924
 PGM10925
 PGM10926
 PGM10927
 PGM10928
 PGM10929
 PGM10930
 PGM10931
 PGM10932
 PGM10933
 PGM10934
 PGM10935
 PGM10936

35
34

```
C75=PCB(7)-.4*(PTOCHO-1.)
A275=A2/C75
AB=BLADIN/A275
IF(J.EQ.2)AB=-BLADIN/A275
IF((AB.LT.-.9).OR.(AB.GT.1.))GO TO 25
IF((A275.GT.65.).AND.(AB.GT.0.4))GO TO 9
IF((A275.LT.40.).AND.(AB.GT.0.6))GO TO 11
CALL FIG(9,AB,B830,AB,SI75(1))
CALL FIG(11,AB11,B840,AB,SI75(2))
CALL FIG(11,AB11,B850,AB,SI75(3))
CALL FIG(11,AB11,B855,AB,SI75(4))
CALL FIG(11,AB11,B860,AB,SI75(5))
CALL FIG(11,AB11,B865,AB,SI75(6))
CALL FIG(7,AB,AB,SI75(7))
CALL FIG(7,ANG30,SI75,A275,SI751)
GO TO 12
```

9

```
WRITE(KK,10)J,K
```

10

```
FORMAT(' ','THE ANGLE OUT WAS GREATER THAN 65 DEG AND THE DATA WAS
1 FROM 65 DEG FOR POW',I3,'STAGE',I3)
CALL FIG(11,AB11,B865,AB,SI751)
GO TO 12
```

11

```
WRITE(KK,5)J,K
```

```
CALL FIG(9,AB,B830,AB,SI751)
```

12

```
IF(PTOCHO.GT.0.8)GO TO 13
```

```
CALL FIG(5,A5,B5,PTOCHO,DELTSI)
```

```
GO TO 16
```

13

```
IF(PTOCHO.GT.1.)GO TO 14
```

```
IF((A2.LT.40.).OR.(A2.GT.70.))GO TO 14
```

```
CALL FIG(5,A15,B1540,PTOCHO,SIDA(1))
```

```
CALL FIG(5,A15,B1550,PTOCHO,SIDA(2))
```

```
CALL FIG(5,A15,B1560,PTOCHO,SIDA(3))
```

```
CALL FIG(5,A15,B1570,PTOCHO,SIDA(4))
```

```
CALL FIG(4,ANG47,SIDA,A2,DELTSI)
```

```
GO TO 16
```

```
WRITE(KK,15)J,K
```

14

```
FORMAT(' ','THE P/C RATIO WAS GREATER THAN DATA FOR OFF INCIDENCE
```

15

PGM10937
PGM10938
PGM10939
PGM10940
PGM10941
PGM10942
PGM10943
PGM10944
PGM10945
PGM10946
PGM10947
PGM10948
PGM10949
PGM10950
PGM10951
PGM10952
PGM10953
PGM10954
PGM10955
PGM10956
PGM10957
PGM10958
PGM10959
PGM10960
PGM10961
PGM10962
PGM10963
PGM10964
PGM10965
PGM10966
PGM10967
PGM10968
PGM10969
PGM10970
PGM10971
PGM10972


```

16 1CALCULATIONS, 3 VALUE FOR DELTA INCIDENCE IS TAKEN FOR P/C=1./,
20 FOR POW,I3, STAGE,I3)
  IF(A2.GT.70.)DELTSI=P1570(5)
  IF(A2.LT.70.)DELTSI=P1570(5)
  IF(A2.LT.60.)DELTSI=B1560(5)
  IF(A2.LT.50.)DELTSI=B1550(5)
  IF(A2.LT.40.)DELTSI=B1540(5)
  SI=DELTSI+SI751
  SIR=CINCI/SI
  IF((SIR.GT.1.5).OR.(SIP.LT.-4.))GO TO 17
  CALL FIG(7,YP2A,YP2B,SIP,YPC)
  YP=YP*YPC
  GO TO 18
  YP=YP*10.
  WRITE(KK,29)J,K
  FORMAT(1,1,1, THE DATA LIMITS WERE EXCEEDED IN POW,I3, STAGE,I3)
  A6=ANGIN*PI/180.
  IF(J.EQ.1)A6=-A6
  A4=ANGOUT*PI/180.
  IF(J.EQ.2)A4=-A4
  ANGMEN=ATAN(.5*(TAN(A4)-TAN(A6)))
  CLSC=2.*(TAN(A6)+TAN(A4))*COS(ANGMEN)
  ZETA=CLSC**2*(COS(A4)**2/COS(ANGMEN)**3)
  YS=.0334*(COS(A4)/COS(P1))*ZETA/ASPECB
  IF(KLEAP.EQ.1)B=.0
  IF(KLEAP.EQ.2)B=.37
  IF(KLEAP.EQ.3)B=.47
  YK=B*(TIPCL/CHORD)**.78*7PETA/ASPECB
  IF(REMACH.GT.1.)YP=YP*(1.+60.*(REMACH-1.))**2)
  YPS=(YP+YS)*(PF/2.P5)**(-.2)
  YT=YPS+YK
  CALL FIG(7,TEX,TEY,TET,YDEC)
  YT=YT*YDEC
  RETURN
  YT=0.5
  WRITE(KK,26)J,K

```



```

FORMAT(' ', 'THE INPUT DATA WAS GREATER THAN LIMITS OF PROGRAM & A
1 VALUE OF Y*=.5 WAS ASSIGNED FOR ROW', I2, ' STAGE', I3)
GO TO 24

```

```

END

```

```

FUNCTION REACT2(A,B)

```

```

COMMON PI, GO, PJ, HJ, GAMMA, VX, SOMEGA, DHOS, CP, GAMMA2, GAMMA1, KK

```

```

C COMPUTES REACTION OF ROTOR AT STATION 2

```

```

REACT2=1.+DHOS*GO*HJ/((SOMEGA*A)**2*2.)-(VX/(SOMEGA*A))*TAN(B)

```

```

RETURN

```

```

END

```

```

SUBROUTINE SECMOD(SMTANG, IC, SM, AREAC2)

```

```

COMMON PI, GO, PJ, HJ, GAMMA, VX, SOMEGA, DHOS, CP, GAMMA2, GAMMA1, KK

```

```

REAL A(10), B(10), C(10), N, D(8), E(8)

```

```

DATA A/10., 20., 30., 40., 50., 60., 80., 100., 120., 140./

```

```

DATA B/1.9417, 1.8561, 1.7705, 1.6849, 1.5992, 1.5136, 1.3424, 1.1712, 1.,
1.8289/

```

```

DATA C/1049.5, 1024.2, 991.52, 946.67, 896.36, 827.879, 661.212, 458.788,
1270.30, 66.67/

```

```

DATA D/0., 20., 40., 60., 80., 100., 120., 140./

```

```

DATA E/.02131, .04, .06423, .09525, .13097, .1695, .21305, .25692/

```

```

I=10

```

```

CALL FIG(I, A, B, SMTANG, N)

```

```

CALL FIG(I, A, C, SMTANG, X)

```

```

SM=(1./X)*(10.*IC)**N*2.35

```

```

CALL FIG(8, D, E, SMTANG, AREAC2)

```

```

RETURN

```

```

END

```

```

SUBROUTINE STOCRA(ANGLIN, ANGLOR, PCORAT)

```

```

COMMON PI, GO, PJ, HJ, GAMMA, VX, SOMEGA, DHOS, CP, GAMMA2, GAMMA1, KK

```

```

DIMENSION A(0(5), B(0(5), B50(4), A(8), B(8), C(0), D(8), E(7), F(7)

```

```
1), G(6), H(6), X(5), Y(5)

```

```

DATA B40/40., 50., 60., 67.817, 70./

```

```

DATA B50/50., 60., 67.817, 70./

```

```

DATA A/0., 10., 20., 30., 40., 50., 60., 70./

```

```

DATA B/.72759, .71675, .68719, .65961, .62611, .58966, .55517, .53153/

```

```

DATA C/0., 10., 20., 30., 40., 50., 60., 70./

```

```

PGM11009
PGM11010
PGM11011
PGM11012
PGM11013
PGM11014
PGM11015
PGM11016
PGM11017
PGM11018
PGM11019
PGM11020
PGM11021
PGM11022
PGM11023
PGM11024
PGM11025
PGM11026
PGM11027
PGM11028
PGM11029
PGM11030
PGM11031
PGM11032
PGM11033
PGM11034
PGM11035
PGM11036
PGM11037
PGM11038
PGM11039
PGM11040
PGM11041
PGM11042
PGM11043
PGM11044

```



```

DATA D/.75517,,74138,,71478,,68325,,64375,,61527,,57980,,55419/
DATA E/0.,10.,20.,30.,40.,50.,60./
DATA F/.83399,,81626,,79458,,76108,,71675,,68719,,64581/
DATA G/0.,10.,20.,30.,40.,50./
DATA H/.90985,,88424,,85074,,81626,,76995,,74138/
DATA X/0.,10.,20.,30.,40./
DATA Y/.96601,,93547,,89015,,84778,,79571/
IF(ANGLOT.GT.70.)GO TO 3
IF((ANGLOT.GF.67.817).AND.(ANGLIN.LE.70.))GO TO 4
IF((ANGLOT.GF.60.).AND.(ANGLIN.LE.60.))GO TO 5
IF((ANGLOT.GF.50.).AND.(ANGLIN.LE.50.))GO TO 6
IF((ANGLOT.GF.40.).AND.(ANGLIN.LE.40.))GO TO 7
CALL FIG(8,A,B,ANGLIN,PCPAT)
GO TO 2
3 CALL FIG(8,C,D,ANGLIN,A681)
4 CALL FIG(8,A,B,ANGLIN,A682)
PCPAT=A681-(A681-A682)*(70.-ANGLOT)/2.183
GO TO 2
5 CALL FIG(7,F,F,ANGLIN,A601)
CALL FIG(8,C,D,ANGLIN,A602)
PCPAT=A601-(A601-A602)*(70.-ANGLOT)/10.
GO TO 2
6 CALL FIG(6,G,H,ANGLIN,A50(1))
CALL FIG(7,F,F,ANGLIN,A50(2))
CALL FIG(8,C,D,ANGLIN,A50(3))
CALL FIG(8,A,B,ANGLIN,A50(4))
CALL FIG(4,B50,A50,ANGLOT,PCPAT)
GO TO 2
7 CALL FIG(5,X,Y,ANGLIN,A40(1))
CALL FIG(6,S,H,ANGLIN,A40(2))
CALL FIG(7,F,F,ANGLIN,A40(3))
CALL FIG(8,C,D,ANGLIN,A40(4))
CALL FIG(8,A,B,ANGLIN,A40(5))
CALL FIG(5,B40,A40,ANGLOT,PCPAT)
2 RETURN
END

```

```

PGM11045
PGM11046
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PGM11079
PGM11080

```


SUBROUTINE BLADE(A,B,C,J)
 C CALCULATES BLADE OF GAS ANGLES FOR ROTOR AND STATORS
 DIMENSION D(6),F(6)
 DATA D/24.34,40.,50.,60.,70.,80./
 DATA F/30.,47.717,53.962,62.453,70.943,78.868/
 I=6

IF(J.EQ.2)GO TO 3
 IF(B.LT.1.)GO TO 1
 C=?

GO TO 2

1 CALL FIG(I,D,F,A,C)
 IF(B.LT.0.5)GO TO 2
 C=C-((B-.5)/.5)*(C-A)
 2 RETURN

3 IF(B.LT.1.)GO TO 5
 A=C

GO TO 2

5 CALL FIG(F,P,D,C,A)
 IF(P.LT.0.5)GO TO 2
 A=A+((B-.5)/.5)*(C-A)
 GO TO 2

END

SUBROUTINE FIG(I,D,F,X,Y)
 DIMENSION D(I),F(I),DD(100),A(4),B(4)
 IF(((X.GT.D(I)).AND.(D(1).LT.D(I))).OR.((X.LT.D(1)).AND.(D(I).GT.D(1(1)).OR.((X.LT.D(I)).AND.(D(1).GT.D(I))).OR.((2X.GT.D(1)).AND.(D(1).GT.D(I))))GO TO 31
 IF(D(1).GT.D(I))GO TO 9
 IF(I.EQ.4)GO TO 2

N=I-1

J=2

IF(X.GE.D(N))GO TO 1
 IF(X.LE.D(J))GO TO 2
 L=3
 IF(X.LT.D(L))GO TO 3
 L=L+1

PGM11081
 PGM11082
 PGM11083
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 PGM11085
 PGM11086
 PGM11087
 PGM11088
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 PGM11092
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 PGM11100
 PGM11101
 PGM11102
 PGM11103
 PGM11104
 PGM11105
 PGM11106
 PGM11107
 PGM11108
 PGM11109
 PGM11110
 PGM11111
 PGM11112
 PGM11113
 PGM11114
 PGM11115
 PGM11116

| | | |
|----|--|-----------|
| 3 | GO TO 4 | PGM111117 |
| | L=L-2 | PGM111118 |
| | DO 8 K=1,4 | PGM111119 |
| | A(K)=D(L) | PGM111120 |
| | B(K)=F(L) | PGM111121 |
| 8 | L=L+1 | PGM111122 |
| | GO TO 7 | PGM111123 |
| 1 | IJ=I-4 | PGM111124 |
| | DO 5 II=1,4 | PGM111125 |
| | A(II)=D(IJ) | PGM111126 |
| | B(II)=F(IJ) | PGM111127 |
| 5 | IJ=IJ+1 | PGM111128 |
| | GO TO 7 | PGM111129 |
| 2 | DO 6 II=1,4 | PGM111130 |
| | A(II)=D(II) | PGM111131 |
| 6 | B(II)=F(II) | PGM111132 |
| 7 | CALL BK(A,B,X,Y) | PGM111133 |
| 32 | RETURN | PGM111134 |
| 9 | DD(1)=D(I) | PGM111135 |
| | FF(1)=F(I) | PGM111136 |
| | J=I | PGM111137 |
| | DO 15 M=2,I | PGM111138 |
| | J=J-1 | PGM111139 |
| 15 | DD(M)=D(J) | PGM111140 |
| | FF(M)=F(J) | PGM111141 |
| | DO 16 M=1,I | PGM111142 |
| | D(M)=DD(M) | PGM111143 |
| 16 | F(M)=FF(M) | PGM111144 |
| | GO TO 17 | PGM111145 |
| 31 | IF((X.GT.D(I)).AND.(D(1).LT.D(I)).OR.((X.LT.D(I)).AND.(D(1).GT.D(I))))Y=F(I) | PGM111146 |
| | IF(Y.NE.F(I))Y=F(1) | PGM111147 |
| | GO TO 32 | PGM111148 |
| | END | PGM111149 |
| | SUBROUTINE BK(X,Y,XAEG,YOUT) | PGM111150 |
| | DIMENSION X(4),Y(4) | PGM111151 |
| | | PGM111152 |


```

99      YOUT=C.0
20      XSQ=XARG**2
10      XCU=XSQ*XARG
      DO 10 K=1,4
      PI1=1.
      PI2=1.
      SUM1=0.0
      SUM2=0.0
      DO 20 J=1,4
      IF (J.EO.K) GO TO 20
      PI1=PI1*(X(K)-X(J))
      SUM1=SUM1+X(J)
      PI2=PI2*X(J)
      DO 99 I=2,4
      IF (I.EO.K) GO TO 99
      IF (I.LE.J) GO TO 99
      SUM2=SUM2+X(J)*X(I)
      CONTINUE
      CONTINUE
      YOUT=YOUT+1./PI1*(XCU-SUM1*XSQ+SUM2*XARG-PI2)*Y(K)
      IF ((Y(1).GE.Y(2)).AND.(Y(2).GE.Y(3)).AND.(Y(3).GE.Y(4)).AND.(YARG.
11E.X(2)).AND.((YOUT.GT.Y(1)).OR.(YOUT.LT.Y(2)))) YOUT=Y(1)-(Y(1)-Y(
12))*XARG-X(1))/(X(2)-X(1))
      IF ((Y(1).GE.Y(2)).AND.(Y(2).GE.Y(3)).AND.(Y(3).GE.Y(4)).AND.(XARG.
11E.X(3)).AND.(XARG.GT.X(2)).AND.((YOUT.GT.Y(2)).OR.(YOUT.LT.Y(3))))
2) YOUT=Y(2)-(Y(2)-Y(3))*XARG-X(2))/(X(3)-X(2))
      IF ((Y(1).GE.Y(2)).AND.(Y(2).GE.Y(3)).AND.(Y(3).GE.Y(4)).AND.(XARG.
11E.X(4)).AND.(XARG.GT.X(3)).AND.((YOUT.GT.Y(3)).OR.(YOUT.LT.Y(4))))
2) YOUT=Y(3)-(Y(3)-Y(4))*XARG-X(3))/(X(4)-X(3))
      IF ((Y(1).LE.Y(2)).AND.(Y(2).LE.Y(3)).AND.(Y(3).LE.Y(4)).AND.(XARG.
11E.X(2)).AND.((YOUT.LT.Y(1)).OR.(YOUT.GT.Y(2)))) YOUT=Y(1)-(Y(1)-Y(
12))*XARG-X(1))/(X(2)-X(1))
      IF ((Y(1).LE.Y(2)).AND.(Y(2).LE.Y(3)).AND.(Y(3).LE.Y(4)).AND.(XARG.
11E.X(3)).AND.(XARG.GT.X(2)).AND.((YOUT.LT.Y(2)).OR.(YOUT.GT.Y(3))))
2) YOUT=Y(2)-(Y(2)-Y(3))*XARG-X(2))/(X(3)-X(2))
      IF ((Y(1).LE.Y(2)).AND.(Y(2).LE.Y(3)).AND.(Y(3).LE.Y(4)).AND.(XARG.

```



```

1LE,X(4)).AND.(XARG.GT.X(3)).AND.((YOUT.LT.Y(3)).OR.(YOUT.GT.Y(4)))
2)YOUT=Y(3)-(Y(3)-Y(4))*(XARG-X(3))/(X(4)-X(3))
RETURN
END
SUBROUTINE TESTP(I,N)
C DETERMINES IF AN INTEGER IS A PRIME NO. GIVES AN OUTPUT OF 1 IF PRIME 0 IF NOT
INTEGER TEST
I=0
TEST=2
IF(N/TEST*TEST.NE.N)GO TO 8
GO TO 11
TEST=1
TEST=TEST+2
IF((N/TEST*TEST.EQ.N).AND.(TEST.NE.N))GO TO 11
IF(TEST.EQ.N)GO TO 10
GO TO 3
I=1
RETURN
END
FUNCTION REACTI(A,B)
C COMPUTES REACTION OF ROTOR AT STATION 3
COMMON PI,GO,PJ,HJ,GAMMA,VX,SOMEGA,DHOS,CP,GAMMA2,GAMMA1,KK
REACTI=1.-DHOS*GO*HJ/((SOMEGA*4)*2*2.)-(VX/(SOMEGA*A))*TAN(B)
RETURN
END
SUBROUTINE TPD2(A,B,C,D,E,F,G,H)
C DETERMINES TEMPERATURE,PRESSURE, AND DENSITY AT STATION 2
COMMON PT,GO,PJ,HJ,GAMMA,VX,SOMEGA,DHOS,CP,GAMMA2,GAMMA1,KK
V=VX/COS(G)
IF(V.LT.0.)V=-V
C=A-V**2/(CP*GO*HJ*.7.)
IF(C.LT.0.5*A)C=0.5*A
F=E*(C/B)**GAMMA2
D=F*(A/C)**GAMMA1
H=F/(C*PJ)
RETURN

```



```

END
SUBROUTINE TP1(A,B,C,D,E,F)
C COMPUTES TEMPERATURE,PRESSURE, AND DENSITY AT STATION 1 AND 3
COMMON PI,GO,RJ,HJ,GAMMA,VX,SOMEGA,DHOS,CP,GAMMA2,GAMMA1,KK
V=VX/COS(C)
IF(V.LT.C.C)V=-V
B=A-V**2/(CP*GO*HJ*2.)
IF(B.LT.0.5*A)B=0.5*A
E=D*(B/A)**GAMMA1
F=E/(RJ*B)
RETURN
END
FUNCTION ANGLE2(A,B)
C DETERMINES ANGLE ALPHA 2
COMMON PI,GO,RJ,HJ,GAMMA,VX,SOMEGA,DHOS,CP,GAMMA2,GAMMA1,KK
ANGLE2=ATAN(DHOS*GO*HJ/(VX*SOMEGA*A)+TAN(R))
RETURN
END
FUNCTION ANGLE(A,E)
C DETERMINES RELATIVE ANGLE RETA
COMMON PI,GO,RJ,HJ,GAMMA,VX,SOMEGA,DHOS,CP,GAMMA2,GAMMA1,KK
ANGLEB=ATAN(TAN(A)-SOMEGA*R/VX)
RETURN
END
SUBROUTINE DEN2(A,B,C,D)
COMMON PI,GO,RJ,HJ,GAMMA,VX,SOMEGA,DHOS,CP,GAMMA2,GAMMA1,KK
COMPUTES INTEGRAL OF DENSITY TIMES AREA TO GET AXIAL VELOCITY BY SIMPSON'S RULE
REAL A(3),B(3)
DENSIT=(1./3.)*((A(3)-A(1))/2.)*(A(1)*R(1)+4.*A(2)*B(2)+A(3)*B(3))
D=C/(2.*PI*DENSIT)
RETURN
END
FUNCTION ANGLE(L,B,C)
C DETERMINES ANGLE AT ANOTHER LOCATION BY FREE VELOCITY DEFINITION OF F*V(THETA)=K
ANGLE=ATAN(A*TAN(B)*TAN(E)/C)
RETURN

```


END

\$ENTRY

9000 HP SINGLE STAGE GAS TURBINE ENGINE

00 1 60 111.333.2715900. 119.42 .2 .02 .015 1767.8 312.48

1.8 .62161.418.1 20000. 50 0.289 29000000. 20000. 93000.

.9168.08333 1.680130000. .000 .292 0 0 1.25 .1

0.020.040.060.080.100.120.140.160.170.200.220.240.260.280.300.320.340.360.380.40

0.420.440.460.480.500.520.540.560.580.600.620.640.660.680.700.720.740.760.780.80

0.820.840.860.880.900.920.940.960.981.001.021.041.061.081.101.121.141.161.181.20

0.01811 0.03596 0.05388 0.07171 0.08945 0.10706 0.12448 0.14154 0.15885 0.17562

0.19226 0.20849 0.22446 0.24049 0.25612 0.27135 0.28590 0.30026 0.31435 0.32763

0.34118 0.35386 0.36627 0.37821 0.38948 0.40061 0.41108 0.42114 0.43067 0.43979

0.44838 0.45669 0.46421 0.47159 0.47796 0.48433 0.49010 0.49560 0.50030 0.50459

0.50815 0.51170 0.51479 0.51734 0.51948 0.52123 0.52257 0.52337 0.52391 0.52418

0.52391 0.52337 0.52257 0.52136 0.51989 0.51801 0.51599 0.51358 0.51076 0.50795

0.99973 0.99893 0.99760 0.99574 0.99335 0.99040 0.98700 0.98317 0.97870 0.97380

0.96835 0.96269 0.95630 0.94950 0.94200 0.93440 0.92650 0.91800 0.90920 0.90030

0.89080 0.88100 0.87060 0.86000 0.84950 0.83860 0.82720 0.81570 0.80400 0.79220

0.78020 0.76780 0.75550 0.74270 0.73070 0.71800 0.70520 0.69140 0.67910 0.66650

0.65500 0.64080 0.62760 0.61410 0.60250 0.59000 0.57550 0.56500 0.55100 0.54000

0.52800 0.51700 0.50400 0.49100 0.48000 0.46820 0.45650 0.44530 0.43330 0.42250

0.96151 1.91184 2.86589 3.82740 4.77028 5.72284 6.67004 7.61008 8.56787 9.50329

10.4528711.3666812.3132813.2673414.2288415.1456416.0475016.9941117.9034418.82924

19.7519220.6314421.5407722.4501023.3370924.2762525.1632126.0501926.9520727.80022

28.6589229.5459030.4105231.2975032.0950232.9447233.8019034.6516035.4789436.20883

37.0516437.9396138.7659539.6082240.3982841.1436542.0306142.7088943.4169344.16234

44.9822245.6903146.3611347.1735747.8518448.5072049.3425650.0282750.7438251.39220

400. 600. 800. 1000. 1200. 1400. 1600. 1800. 2000. 2200. 2400.

100. 135. 166. 192. 218. 242. 264. 284. 302. 320. 338.

13673 HP SINGLE STAGE AIRCRAFT GAS TURBINE ENGINE

10 2 60 111.333.274 13673. 100. .2 .02 .015 2960. 38.2

2.8 1.034 2.0 0.0 13856. 50 2.289 29000000. 45000. 93000.

0.9 .1 6. 130000. -.5 -.25 0 2 1.25 .1

/*EOJ *****

\$*
\$JOB
\$*

VICTOR ENDO MAY 9, 1977
SIPP, TIME=30
VICTOR ENDO MAY 9, 1977

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COMMON PI, GC, PJ, HJ, GAMMA, VX, SCOMEGA, DHOS, CP, GAMMA2, GAMMA1, KK
COMMON/APEA1/TET, TC, TIPCLA
DIMENSION SPRES(3,3,10), RADIUS(3,3,10), ALPHA(3,3,10), BETA(3,3,10)
DIMENSION DEN(3), PAD(3), TTEMP(3,10), TPRES(3,10), STEMP(3,3,10)
DIMENSION REACT(3,10), PHO(3,3,10), REALMS(3,2,10), RELMP(3,2,10), TTE
1MPR(3,2,12), VAXIAL(3,12)
DIMENSION RBLADE(3,2,10), SBLADE(3,2,10), CHOPDR(3,2,10), SIGMAB(2,10
1), SIGMAC(12), BLADEH(2,12), PTOCHO(2,12), ASPECR(2,12), TEM(50), V(50)
DIMENSION SPEED(5), WDOTP(7), OMEGAB(5), POIS(7), RMOC(3,10), BETAC(3,1
10), ALPHAC(3,10), TOTC(3,10), TSC(3,10), POC(3,10), PSC(3,10), PPOC(3,10
3), VEL(3,10), PVEL(3,10), RMAC(100), WTAP(100), PSPT(100), VELTOT(100), P
101PO2(7), ENTMJ(7), ENSMJ(7), PTOC(3,10), A(6), B(6)
DIMENSION WTAP11(70), WTAP12(70), RMAC11(70), RMAC12(70), VROT11(70), V
1EOT12(70), PSPT11(70), PSPT12(70)
DATA SPEED/.4,.6,.8,.1,.1,1/
DATA WDOTP/.6,.8,.9,.95,.99,1,.1,1,1/
KK=6
LL=5
LKL=0
PEAD(LL,779) CP, GO, HJ, RJ, VX, TOI, POT, E, W, KKK
FORMAT(F5.3, F10.4, P8.3, F10.4, F10.1, F10.1, F10.3, F5.3, F10.3, I2)
PEAD(LL,780) GAMMA1, PI, SCOMEGA, TET, TC, TIPCLA, GAMMA, KLM1, KLM, KKTMA
FORMAT(F10.5, F10.8, F10.3, 4F5.3, I2, I3, I2)
READ(LL,782) ((RADIUS(I,J,K), I=1,3), J=1,3), K=1, KKK)
FORMAT(8F10.4)
READ(LL,781) ((TTEMP(J,K), J=1,3), K=1, KKK)
FORMAT(8F10.1)
READ(LL,781) ((STEMP(2,J,K), J=1,3), K=1, KKK)
READ(LL,781) ((TPPRES(J,K), J=1,3), K=1, KKK)
READ(LL,781) ((SPRES(I,J,K), I=1,3), K=1, KKK)
READ(LL,782) ((PHO(2,J,K), J=1,3), K=1, KKK)
READ(LL,782) ((ALPHA(2,J,K), J=1,3), K=1, KKK)
READ(LL,782) ((BETA(2,J,K), J=1,3), K=1, KKK)

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779

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782

781

COMPUTER PROGRAM #2

PGM200001
PGM200002
PGM200003
PGM200004
PGM200005
PGM200006
PGM200007
PGM200008
PGM200009
PGM200010
PGM200011
PGM200012
PGM200013
PGM200014
PGM200015
PGM200016
PGM200017
PGM200018
PGM200019
PGM200020
PGM200021
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PGM200027
PGM200028
PGM200029
PGM200030
PGM200031
PGM200032
PGM200033
PGM200034
PGM200035
PGM200036


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398 READ(LL,782) ((PEALMS(2,J,K),J=1,2),K=1,KKK)
    READ(LL,782) ((RELMR(2,J,K),J=1,2),K=1,KKK)
    READ(LL,782) ((SBLADE(2,J,K),J=1,2),K=1,KKK)
    READ(LL,782) ((RBLADE(2,J,K),J=1,2),K=1,KKK)
    READ(LL,782) ((CHORDP(2,J,K),J=1,2),K=1,KKK)
    READ(LL,782) ((ASPECE(J,K),J=1,2),K=1,KKK)
    READ(LL,782) ((PTOCHO(J,K),J=1,2),K=1,KKK)
    LKL=LKL+1
    IF(LKL.GT.1)GO TO 299
    READ(LL,200) (RMAC(I),I=1,60)
    READ(LL,201) (WTAP(I),I=1,60)
    READ(LL,201) (PSPT(I),I=1,60)
    READ(LL,201) (VELTOT(I),I=1,60)
    FORMAT(10F8.5)
200 FORMAT(20F4.2)
    READ(LL,227) (TFM(I),I=1,KTM)
    READ(LL,227) (V(I),I=1,KTM)
    FORMAT(20F5.C)
227 KLM=60
299 KLM2=KLM-KLM1
    DO 230 I=1,KLM1
    II=I+KLM1-1
    RMAC11(I)=RMAC(I)
    WTAP11(I)=WTAP(I)
    PSPT11(I)=PSPT(I)
    VEOT11(I)=VELTOT(I)
    IF(I.GT.KLM2)GO TO 230
    RMAC12(I)=RMAC(II)
    WTAP12(I)=WTAP(II)
    PSPT12(I)=PSPT(II)
    VEOT12(I)=VELTOT(II)
    CONTINUE
230 DO 350 MK=1,5
    OMEGAB(MK)=SOMEGA*SPEED(MK)
    DO 350 MJ=1,7
    K=1

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PGM200037
PGM200038
PGM200039
PGM200040
PGM200041
PGM200042
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PGM200044
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PGM200064
PGM200065
PGM200066
PGM200067
PGM200068
PGM200069
PGM200070
PGM200071
PGM200072

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BETAC(1,K)=0.0
ALPHAC(1,K)=0.0
TOTC(1,K)=TOI
TSC(1,K)=TOI-VX**2/(2.*CP*HJ*GO)
POC(1,K)=POI
PSC(1,K)=POC(1,K)/(TOTC(1,K)/TSC(1,K))*GAMMA1
VAXIAL(1,K)=VX
RHOC(1,K)=PSC(1,K)/(TSC(1,K)*RJ)
POI=PCC(1,K)
TOI=TOI
GANG1=0.0

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308
306

```

J=1
L=J+1
JJ=0
U=OMEGAB(MK)*RADIUS(2,L,K)
PHO2=PHO(2,L,K)
RMAC2=.2*REALMS(2,2,K)
IF(J.EQ.2)PMAC2=.2*RELMR(2,2,K)
BLADEF2=SBLADE(2,2,K)
IF(J.EQ.2)BLADEF2=RBLADE(2,2,K)
ANNUA=PI*(RADIUS(3,L,K)**2-RADIUS(1,L,K)**2)
JJ=JJ+1

```

300

```

IF(J.EQ.2)BLADE2=-BLADE2
CALL BLADE(GANG2,RMAC2,BLADE2,2)
IF(J.EQ.2)GANG2=-GANG2
IF(J.EQ.2)BLADE2=-BLADE2
CALL FIG(KLM,RMAC,VELITOT,PMAC2,VFOTOT2)
V2=SQRT(TOI)*VFOTOT2
CALL FIG(KLM,RMAC,PSIT,PMAC2,PSPT2)
T2=TOI*PSPI2**(1./GAMMA1)
BLADE1=SBLADE(2,1,K)
IF(J.EQ.2)BLADE1=PBPIADE(2,1,K)
IF(J.EQ.1)KLEAR=1
IF(J.EQ.2)KLEFAP=3
CALL PLOSS(GANG1,GANG2,PHO2,T2,BLADE1,PTCCHC(J,K),ASPECB(J,K),PMAC

```

12,V2,CHORDS(2,J,K),J,K,KLEAF,YT,KTM,TFM,V)

PGM20073
PGM20074
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PGM20080
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PGM20093
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PGM20096
PGM20097
PGM20098
PGM20099
PGM20100
PGM20101
PGM20102
PGM20103
PGM20104
PGM20105
PGM20106
PGM20107
PGM20108


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316 P02=P01/(YT*(1.-PSP*2)+1.)
    Q2=WDOTP(MJ)*W*SQRT(T01)/(ANNULA*COS(GANG2*PI/180.))*PC2)
    IF(O2.GT.WTAP(KLM1))O2=WTAP(KLM1)*.99
    IF(RMAC2.GT.PMAC(KLM1))GO TO 316
    CALL FIG(KLM1,WTAP11,RMAC11,O2,RMAC2P)
    CALL FIG(KLM1,WTAP11,PSPT11,O2,PSPT2)
    CALL FIG(KLM1,WTAP11,VEOT11,O2,VEOT2)
    GO TO 317
CONTINUE
    CALL FIG(KLM2,WTAP12,RMAC12,O2,RMAC2P)
    CALL FIG(KLM2,WTAP12,PSPT12,O2,PSPT2)
    CALL FIG(KLM2,WTAP12,VEOT12,O2,VEOT2)
317 P02=P01/(YT*(1.-PSPT2)+1.)
    P2=P02*PSPT2
    IF(PSPT2.LT.0.0)GO TO 399
    T2=T01*PSPT2*(1./GAMMA1)
    PHO2=P2/(T2*RJ)
    V2=SQRT(T01)*VEOT2
    IF(ABS(RMAC2P/RMAC2-1.).LT.5)GO TO 302
    IF((RMAC2P.GT.0.9).AND.(JJ.GT.3))GO TO 304
    IF(JJ.EQ.10)GO TO 302
    IF(ABS(RMAC2P/RMAC2-1.).LT.0.03)GO TO 318
    RMAC2=PMAC2P
    GO TO 300
    RMAC2=PMAC2+(RMAC2P-PMAC2)/2.
    GO TO 300
    Q2PRIM=1.0001*Q2
    IF((RMAC2P.GT.1.0))GO TO 330
    CALL FIG(KLM1,WTAP11,PSPT11,O2PRIM,PSPTR)
    GO TO 331
    CALL FIG(KLM2,WTAP12,PSPT12,O2PRIM,PSPTR)
    P02P01=1./(YT*(1.-PSPTR)+1.)
    WPRIME=O2PRIM*P02P01*ANNULA*COS(GANG2*PI/180.))*P01/SQRT(T01)
    IF(WPRIMEF.LT.W*WDOTI(MJ))GO TO 305
    GO TO 315
    WRITE(KK,310)J,K,STFED(MK),DOI/144.
305

```

PGM20109
 PGM20110
 PGM20111
 PGM20112
 PGM20113
 PGM20114
 PGM20115
 PGM20116
 PGM20117
 PGM20118
 PGM20119
 PGM20120
 PGM20121
 PGM20122
 PGM20123
 PGM20124
 PGM20125
 PGM20126
 PGM20127
 PGM20128
 PGM20129
 PGM20130
 PGM20131
 PGM20132
 PGM20133
 PGM20134
 PGM20135
 PGM20136
 PGM20137
 PGM20138
 PGM20139
 PGM20140
 PGM20141
 PGM20142
 PGM20143
 PGM20144


```

310  FORMAT(' ',IRON,I2,' IN STAGE',I3,' FOR N/SORT(TO1) = ',P4.1,' ZO
      1R PO1 = ',P8.1,' CHECKED')
      CALL FIG(KLM2,WTAF12,VEOT12,Q2,V3IOF2)
      CALL FIG(KLM2,WTAF12,PSPT12,Q2,PSPT2)
      V2=SORT(TO1)*VEOT2
      PO2=PO1/(VT*(1.-PSPT2)+1.)
      T2=TO1*PSPT2*(1./GAMMA1)
      P2=PO2*PSPT2
      RHO2=P2/(T2*PJ)
      GANG2=ARCCOS(W*WDOCP(MJ)*SORT(TO1)/(PO2*Q2*ANNULM))*180./PI
      IF(J.EQ.2) GO TO 307
      J=J+1
      ALPHAC(2,K)=GANG2
      BETAC(2,K)=(ATAN(TAN(GANG2*PI/180.)-U/(V2*CCS(GANG2*PI/180.))))*18
      0./PI
      RHOC(2,K)=PHC2
      VWSQ=(V2*CCS(GANG2*PI/180.))*2*(1.+(TAN(BETAC(2,K)*PI/180.))*2)
      VAXIAL(2,K)=V2*CCS(GANG2*PI/180.)
      TOTC(2,K)=TO1
      TSC(2,K)=TC1-V2**2/(2.*CP*HJ*GO)
      TO1=T2+VWSQ/(2.*CP*HJ*GO)
      PSC(2,K)=P2
      POC(2,K)=PO2
      PVTOTP=SORT(VWSQ)/SORT(TO1)
      CALL FIG(KLM,V3IOT,PSPT,SVTOTR,PSPT2)
      PO1=P2/PSPT2
      GANG1=BETAC(2,K)
      GO TO 306
      TSC(3,K)=TC1-V2**2/(2.*GO*HJ*CP)
      PSC(3,K)=PO2*PSPT2
      RHOC(3,K)=PHC2
      BETAC(3,K)=GANG2
      OMEGAB(MK)*RADIUS(2,2,K)
      VAXIAL(3,K)=V2*CCS(GANG2*PI/180.)
      ALPHAC(2,K)=(ATAN(TAN(GANG2*PI/180.)+U/(V2*CCS(GANG2*PI/180.))))*1
      80./PI
302  PGM20145
      PGM20146
      PGM20147
      PGM20148
      PGM20149
      PGM20150
      PGM20151
      PGM20152
      PGM20153
      PGM20154
      PGM20155
      PGM20156
      PGM20157
      PGM20158
      PGM20159
      PGM20160
      PGM20161
      PGM20162
      PGM20163
      PGM20164
      PGM20165
      PGM20166
      PGM20167
      PGM20168
      PGM20169
      PGM20170
      PGM20171
      PGM20172
      PGM20173
      PGM20174
      PGM20175
      PGM20176
      PGM20177
      PGM20178
      PGM20179
      PGM20180
307  36

```



```

309 VELSO=(V2* $\cos$ (GANG2*PI/180.))*2*(1.+ $\tan$ (ALPHAC(3,K)*PI/180.))*2)
    TOTC(3,K)=TSC(3,K)+VELSO/(2.*GO*4J*CP)
    POC(3,K)=PSC(3,K)*(TOTC(3,K)/TSC(3,K))*GAMMA1
    IF(K.EQ.KKK)GO TO 309
    L=K
    K=K+1
    ALPHAC(1,K)=ALPHAC(3,L)
    BETAC(1,K)=BETAC(3,L)
    TOTC(1,K)=TOTC(3,L)
    TSC(1,K)=TSC(3,L)
    POC(1,K)=POC(3,L)
    PSC(1,K)=PSC(3,L)
    VAXIAL(1,K)=VAXIAL(3,L)
    RHOC(1,K)=RHOC(3,L)
    GANG1=ALPHAC(1,K)
    TOT1=TOTC(1,K)
    PO1=POC(1,K)
    GO TO 308
    PO1PO2(MJ)=PO1/POC(3,KKK)
    ENTMJ(MJ)=(TOTC(1,1)-TOTC(3,KKK))/(TOTC(1,1)*(1.-(POC(3,KKK)/POC(1,1)))+(1./GAMMA1))
    ENSMJ(MJ)=(TOTC(1,1)-TOTC(3,KKK))/(TOTC(1,1)*(1.-(PSC(3,KKK)/POC(1,1)))+(1./GAMMA1))
    WRITE(KK,311)SPEED(MK),PO1PO2(MJ)
    FORMAT('0',FOR DESIGN POINT CALCULATIONS AT MEAN RADIUS A LIST OF
    1 CYCLE CALCULATIONS WILL BE COMPARED WITH PERFORMANCE',',CALCUL
    2 TIONS OF IMPROVED FINSLEY MATHIESON METHOD 1ST COLUMN IS CYCLES CAL
    3 CULATIONS AT DESIGN POINT',',FOR N/SORT(TO1)=',F5.2,', PO1/PO2=',
    3,F5.2)
    WRITE(KK,345)
    345 FORMAT('0',STAGE VX STATION 1 VX STATION 2 VX STATION 3')
    WRITE(KK,344)(K1,(VAXIAL(J1,K1),J1=1,3),K1=1,KK)
    344 FORMAT('0',I4,3F13.0)
    WRITE(KK,312)
    312 FORMAT('0',STATION STAGE TOTAL TEMPERATURE STATIC TEMP(R) TOT
    1 AL PRES(PSI) STATIC PRES(PSI) DENSITY(IB/FT**3) ALPHA REFA(')

```


DATA B180/.07149,.06468,.0594,.05882,.06128,.06553,.07285,.0834,.0
 19532/
 DATA B275/.06979,.05872,.05021,.04766,.04817,.05140,.05651,.06383,
 1.074/
 DATA B370/.06894,.05532,.04511,.03966,.03830,.04,.0434,.04936,.057
 187/
 DATA B365/.06809,.05498,.04255,.03574,.03098,.02928,.03115,.03643,
 1.04494/
 DATA B360/.06639,.05362,.04068,.03149,.02689,.02417,.02451,.02809,
 1.03404/
 DATA B350/.06519,.05055,.03949,.0303,.02502,.02264,.02128,.02077,.
 102247/
 DATA B340/.06468,.0497,.03745,.02911,.02383,.02043,.01855,.01804,.
 101872/
 DATA A1/.3,.4,.5,.6,.7,.8,.9,1,1.1/
 DATA B170/.16113,.14186,.13640,.13981,.14868,.16192,.17596,.19096,
 1.2077/
 DATA B165/.15613,.13026,.11967,.11628,.12089,.12907,.13981,.15192,
 1.1668/
 DATA B160/.14334,.11793,.10571,.10196,.10401,.10827,.11475,.12361,
 1.1328/
 DATA B155/.14743,.11509,.09787,.08747,.08491,.08730,.09378,.10315,
 1.1166/
 DATA B150/.14408,.11168,.09122,.07843,.0731,.07502,.0798,.08791,.0
 1987/
 DATA B140/.13981,.10656,.0861,.07315,.06591,.0653,.0682,.07398,.08
 1/
 DATA A5/.4,.5,.6,.7,.8/
 DATA B5/8.0576,6.8921,4.6043,1.4532,-2.3022/
 DATA A15/.8,.85,.9,.95,1.1/
 DATA B1540/-2.3022,-3.3813,-4.6763,-5.5396,-6.4029/
 DATA B1550/-2.3044,-4.4604,-6.6197,-8.9928,-11.3669/
 DATA B1560/-2.3066,-4.7626,-8.0432,-11.3669,-14.777/
 DATA B1570/-2.3088,-5.3237,-8.6403,-14.1726,-19.1367/
 DATA A7/-.9,-.6,-.4,-.2,0,.2,.4/
 DATA B7/11.218,25.671,33.437,40.988,44.223,43.576,40.34/

PGM20253
 PGM20254
 PGM20255
 PGM20256
 PGM20257
 PGM20258
 PGM20259
 PGM20260
 PGM20261
 PGM20262
 PGM20263
 PGM20264
 PGM20265
 PGM20266
 PGM20267
 PGM20268
 PGM20269
 PGM20270
 PGM20271
 PGM20272
 PGM20273
 PGM20274
 PGM20275
 PGM20276
 PGM20277
 PGM20278
 PGM20279
 PGM20280
 PGM20281
 PGM20282
 PGM20283
 PGM20284
 PGM20285
 PGM20286
 PGM20287
 PGM20288


```

IF((ABS(ANGOUT).GT.80.).OF.(ABS(ANGOUT).LT.30.))GO TO 25
IF(ABS(BIADIN).LT.0.5)GO TO 6
IF(ABS(ANGOUT).GT.70.)GO TO 2
IF(ABS(ANGOUT).LT.40.)GO TO 4
CALL FIG(9,AI,BI40,PTOCHO,YPIA(1))
CALL FIG(9,AI,BI50,PTOCHO,YPIA(2))
CALL FIG(9,AI,BI55,PTOCHO,YPIA(3))
CALL FIG(9,AI,BI60,PTOCHO,YPIA(4))
CALL FIG(9,AI,BI65,PTOCHO,YPIA(5))
CALL FIG(9,AI,BI70,PTOCHO,YPIA(6))
CALL FIG(6,ANG70,YPIA,A2,YPI)
GO TO 6

```

```

2 WRITE(KK,3)J,K
3 FORMAT(' ','THE ANGLE CUT WAS GREATER THAN 70 DEG AND THE DATA WAS
1 FROM 70 DEG FOR POW',I3,'STAGE',I3)
GO TO 6

```

```

4 WRITE(KK,5)J,K
5 FORMAT(' ','THE ANGLE CUT WAS LFSS
1 FROM 40 DEG FOR ROW',I3,'STAGE',I3)
CALL FIG(9,AI,BI40,PTOCHO,YPI)
IF(ABS(ANGOUT).LT.40.)GO TO 7
CALL FIG(9,A3,B340,PTOCHO,YPNA(1))
CALL FIG(9,A3,B350,PTOCHO,YPNA(2))
CALL FIG(9,A3,B360,PTOCHO,YPNA(3))
CALL FIG(9,A3,B365,PTOCHO,YPNA(4))
CALL FIG(9,A3,B370,PTOCHO,YPNA(5))
CALL FIG(9,A3,B375,PTOCHO,YPNA(6))
CALL FIG(9,A3,B180,PTOCHO,YPNA(7))
CALL FIG(7,ANG80,YPNA,A2,YPN)
GO TO 8

```

```

7 WRITE(KK,5)J,K
8 CALL FIG(9,A3,B340,PTOCHO,YPN)
YP=(YPN+(BIADIN/ANGOUT)*2*(YPI-YPN))* (TC/.2) ** (-BIADIN/ANGOUT)
IF(ABS(CINCT).LT.0.5)GO TO 13
IF(PTOCHO.GT.1.)GO TO 35
CALL FIG(7,PCA,PCB,PTOCHO,C75)

```

PGM20325
 PGM20326
 PGM20327
 PGM20328
 PGM20329
 PGM20330
 PGM20331
 PGM20332
 PGM20333
 PGM20334
 PGM20335
 PGM20336
 PGM20337
 PGM20338
 PGM20339
 PGM20340
 PGM20341
 PGM20342
 PGM20343
 PGM20344
 PGM20345
 PGM20346
 PGM20347
 PGM20348
 PGM20349
 PGM20350
 PGM20351
 PGM20352
 PGM20353
 PGM20354
 PGM20355
 PGM20356
 PGM20357
 PGM20358
 PGM20359
 PGM20360


```

GO TO 34
C75=PCB(7)--.4*(PTOCHO-1.)
A275=A2/C75
AB=BLADIN/A275
IF(J.EQ.2)AB=-BLADIN/A275
IF((AB.LT.-.9).OR.(AB.GT.1.))GO TO 25
IF((A275.GT.65.))AND.(AB.GT.0.4))GO TO 9
IF((A275.LT.40.))AND.(AB.GT.0.6))GO TO 11
CALL FIG(9,A8,B830,AB,SI75(1))
CALL FIG(11,A811,B840,AB,SI75(2))
CALL FIG(11,A811,B850,AB,SI75(3))
CALL FIG(11,A811,B855,AB,SI75(4))
CALL FIG(11,A811,B860,AB,SI75(5))
CALL FIG(11,A811,B865,AB,SI75(6))
CALL FIG(7,A7,B7,AB,SI75(7))
CALL FIG(7,ANG30,SI75,A275,SI751)
GO TO 12

```

```

9 WRITE(KK,10)J,K
10 POPMAT(' ','THE ANGLE OUT WAS GREATER THAN 65 DEG AND THE DATA WAS
1 FROM 65 DEG FOR ROW',I3,'STAGE',I3)
CALL FIG(11,A811,B865,AB,SI751)
GO TO 12

```

```

11 WRITE(KK,5)J,K
CALL FIG(9,A8,B830,AB,SI751)
12 IF(PTCCHO.GT.0.8)GO TO 13
CALL FIG(5,A5,B5,PTCCHO,DELTST)
GO TO 16
13 IF(PTOCHO.GT.1.)GO TO 14
IF((A2.LT.40.))OP.(A2.GT.70.))GO TO 14
CALL FIG(5,A15,B1540,PTOCHO,SIDA(1))
CALL FIG(5,A15,B1550,PTOCHO,SIDA(2))
CALL FIG(5,A15,B1560,PTOCHO,SIDA(3))
CALL FIG(5,A15,B1570,PTCCHO,SIDA(4))
CALL FIG(4,ANG47,SIDA,A2,DELTST)
GO TO 16
WRITE(KK,15)J,K

```

```

PGM20361
PGM20362
PGM20363
PGM20364
PGM20365
PGM20366
PGM20367
PGM20368
PGM20369
PGM20370
PGM20371
PGM20372
PGM20373
PGM20374
PGM20375
PGM20376
PGM20377
PGM20378
PGM20379
PGM20380
PGM20381
PGM20382
PGM20383
PGM20384
PGM20385
PGM20386
PGM20387
PGM20388
PGM20389
PGM20390
PGM20391
PGM20392
PGM20393
PGM20394
PGM20395
PGM20396

```



```

15  FORMAT(' ', THE P/C RATIO WAS GREATER THAN DATA FOR OFF INCIDENCE
    *CALCULATIONS, A VALUE FOR DELTA INCIDENCE IS TAKEN FOR P/C=1./' ',
    2* FOR ROW, I3, ' STAGE', I3)
    IF(A2.GT.70.) DELTISI=R1570(5)
    IF(A2.LT.70.) DELTISI=B1570(5)
    IF(A2.LT.60.) DELTISI=R1560(5)
    IF(A2.LT.50.) DELTISI=B1550(5)
    IF(A2.LT.40.) DELTISI=B1540(5)
    SI=DELTISI+SI751
    SIP=CINCI/SI
    IF((SIR.GT.1.5).OR.(SIR.LT.-4.)) GO TO 17
    CALL FIG(7, YP2A, YP2B, SIR, YPC)
    YP=YPC*YPC
    GO TO 18
    YP=YPC*10.
    WRITE(KK, 29) J, K
    FORMAT(' ', THE DATA LIMITS WERE EXCEEDED IN ROW, I3, ' STAGE', I3)
    A6=ANGIN*PI/180.
    IF(J.EQ.1) A6=-A6
    A4=ANGOUT*PI/180.
    IF(J.EQ.2) A4=-A4
    ANGMEN=ATAN(.5*(TAN(A4)-TAN(A6)))
    CLSC=2.*(TAN(A6)+TAN(A4))*COS(ANGMEN)
    ZETA=CLSC**2*(COS(A4)**2/COS(ANGMEN)**3)
    YS=.0334*(COS(A4)/COS(B1))*ZETA/ASPEC
    IF(KLEAP.EQ.1) B=.0
    IF(KLEAP.EQ.2) B=.37
    IF(KLEAP.EQ.3) B=.47
    YK=B*(TIPCL/CHORD)**.78*ZETA/ASPEC
    IF(REMACH.GT.1.) YP=YPC*(1.+60.*(REMACH-1.))**2)
    YPS=(YP+YS)*(PE/2.E5)**(-.2)
    YT=YPS*YK
    CALL FIG(7, TET, TEY, TET, YPEC)
    YT=YT*YPC
    RETURN
    YP=0.5

```

24
25

PGM20397
PGM20398
PGM20399
PGM20400
PGM20401
PGM20402
PGM20403
PGM20404
PGM20405
PGM20406
PGM20407
PGM20408
PGM20409
PGM20410
PGM20411
PGM20412
PGM20413
PGM20414
PGM20415
PGM20416
PGM20417
PGM20418
PGM20419
PGM20420
PGM20421
PGM20422
PGM20423
PGM20424
PGM20425
PGM20426
PGM20427
PGM20428
PGM20429
PGM20430
PGM20431
PGM20432


```

26      WRITE(KK,26)J,K
      FORMAT(' ',THE INPUT DATA WAS GREATER THAN LIMITS OF PROGRAM & A
1       VALUE OF Y=.5 WAS ASSIGNED FOR TOW,I2,' SIZE',I3)
      GO TO 24
      END
      SUBROUTINE BLADE(A,B,C,J)
      C CALCULATES BLADE OF GAS ANGLES FOR ROTOR AND STATORS
      DIMENSION D(6),P(6)
      DATA D/24.34,40.,50.,60.,70.,80./
      DATA P/30.,47.717,53.962,62.452,70.943,78.868/
      IF(J.F0.2)GO TO 3
      IF(B.LI.1.)GO TO 1
      C=A
      GO TO 2
      CALL FIG(6,D,P,A,C)
      IF(B.LI.0.5)GO TO 2
      C=C-((B-.5)/.5)* (C-A)
      RETURN
      IF(B.LI.1.)GO TO 5
      A=C
      GO TO 2
      CALL FIG(6,P,D,C,A)
      IF(B.LI.0.5)GO TO 2
      A=A+((B-.5)/.5)* (C-A)
      GO TO 2
      END
      SUBROUTINE FIG(I,D,P,X,Y)
      DIMENSION D(I),P(I),DD(100),FF(100),A(4),B(4)
      IF((X.GT.D(I)).AND.(D(1).LT.D(I))).OR.((X.LT.D(1)).AND.(D(I).GT.D
1(1))).OR.((X.LT.D(I)).AND.(D(1).GT.D(I))).OR.((
2X.GT.D(1)).AND.(D(1).GT.D(I)))GO TO 31
      IF(D(1).GT.D(I))GO TO 9
      IF(J.F0.4)GO TO 2
      N=I-1
      J=2
      IF(X.GE.D(N))GO TO 1

```

PGM20433
 PGM20434
 PGM20435
 PGM20436
 PGM20437
 PGM20438
 PGM20439
 PGM20440
 PGM20441
 PGM20442
 PGM20443
 PGM20444
 PGM20445
 PGM20446
 PGM20447
 PGM20448
 PGM20449
 PGM20450
 PGM20451
 PGM20452
 PGM20453
 PGM20454
 PGM20455
 PGM20456
 PGM20457
 PGM20458
 PGM20459
 PGM20460
 PGM20461
 PGM20462
 PGM20463
 PGM20464
 PGM20465
 PGM20466
 PGM20467
 PGM20468

PGM20469
PGM20470
PGM20471
PGM20472
PGM20473
PGM20474
PGM20475
PGM20476
PGM20477
PGM20478
PGM20479
PGM20480
PGM20481
PGM20482
PGM20483
PGM20484
PGM20485
PGM20486
PGM20487
PGM20488
PGM20489
PGM20490
PGM20491
PGM20492
PGM20493
PGM20494
PGM20495
PGM20496
PGM20497
PGM20498
PGM20499
PGM20500
PGM20501
PGM20502
PGM20503
PGM20504

```

IF(X.LE.D(J))GO TO 2
L=3
IF(X.LT.D(L))GO TO 3
L=L+1
GO TO 4
L=L-2
DO 8 K=1,4
A(K)=D(L)
B(K)=F(L)
L=L+1
GO TO 7
IJ=I-4
DO 5 II=1,4
A(II)=D(IJ)
B(II)=F(IJ)
IJ=IJ+1
GO TO 7
DO 6 II=1,4
A(II)=D(II)
B(II)=F(II)
CALL BK(A,B,X,Y)
RETURN
DD(1)=D(I)
FF(1)=F(I)
J=I
DO 15 M=2,I
J=J-1
DD(M)=D(J)
FF(M)=F(J)
DO 16 M=1,I
D(M)=DD(M)
F(M)=FF(M)
GO TO 17
IF((X.GT.D(I)).AND.(D(1).LT.D(I)).OR.((X.LT.D(I)).AND.(D(1).GT.D
1(I))))Y=F(I)
IF(Y.NE.F(I))Y=F(1)

```

4

3

8

1

5

2

6

7

32

9

15

16

31


```

GO TO 22
END
SUBROUTINE BK(Y,Y,XARG,YOUT)
DIMENSION X(4),Y(4)
YOUT=0.0
XSQ=XARG**2
XCU=XSC*XARG
DO 10 K=1,4

```

```

    PI1=1.
    PI2=1.
    SUM1=0.0
    SUM2=0.0
    DO 20 J=1,4
      IF(J.EQ.K) GO TO 20
      PI1=PI1*(X(K)-X(J))
      SUM1=SUM1+X(J)
      PI2=PI2*X(J)
    DO 99 I=2,4
      IF(I.EQ.K) GO TO 99
      IF(I.LE.J) GO TO 99
      SUM2=SUM2+X(J)*X(I)

```

```

99
20
10

```

```

CONTINUE
CONTINUE
YOUT=YOUT+1./PI1*(XCU-SUM1*XSQ+SUM2*XARG-PI2)*Y(K)
IF((Y(1).GE.Y(2)).AND.(Y(2).GE.Y(3)).AND.(Y(3).GE.Y(4)).AND.(XARG.
1LE.X(2)).AND.((YOUT.GT.Y(1)).OR.(YOUT.LE.Y(2)))) YOUT=Y(1)-(Y(1)-Y(
12))*XARG-X(1))/(X(2)-X(1))
IF((Y(1).GE.Y(2)).AND.(Y(2).GE.Y(3)).AND.(Y(3).GE.Y(4)).AND.(XARG.
1LE.X(3)).AND.(XARG.GT.X(2)).AND.((YOUT.GT.Y(2)).OR.(YOUT.LE.Y(3))))
2) YOUT=Y(2)-(Y(2)-Y(3))*XARG-X(2))/(X(3)-X(2))
IF((Y(1).GE.Y(2)).AND.(Y(2).GE.Y(3)).AND.(Y(3).GE.Y(4)).AND.(XARG.
1LE.X(4)).AND.(XARG.GT.X(3)).AND.((YOUT.GT.Y(3)).OR.(YOUT.LE.Y(4))))
2) YOUT=Y(3)-(Y(3)-Y(4))*XARG-X(3))/(X(4)-X(3))
IF((Y(1).LE.Y(2)).AND.(Y(2).LE.Y(3)).AND.(Y(3).LE.Y(4)).AND.(XARG.
1LE.X(2)).AND.((YOUT.LE.Y(1)).OR.(YOUT.GT.Y(2)))) YOUT=Y(1)-(Y(1)-Y(
12))*XARG-X(1))/(X(2)-X(1))

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PGM20508
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|----------|----------|-----------|-----------|-----------|-----------|-----------|----------|----------|----------|----------|
| 0.99973 | 0.99893 | 0.99760 | 0.99574 | 0.99335 | 0.99040 | 0.98700 | 0.98317 | 0.97870 | 0.97380 | PGM20577 |
| 0.96835 | 0.96260 | 0.95630 | 0.94950 | 0.94200 | 0.93440 | 0.92650 | 0.91800 | 0.90920 | 0.90030 | PGM20578 |
| 0.89080 | 0.88100 | 0.87060 | 0.86000 | 0.84950 | 0.83860 | 0.82720 | 0.81570 | 0.80400 | 0.79220 | PGM20579 |
| 0.78020 | 0.76750 | 0.75550 | 0.74270 | 0.73070 | 0.71800 | 0.70520 | 0.69140 | 0.67910 | 0.66650 | PGM20580 |
| 0.65500 | 0.64080 | 0.62760 | 0.61410 | 0.60250 | 0.58900 | 0.57550 | 0.56500 | 0.55100 | 0.54000 | PGM20581 |
| 0.52800 | 0.51700 | 0.50400 | 0.49100 | 0.48000 | 0.46820 | 0.45650 | 0.44530 | 0.43330 | 0.42050 | PGM20582 |
| 0.96151 | 1.91194 | 2.86589 | 3.82740 | 4.77028 | 5.72284 | 6.67004 | 7.61008 | 8.56787 | 9.50320 | PGM20583 |
| 10.45287 | 11.36668 | 12.31329 | 13.26734 | 14.22884 | 15.14564 | 16.04750 | 16.99411 | 17.90344 | 18.82024 | PGM20584 |
| 19.75192 | 20.63144 | 21.54077 | 22.45010 | 23.37002 | 24.27625 | 25.16321 | 26.05019 | 26.95207 | 27.80922 | PGM20585 |
| 28.65892 | 29.54590 | 30.41052 | 31.26750 | 32.09502 | 32.90472 | 33.69723 | 34.48190 | 35.25635 | 36.02093 | PGM20586 |
| 37.05164 | 37.93861 | 38.76595 | 39.60822 | 40.39829 | 41.14365 | 42.03061 | 42.70889 | 43.41698 | 44.16234 | PGM20587 |
| 44.98222 | 45.69031 | 46.36113 | 47.17357 | 47.85184 | 48.50720 | 49.34256 | 50.02827 | 50.74382 | 51.39220 | PGM20588 |
| 400.600 | 800.1000 | 1200.1400 | 1600.1600 | 1900.2000 | 2200.2200 | 2400.2400 | | | | PGM20589 |
| 100.135 | 166.192 | 218.218 | 242.264 | 284.302 | 320.320 | 338.338 | | | | PGM20590 |
| | | | | | | | | | | PGM20591 |

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